

CR-112245

ACTIVE TRANSMISSION  
ISOLATION/ROTOR LOADS  
MEASUREMENT SYSTEM

FINAL REPORT

by Irwin J. Kenigsberg, John J. DeFelice

**CASE FILE  
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Prepared Under Contract No. NAS1-11549 by  
Sikorsky Aircraft Division of United Aircraft Corporation  
Stratford, Connecticut

for

NATIONAL AERONAUTICS AND SPACE ADMINISTRATION  
AND  
UNITED STATES ARMY

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# LISTING OF SYMBOLS

$W_{UB1}$	-	Weight of the upper part of the upper body.
$X_{UB1}$	-	x-location of the C.G. of the upper part of the upper body.
$Y_{UB1}$	-	y-location of the C.G. of the upper part of the upper body.
$Z_{UB1}$	-	z-location of the C.G. of the upper part of the upper body.
$I_{OXUB1}$	-	Moment of Inertia about the x-axis of the upper part of the upper body.
$I_{OYUB1}$	-	Moment of Inertia about the y-axis of the upper part of the upper body.
$W_{UB2}$	-	Weight of the lower part of the upper body.
$X_{UB2}$	-	x-location of the C.G. of the lower part of the upper body.
$Y_{UB2}$	-	y-location of the C.G. of the lower part of the upper body.
$Z_{UB2}$	-	z-location of the C.G. of the lower part of the upper body.
$I_{OXUB2}$	-	Moment of Inertia about the x-axis of the lower part of the upper body.
$I_{OYUB2}$	-	Moment of Inertia about the y-axis of the lower part of the upper body.
$W_{LB}$	-	Weight of the lower body.
$X_{LB}$	-	x-location of the C.G. of the lower body.
$Y_{LB}$	-	y-location of the C.G. of the lower body.
$Z_{LB}$	-	z-location of the C.G. of the lower body.
$I_{OXLB}$	-	Moment of Inertia about the x-axis of the lower body.
$I_{OYLB}$	-	Moment of Inertia about the y-axis of the lower body.
$X_{Ii}$	-	x-location of the active isolator (i).
$Y_{Ii}$	-	y-location of the active isolator (i).
$Z_{Ii}$	-	z-location of the active isolator (i).



# LISTING OF SYMBOLS (continued)

$X_{LCi}$	-	x-location of the drag strut mounting point on the transmission frame.
$Y_{LCi}$	-	y-location of the drag strut mounting point on the transmission frame.
$Z_{LCi}$	-	z-location of the drag strut mounting point on the transmission frame.
$X_{TAPi}$	-	x-location of the torque application point on the transmission frame.
$Y_{TAPi}$	-	y-location of the torque application point on the transmission frame.
$Z_{TAPi}$	-	z-location of the torque application point on the transmission frame.
ALPHA	-	The angle the torque application strut makes with the lateral centerline.
$X_P$	-	x-location of the point where the propulsive load is applied.
$Y_P$	-	y-location of the point where the propulsive load is applied.
$Z_P$	-	z-location of the point where the propulsive load is applied.
BETA	-	The angle the propulsive loading cable makes with the longitudinal centerline.
$X_A$	-	x-location of the reference point A located at the rotor head.
$Y_A$	-	y-location of the reference point A located at the rotor head.
$Z_A$	-	z-location of the reference point A located at the rotor head.
$A_H$	-	Air volume reaction surface - high pressure side.
$A_L$	-	Air volume reaction surface - low pressure side.
$P_A$	-	Isolator air precharge - high side.
$P_B$	-	Isolator air precharge - low side.

# LISTING OF SYMBOLS (continued)

G	-	Isolator high gain.
Cv	-	Isolator element viscous damping coefficient.
XAi	-	x-location of accelerometer (i).
YAi	-	y-location of accelerometer (i).
ZAi	-	z-location of accelerometer (i).
SF	-	vibratory force applied with the hydraulic shaker.
SFREQ	-	frequency at which the shaker force is applied.
XSH	-	x-location of the hydraulic shaker.
YSH	-	y-location of the hydraulic shaker.
ZSH	-	z-location of the hydraulic shaker.
THTLAT	-	The angle between the shaker force and the Z axis in the Y-Z plane.
THTLON	-	The angle between the shaker force and the Z axis in the X-Z plane.
DR	-	propulsive load.
TAP1	-	Load in torque application system load cell located on the port side of the aircraft.
TAP2	-	Load in torque application system load cell located on the starboard side of the aircraft.
FLC1T	-	Load in front longitudinal drag link.
FLC2T	-	Load in front lateral drag link.
FLC3T	-	Load in rear longitudinal drag link.
FLC4T	-	Load in rear lateral drag link.
PiH	-	High side operating pressure of isolator (i).
PiL	-	Low side operating pressure of isolator (i).

# LISTING OF SYMBOLS (continued)

DEL <sub>i</sub>	-	Relative displacement of isolator (i).
ACC <sub>i</sub>	-	Steady and vibratory acceleration plus the phase angle measured in accelerometer (i).
Ba,b	-	A damping coefficient on low and high pressure sides of isolator element.
Ga,b	-	Servovalve flow gains for low and high pressure sides of isolator element.
Pd	-	Isolator element drain pressure.
Ph,l	-	Precharge pressure of isolator element, high and low pressure source.
qa,b	-	Flow rates of servovalve for low and high pressure sides of isolator element.
Sa,b	-	Piston area for low and high pressure sides of isolator element.
Vcao,bo	-	Volume contained between diaphragms and air restriction for low and high pressure sides of isolator element.
Vca <sup>l</sup> ,b <sup>l</sup>	-	Volume contained between diaphragms and air restrictor for low and high pressure sides of isolator element under operating conditions.
Vt	-	Volume contained between housing and air restrictor for general air spring.
Vta,b	-	Volume contained above air restrictor for low and high operating pressure sides of isolator element.
S	-	Steady value.
V	-	Vibratory value.
P	-	Phase angle referenced against the contactor of the hydraulic shaker.

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## SUMMARY

Modifications were incorporated into the Sikorsky active transmission isolation system to provide the capability of utilizing the system as a rotor force measuring device. These included; isolator redesign to improve operation and minimize friction, installation of pressure transducers in each isolator and load cells in series with each torque restraint link. Full scale vibration tests performed during this study on a CH-53A helicopter airframe verified that these modifications do not degrade the systems wide band isolation characteristics.

Bench tests performed on each isolator unit indicated that steady and transient loads can be measured to within 1 percent of applied load. Individual isolator vibratory load measurement accuracy was determined to be 4 percent. Load measurement accuracy was found to be independent of variations in all basic isolator operating characteristics.

Full scale system load calibration tests on the CH-53A airframe established the feasibility of simultaneously providing wide band vibration isolation and accurate measurement of rotor loads. Principal rotor loads (lift, propulsive force, and torque) were measured to within 2 percent of applied load. This is within the accuracy projected for this system during the pre-design study of the Rotor Systems Research Aircraft (Ref. 1). The time histories of transient loads were accurately determined while relative motions at the transmission airframe interface were maintained within design tolerances of control systems and high speed engine drive shafts.

In addition to the measurement of principal rotor forces, tests were performed to establish the feasibility of measuring vibratory rotor forces. It was determined that vibratory forces could be measured with an average error of 20 percent of applied force at all frequencies where effective isolation was provided. The higher error in vibratory load measurement results from the requirement to adjust measured load cell and isolator loads to include the inertia forces resulting from transmission motion. The principal contributor to the increased error was found to be the sensitivity of the accelerometers utilized in the tests. Evaluation of piezoelectric accelerometers is therefore recommended for improved accuracy in future force measurement tests. It is anticipated that through the use of the piezoelectric accelerometers, the accuracy of vibratory load measurement can be substantially improved.

As a result of these tests it is concluded that it is feasible to use a rotor force measurement system in determining steady, transient and vibratory rotor loads while providing tunable transmission isolation. It is recommended that the active transmission isolation system be further developed for vibration control and rotor force measurement in conjunction with slowed rotor operation.

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## INTRODUCTION

A basic requirement of the proposed Rotor System Research Aircraft (RSRA) is an in flight rotor force measurement system (Ref. 1). In addition, if the RSRA is to accommodate certain rotors which operate over a wide range of blade passage frequencies, such as the slowed rotor, an active vibration suppression system is required to avoid rotor/airframe dynamic resonances. A system of this type would also provide tuning for other rotors whose excitation frequencies fall beyond the operational bands made available through structural tuning of the airframe.

Full scale active transmission isolation has been successfully demonstrated under USAAMRDL Contract DAAJ02-69-C-0101 (Ref. 2). The isolation system installed on a CH-53A helicopter consists of three unidirectional hydropneumatic isolators installed vertically at the transmission/airframe interface and horizontal rigid links connecting the transmission to the airframe to provide torsional restraint. The system exhibits wide band vibration isolation at all frequencies above isolator resonances while limiting steady and transient relative airframe/transmission deflections within design targets for control systems and high speed engine drive shafts.

Load calibration tests of an individual active isolator performed under NASA Contract NAS1-10850 established the feasibility of utilizing the units as load measuring devices.

The objective of this study is to establish, through full scale experimental ground tests, the feasibility of providing simultaneous wide band active transmission isolation and accurate measurement of steady, transient and vibratory rotor loads.

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## ISOLATOR MODIFICATIONS

The active isolators as described in Reference 2, were modified and reworked to remove wear resulting from previous operation and to improve their accuracy as load measurement devices. Features of the isolator units are illustrated in Figures 1 and 2. Figure 3 shows the isolator modifications. That is, the existing sleeves were replaced and the piston head was turned down and replaced with a self-aligning Beryllium-copper head to reduce friction. A tight press-fit stainless steel cylindrical sleeve and a ball bushing were installed to eliminate the potential of the piston shaft cocking. The piston shaft was built up through chrome plating and ground to eliminate slop. In addition, a stainless steel stop and concentric Beryllium-copper self-aligning seal was installed at the base of the piston shaft. Spacers were installed between the sleeve and caps to eliminate cocking, caused by torquing the head bolts. Hydraulic pressure transducers were installed and the anti-rotation device in the servo-linkage was modified to prevent binding during operation.

## ISOLATOR BENCH TESTING

The modified active isolators underwent load calibration and functional bench testing prior to installation on the test vehicle for the full scale system calibration. Steady, vibratory and transient loads were applied to each isolator. Effects of variation in isolator precharge, gain, and damping were performed to evaluate the effect of said parameters on load calibration.

Prior to installation in the bench test rig, dry breakout friction was evaluated on each isolator. All three isolators exhibited breakout friction of under 7 pounds over the limits of piston travel, as is illustrated in Figure 4. These checks were made during each stage of assembly, to identify whether misalignment existed, and also at the conclusion of final assembly. The tests were repeated under hydraulic pressure in the bench test rig. No change in breakout friction was identified.

Isolator testing was performed in the bench test facility shown in Figure 5. Mounted in series with the instrumented isolator unit was a hydraulic ram used for applying steady and vibratory loads and a load cell used to measure the applied load.

Instrumentation, shown in Figure 6, consisted of two absolute pressure transducers, a displacement potentiometer for monitoring isolator operation, pressure monitoring gages, and three load cells to establish the applied load. Load cells of 5, 10, and 20 thousand pounds were utilized to minimize the applied load error in each test range. Oscillograph traces were utilized to record direct measurement of applied load, isolator pressures and displacement. In addition, two traces were utilized to record the isolator load and error which are electronically calculated from direct measurements by a specially constructed analog circuit.

## ISOLATOR BENCH TESTING (continued)

Basic testing consisted of steady load calibrations including an evaluation of the effects of vibratory load levels which can be anticipated in the in-flight environment. The vibratory loading utilized, was a maximum of  $\pm 1000$  lbs at sample frequencies of 10, 18.5 and 25 HZ. Transient load tests (15,000 to 2,000 lbs in 2 seconds) were also conducted. The tests were then expanded to evaluate the effect of variations in isolator gain, precharge and damping on the load measurement accuracy. The parametric variations tested are listed in Table 1.

The results obtained from these tests are summarized below:

1. With the isolator at its nominal settings, steady loads were measured to within  $\pm 1\%$  of the applied load about a linear bias which can be removed through calibration. The presence of vibratory load at frequencies of 10, 18.5 and 25 HZ, did not affect the accuracy of measuring steady loads (Figure 7). In addition, all transient test points, fall within the established steady error band (Figure 8).
2. All three active isolators exhibit identical measurement accuracy (Figures 7, 9, and 10).
3. Vibratory loads can be measured to within  $\pm 4\%$  of applied vibratory load, independent of steady load applied or excitation frequency. (Figures 11 to 13 illustrate these results).
4. Isolator load measurement accuracy is independent of variations in isolator characteristics. Ten combinations of gain, precharge and damping were considered. The resulting steady and vibratory calibration data is presented in Figures 14 to 21. It is concluded from these tests, that parametric variations of isolator parameters need not be performed during the full scale system test since no effect on load measurement accuracy is detected.

## FULL SCALE TEST DESCRIPTION

### Test Apparatus and Loading Systems

The ground test facility used to evaluate the active isolation system as a rotor loads measurement system is shown in Figure 22. Principal elements are a bungee suspension system to simulate free flight, three load application systems, for application of steady and transient rotor loads, and a hydraulic shaker mounted at the rotor head to simulate steady-state vibratory rotor loads.

The structural static CH-53A test vehicle, configured with a MK II tail rotor pylon and stabilizer, was weighed and ballasted to a 34,200 pound gross weight with a neutral center-of-gravity mass distribution. Concentrated weights were used to dynamically simulate the tail rotor and intermediate gearbox, which were not installed on the test vehicle. The engines were not installed or simulated to simplify the test set-up.

The bungee suspension system labeled in Figure 23 lifted the entire vehicle off the deck to simulate free flight by maintaining all rigid body modes below 1 HZ. A rigid ballast (shown in Figure 23) was used to simulate the mass of the rotor head and blades and to provide attachments to the suspension system, the external longitudinal loading system, and the shaker. The transmission was bolted to a large steel plate, with the isolators suspending the fuselage through holes cut in the skin of the cabin ceiling (Figure 24). The lower isolator attachment was an I-beam frame, weighing approximately 700 pounds, bolted to the lower caps of the fuselage frame (Figure 25). The weight of the transmission attachment plate, approximately 1300 pounds, was offset by using a transmission housing without gears. This permitted the upper body to have a mass moment of inertia and center-of-gravity simulating that of the actual vehicle. System installation was largely dictated by the location of the existing CH-53A test vehicle structural members. It must be emphasized that the weight and complexity of this installation would not exist if the isolation system were integrated into the vehicle's basic design. In-plane and torque restraint was provided at the transmission base by four drag struts located symmetrically about station 342 and the fuselage centerline (Figure 24).

A complete assembly of the test facility is illustrated in Figure 26. Simulated steady and transient propulsive force was applied to the rotor head with a loading system comprised of a hydraulic cylinder, a 5000 pound BLH load cell, an electronic quick release mechanism, and a steel cable attached, in series between the rotor head and the aircraft tail structure (Figure 27). Loading was applied to the rotor head by pressurizing the cylinder, thus producing tension in the cable between the rotor head and the tail structure. Transient loading was accomplished by activating the quick release mechanism which released the applied load instantaneously, applying a step-input load to the rotor head.

## FULL SCALE TEST DESCRIPTION (continued)

### Apparatus and Loading Systems (continued)

Steady main rotor torque loading was provided through a pair of hydraulic cylinders located so as to apply a couple to the main rotor gearbox. The cylinders shown in Figures 24 and 28 attached to the transmission adapter plate and to a specially designed steel framework, which in turn was fastened to the basic airframe. Applied loads were measured with 20,000 pound BLH load cells, located in series with the cylinders. The applied torque was reacted by the four in-plane links (Figures 24 and 28). The links, of equal length, were located symmetrically about station 342 and the fuselage centerline and attached between the transmission plate and the steel loading framework. Torque loads were measured with four 20,000 pound BLH load cells, one located in series with each link. Design of this apparatus provided for location and installation of the torque cylinders, drag links, and steel framework at the base of the transmission so as not to alter the capability of the system to isolate the aircraft from vertical and in-plane vibratory excitations. Spherical bearings were incorporated at each attachment point to allow movement of the upper body with respect to the basic airframe during isolation tests.

The vertical transient loading system shown in Figures 25 and 29 consisted of a hydraulic cylinder attached to the base of the transmission and an electronic quick release mechanism fastened to a steel framework which in turn bolted to the isolator lower attachment frame. A 10,000 pound BLH load cell, in series with the cylinder, measured the applied load. Loading was accomplished by pressurizing the cylinder which pulled the transmission toward the fuselage. Transients were accomplished by activating the quick release mechanism which instantaneously released the applied load, providing a step-input load to the upper body. Steady lift variations were also accomplished using this system.

Steady-state vibratory loadings were simulated with a hydraulically driven unidirectional shaker. The shaker consisted of two counter-rotating eccentric masses with adjustable unbalance which produced a unidirectional sinusoidal excitation proportional to the square of the rotational speed. The shaker was hydraulically powered with a large commercial pump. A manually operated flow bypass control valve was used to adjust rotational speed. The shaker axis was reoriented to provide vertical, longitudinal, and lateral excitations and combinations thereof.



## Test Instrumentation

Four measurement systems were used to meet the requirements of these tests. The first system, involving a normalizing unit and an x-y-y' plotter output display, defined the frequency response of the test vehicle with and without the isolators activated. Steady and vibratory load measurement data was acquired, utilizing the Dymec Load Measurement System and a 36 channel CEC oscillograph. A narrow-band FM tape system was used to record the transient response real-time histories of the load measurement parameters for transient load conditions. The FM tape system was also used as a back-up for measuring vibratory load conditions. The major instrumentation components and master test control panel are shown in Figure 30. Active isolators were instrumented in the same manner as in the load calibration bench tests. Accelerometer locations and their orientation with respect to the test vehicle are illustrated by arrows in the structural arrangement shown in Figure 31. Two sets of accelerometers were utilized, each set located at the individual upper and lower center of gravity of the simulated transmission and rotor head. This instrumentation configuration was selected so as to preclude any errors resulting from relative motion due to flexibility in the shaft. The response of the transmission and simulated rotor head, combined with the measured vibratory load in the isolators and torque restraint links, provide the necessary measurement of vibratory forces.

## VIBRATION ISOLATION SUBSTANTIATION

The ability of the active isolation system to provide wide-band vibration isolation has been demonstrated by the tests described in Reference 2. The purpose of these tests was to verify that modifications required for load measurement did not alter these characteristics.

The steady-state isolator evaluation was made by comparing frequency response sweeps from the isolated and hard-mounted test vehicle configurations. Sweeps in all three principal directions were conducted from 150 to 1800 CPM (2.5 HZ to 30 HZ) with a main rotor unidirectional excitation of 850 pounds achieved at the CH-53A 6P frequency (1110 cpm).

Due to modifications in the individual units and the overall test vehicle configuration, including ballast distribution, system operating pressures were found to vary from previous test data. In order to maintain the spring rate of 10,000 pounds per inch as specified in Reference 2, air precharge pressures were modified as required. The precharge pressure calculations for each unit are presented in Appendix I.

### Isolation Substantiation

The result of selected isolated and unisolated response sweeps are presented in Figures 32 to 43. These sweeps show a substantial reduction of fuselage response due to main rotor excitations in all three principal directions, thus substantiating that the modifications required for the load calibration tests, have not altered the systems isolation characteristics. Effective isolation in the inplane directions was obtained from an effective 3P through the highest frequency tested (555 CPM to 1800 CPM), while the isolation present in the vertical direction exists over a slightly narrower band. The isolation system under discussion, was designed specifically for the CH-53A Helicopter, with an articulated rotor system. Experience with five and six bladed rotors of this type indicate that the principal vibratory excitations are in the inplane direction. Therefore, primary consideration in the design (Reference 2) was given to optimizing isolation for these excitations. The inplane isolator modes at 300 CPM were located as low as possible to obtain maximum 6P isolation, while minimizing 1P amplification. In the original design of this installation the vertical isolator mode at 650 CPM, was constrained by the location of the inplane modes and the geometry of the system. As a result, vertical isolation was not optimized at lower frequencies. Since some degree of isolation existed throughout the desired frequency range, no attempt to further optimize isolation beyond that achieved in the tests of Reference 2 was made. Design considerations required for a wider band of vertical isolation are discussed in Appendix II.

## VIBRATION ISOLATION SUBSTANTIATION (continued)

### Isolation Substantiation (continued)

No response sweep is presented for the pilot-vertical station due to a vertical excitation because of an instrumentation malfunction. However, Figure 33, which shows pilot-vertical response due to a combined longitudinal/vertical excitation is presented. Since isolation is shown for both the combined longitudinal/vertical excitation and pure longitudinal excitation (Figure 41), it is concluded that isolation at the pilot vertical station exists due to vertical excitation alone. This conclusion is further substantiated in Figures 33 through 35 which show reduced vertical response at the other fuselage locations due to vertical excitations.

## ROTOR LOADS MEASUREMENT AND CORRELATION

The principal objective of these tests was to verify the feasibility of simultaneously providing active transmission isolation and measurement of rotor loads. System calibration tests were performed during which known steady, steady-state vibratory and transient loadings were applied to the simulated rotor head, through load application systems installed on the test vehicle, or by known vehicle weight. Transducers installed on the rotor head, transmission housing, isolators, and torque restraint links were monitored and recorded to obtain those parameters necessary to measure the applied loads. All recorded data was processed using the Computer program described in Appendix III. The resulting measured loads are compared in Tables 2 and 3, with the known applied loads to establish the accuracy of the isolation system as a rotor loads measurement device.

Conditions investigated included:

- a) Combinations of steady rotor lift, propulsive force and main rotor torque.
- b) Combinations of principal rotor vibratory excitation at frequencies of 10, 18.5 and 30 HZ, in combination with steady rotor lift.
- c) Steady rotor lift and vertical transient load.
- d) Steady rotor lift and transient propulsive force.
- e) Simulated flight conditions including vibratory loads.

Magnitudes of all rotor loads applied are representative of levels which can be reasonably anticipated in actual flight.

Steady lift was provided by the bungee suspension system which raised the entire test vehicle off the deck to simulate free flight. Measurement of the rotor lift was accomplished by monitoring the loads present in each isolator. Excellent correlation between applied and measured steady rotor lift was obtained. The error in measuring rotor lift, alone and in the presence of superimposed steady torque, propulsive force, and vibratory loadings, averaged 2 percent of the applied lift. Linearity of the measurement system was verified by incrementally varying the lower body weight. The average error over the entire range was 2 percent of applied lift. These results are presented in Figure 44.

Steady main rotor torque was applied in the presence of steady lift and with combinations of propulsive force and vibratory loads, utilizing the torque application system described previously. In all cases, the measured torque correlated to within 2 percent of applied torque.

## ROTOR LOADS MEASUREMENT AND CORRELATION (continued)

The incremental torque loadings presented in Figure 45 show that the torque measurement is also linear.

Propulsive forces were measured with the load cells in the longitudinal transmission torque restraint links, to a 1 percent accuracy when measured alone. The results of incremental loading, to 4000 pounds, (Figure 46), also exhibited linearity. However, when propulsive force was measured during the simultaneous application of main rotor torque, the error in measured propulsive force increased substantially. This decrease in accuracy is largely attributed to deficiencies in the test set-up which includes; large secondary inplane loads introduced by the torque application system and the in-plane restraint links being configured so as to provide simultaneous measurement of main rotor torque, side force, and propulsive force. This coupling and resultant decrease in accuracy was predicted in the RSRA pre-design study (Ref. 1). These test results therefore substantiate the requirement for a rotor balance configuration in which a single longitudinal load sensor and two lateral load sensors are employed to decouple the measurement of propulsive force and main rotor torque. Such a configuration will provide maximum propulsive force measurement accuracy although sacrificing side force measurement accuracy. The 2% accuracy obtained in measuring all principal rotor forces is consistent with the projections of the RSRA pre-design study (Ref. 1).

Vibratory loads were applied at frequencies of 10, 18.5 and 30 HZ. Examination of oscillograph rolls indicated that the sensitivities utilized did not provide adequate resolution of the vibratory signals required for accurate phase measurement. Test conditions were repeated, utilizing magnetic tape and signals were amplified. Examination of this test data at the frequencies of 18.5 and 30 HZ where most effective isolation resulted, indicated that all vibratory transducer signals were either at  $0^{\circ}$  or  $180^{\circ}$  phase angle with respect to the reference force signal. A sample of test data is shown in Appendix III - D. However, difficulty in reading phase remained at the 10 HZ frequency and is attributed to the closer proximity of the excitation frequency to resonances and the resulting modal coupling. This difficulty is identified as the source of larger errors in measuring vibratory loads at 10 HZ.

At the 18.5 and 30 HZ test frequencies, the measurement of applied load was strongly dependant upon accelerometer data. The average error in measuring total vibratory force at these frequencies is 20 percent (Figure 47).

Three vibratory test points at which the error in measuring vibratory load varied from 25 - 50 percent are presented in Appendix IV. In these cases, the contribution of the accelerometer data to the measurement of the vibratory load varied from 65 to 130 percent of the net load. The acceleration levels recorded varied from .06 to .23 g's which is equivalent to a range of 3 to 10 percent of the full scale sensitivity of the accelerometers.

## ROTOR LOADS MEASUREMENT AND CORRELATION (continued)

It is concluded that the future use of piezoelectric accelerometers are required to produce improved accuracy of accelerometer and, therefore, vibratory load measurement. It is projected that vibratory load measurement accuracy can be substantially improved through the use of these transducers.

It can also be concluded from the results previously discussed that vibratory rotor loads can be most accurately measured when the airframe is effectively isolated. Under this condition, measurement of vibratory loads is predominately dependent upon measuring amplitude and phase of transmission accelerations. Future tests should make provisions for providing maximum possible resolution of vibratory signals, thus maximizing phase readout and vibratory load measurement accuracy.

Transient load tests were conducted to evaluate the ability of the rotor balance system to respond to and measure simulated flight maneuver loads. Longitudinal and vertical step input loads of 4,000 and 10,000 pounds, respectively, were applied, and data was recorded on an FM tape system. Data was digitized and fed into computer programs (Appendix III-B), where it was processed, and the time histories of applied load, measured load, and relative motions between the transmission and airframe were defined. The transient data presented in Figures 48 to 53 represent time histories of load and displacement over an elapsed time of five seconds, and are computed with a sampling rate of 240 data points per second. Applied and measured loads were compared for measurement accuracy and responsiveness of the force measuring system. The graphs presented are plotted individually for ease of comparison.

The time histories of the applied transient loads were determined by the load in the force application device and the response of the simulated transmission/rotor head. The measured forces were tabulated from transducers in the active isolators and torque restraint links.

As shown, the active isolation system was able to accurately measure the time history of the applied force even though the rate of application far exceeds that which would be anticipated in flight. Variance between applied and measured load is again attributed to accelerometer sensitivities, which in this case plays the predominant role in defining the time history of applied load. Of particular significance to in-flight operation is the fact that even under the application of these severe transient loads, relative angular and vertical motion between the transmission and airframe are well within the design tolerances of control systems and high speed engine drive shafts.

## CONCLUSIONS

The testing performed under this program has successfully demonstrated the feasibility of providing simultaneous active transmission vibration isolation and measurement of steady, vibratory and transient rotor loads. As a result of these tests, the following conclusions are reached:

### BENCH TEST

- (1) Active isolator units measure steady and transient loads to better than 1 percent of applied load about a linear bias which can be removed through calibration.
- (2) The presence of vibratory loads do not affect steady load measurement accuracy.
- (3) Vibratory loads can be measured to within 4 percent of applied vibratory load, independent of excitation frequency.
- (4) Load measurement accuracy is independent of variations in basic isolator operating characteristics which include servo gain, air precharge pressures, and hydraulic damping.

### SYSTEM TEST

- (1) Isolation system modifications required for rotor loads measurement do not affect isolation characteristics.
- (2) The configuration tested produced significant vibration reductions over the entire frequency range, above isolator resonances.
- (3) The average error in measuring steady rotor lift and main rotor torque is 2 percent of applied load, independent of the flight condition simulated and the presence of vibratory loads.
- (4) The average error in measuring propulsive force is 1 percent of applied load, independent of the presence of vibratory loading.
- (5) The configuration tested which was not originally designed for rotor force measurement utilizes transducers which simultaneously measure both main rotor torque and propulsive force. This coupling produces a substantial degradation in propulsive force measurement error when main rotor torque is applied simultaneously. This substantiates the results of pre-design studies (Ref. 1) which identified this problem and resulted in a proposed RSRA configuration which would decouple propulsive force and main rotor torque measurement.

## CONCLUSIONS (continued)

- (6) At frequencies where effective isolation to all excitations was obtained (above 3/rev on the CH-53A), the average error in measuring vibratory rotor loads was established as 20 percent of applied loads. This error is attributed predominantly to the accuracy of accelerometers utilized. It is anticipated that the use of piezoelectric accelerometers would substantially improve this accuracy.
- (7) The active transmission isolation system is able to accurately measure the time histories of transient loads while maintaining relative displacements between the transmission and airframe within design tolerances of control systems and high speed engine drive shafts.



## RECOMMENDATIONS

It is recommended that any Rotor Force measurement system for the RSRA contain the following features:

- (1) A single longitudinal load sensor to decouple the measurement of propulsive force and main rotor torque.
- (2) Piezoelectric accelerometers to provide greater sensitivity and accuracy in determining transmission and rotor load accelerations required for transient and vibratory load measurement.
- (3) Provisions in data acquisition and processing systems to provide maximum possible resolution of vibratory signals required for accurate phase definition.

It is also recommended that the active isolator system be developed for vibration control and rotor loads measurement on the RSRA for slowed rotor operation. The recommended steps for further development are:

- (1) Design analysis of overall flight vehicle/isolation system dynamic characteristics, including:
  - (a) Rotor stability, stress levels, control loads, and ground/air resonance.
  - (b) Control system/isolator coupling and stability augmentation system.
- (2) Detail Design including:
  - (a) Variable isolator operating characteristics.
  - (b) Rotor Balance instrumentation.
  - (c) Fail safe and reliability considerations.
  - (d) Hydraulic supply and temperature control.
  - (e) Engine and control systems interface.
- (3) Fabrication and testing on the RSRA including:
  - (a) Structural proof tests.
  - (b) Tiedown tests to substantiate compatibility of dynamic components.
  - (c) Flight tests.

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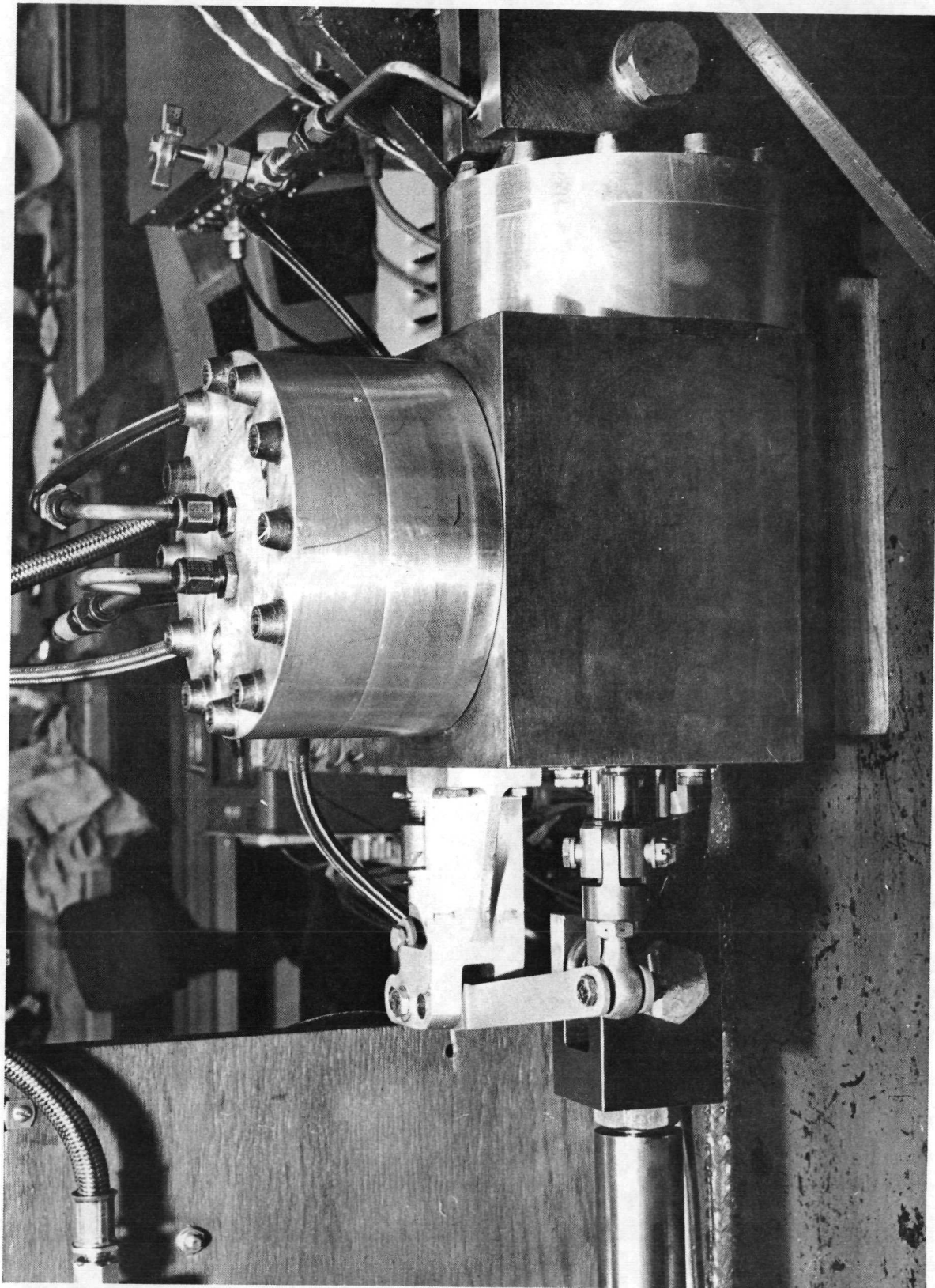


Figure 1. - Assembled Active Isolator.



Figure 2. - Disassembled Active Isolator.

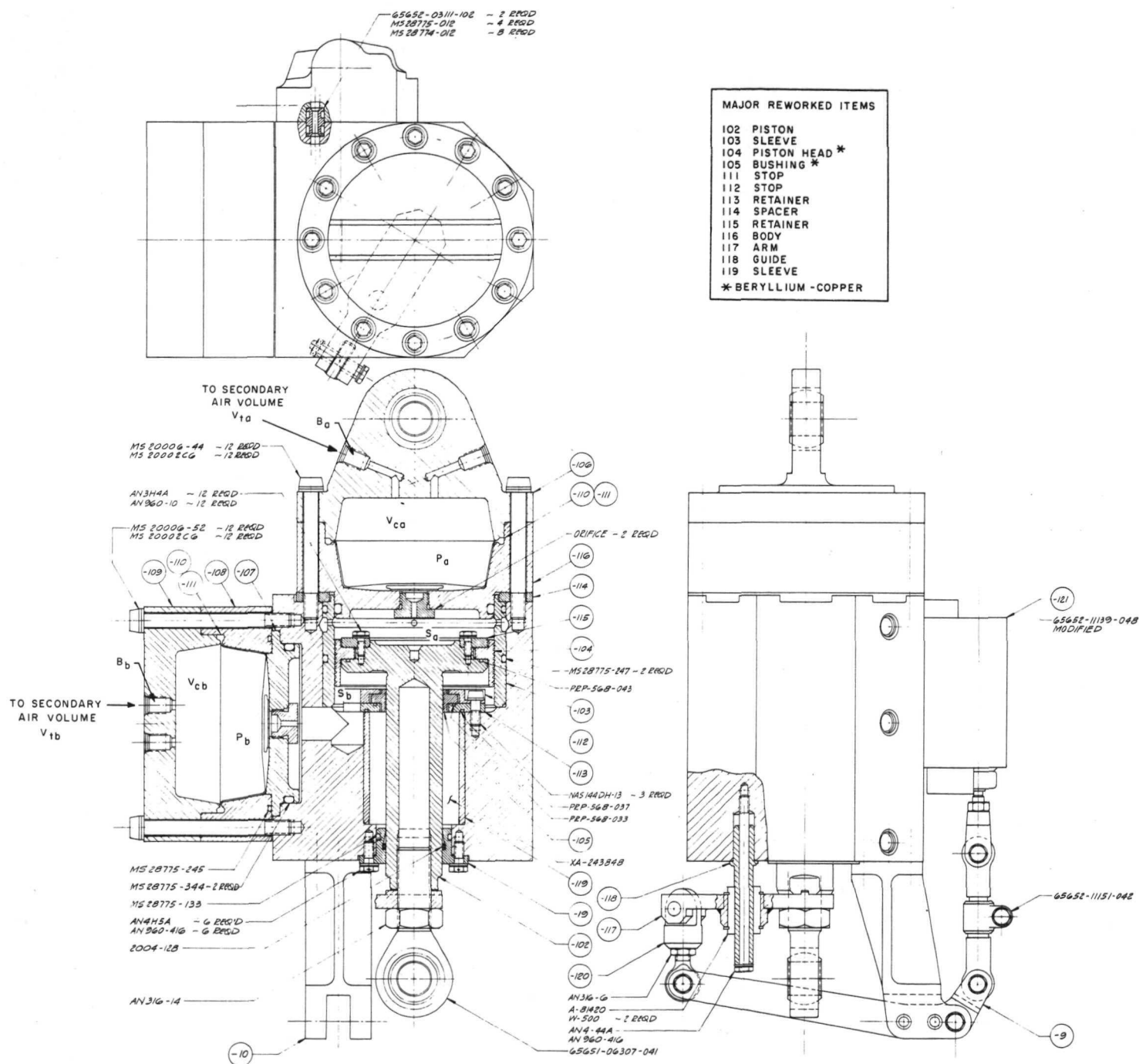


Figure 3. - Active Isolator Assembly Drawing.



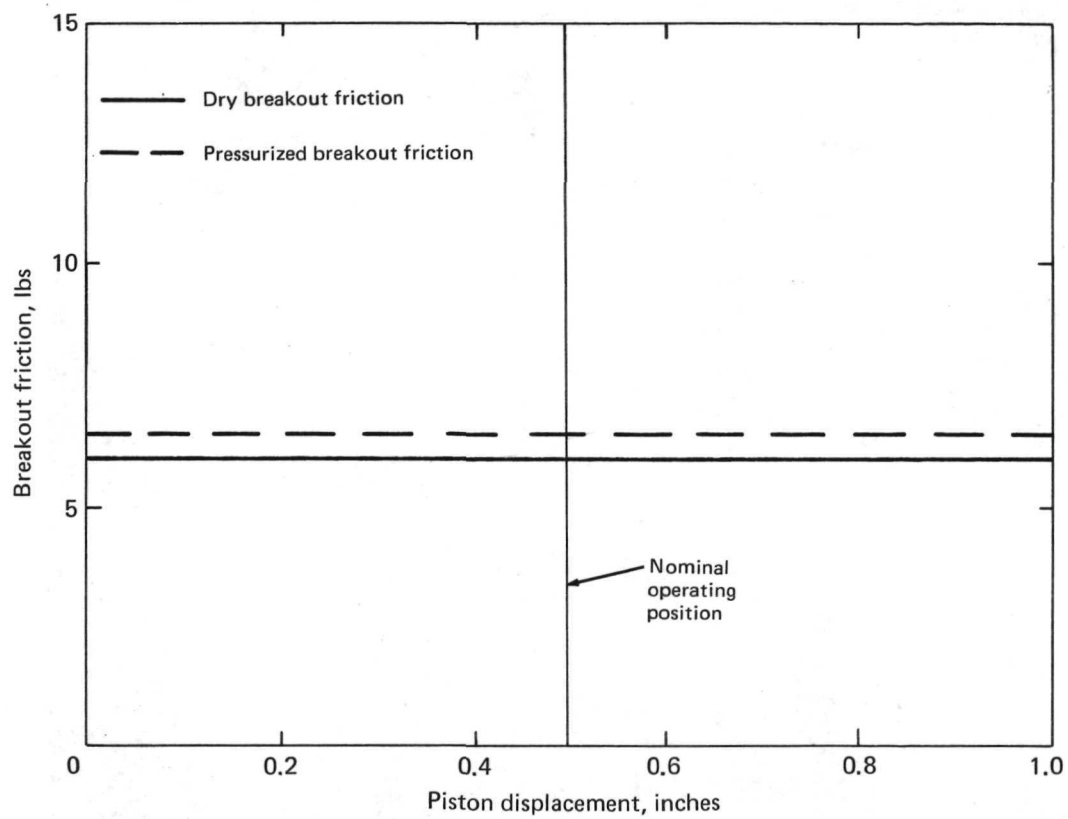


Figure 4. - Piston Breakout Loading.



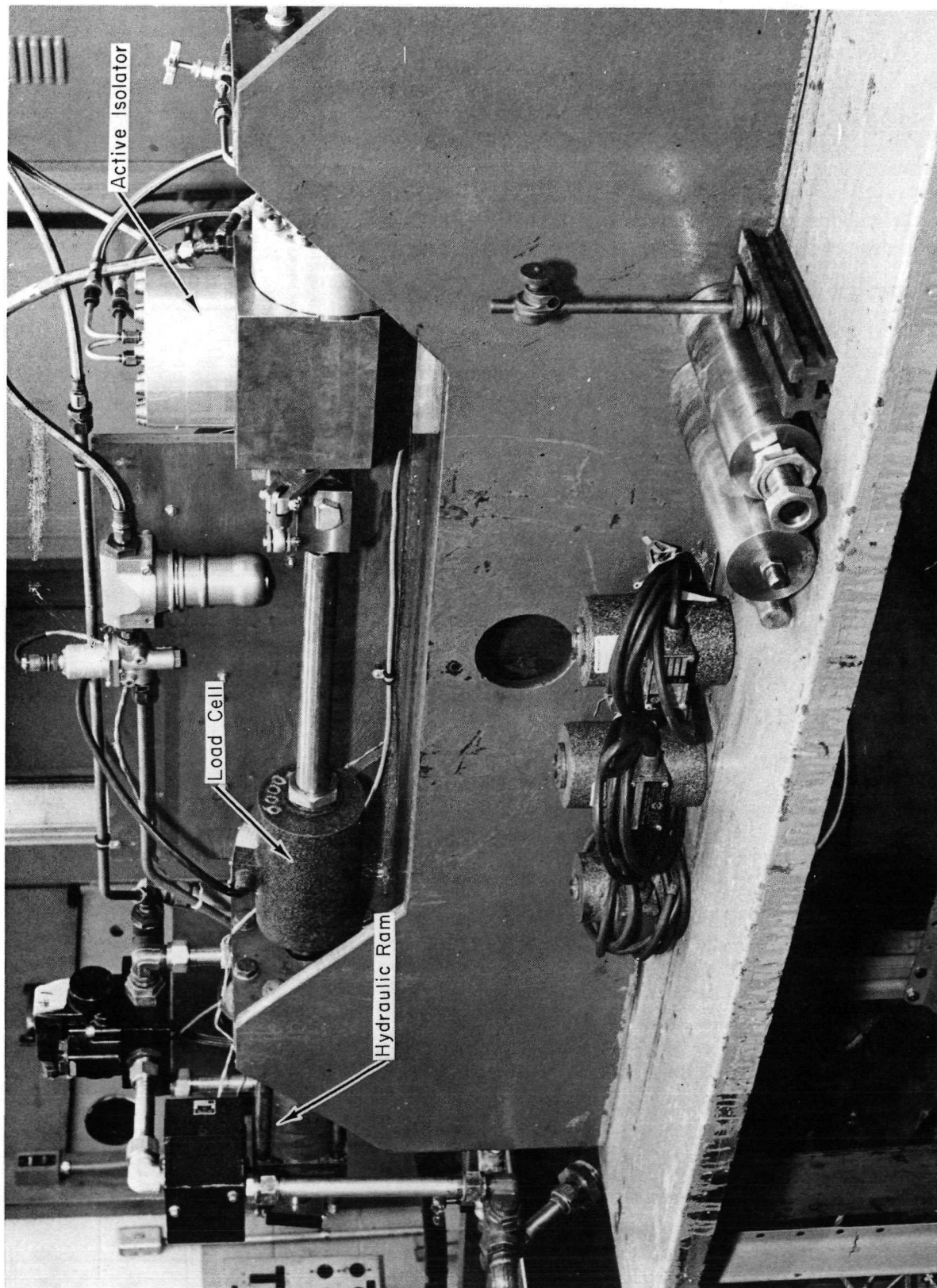


Figure 5. - Bench Test Facility.



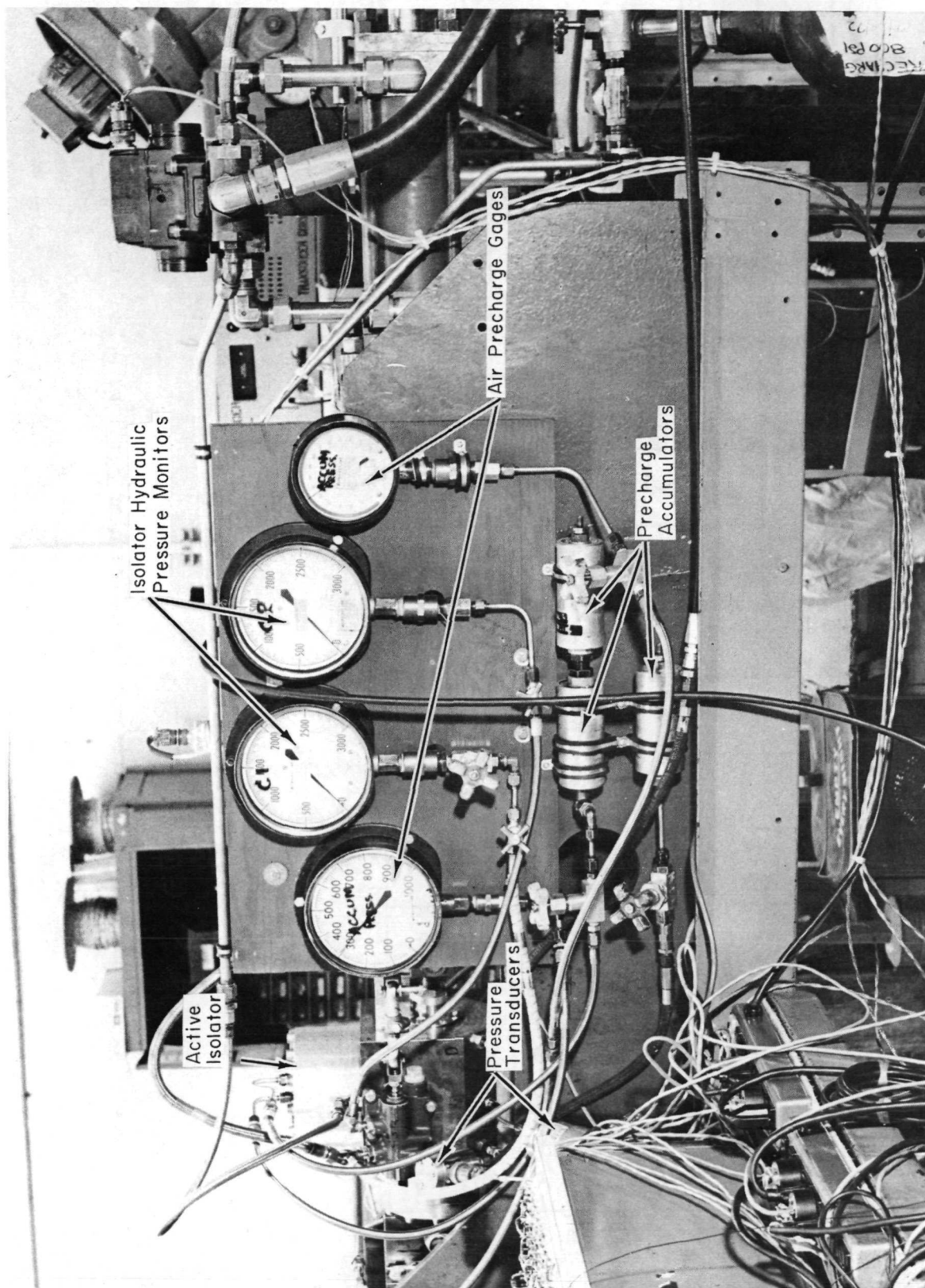


Figure 6. - Isolator Instrumentation.

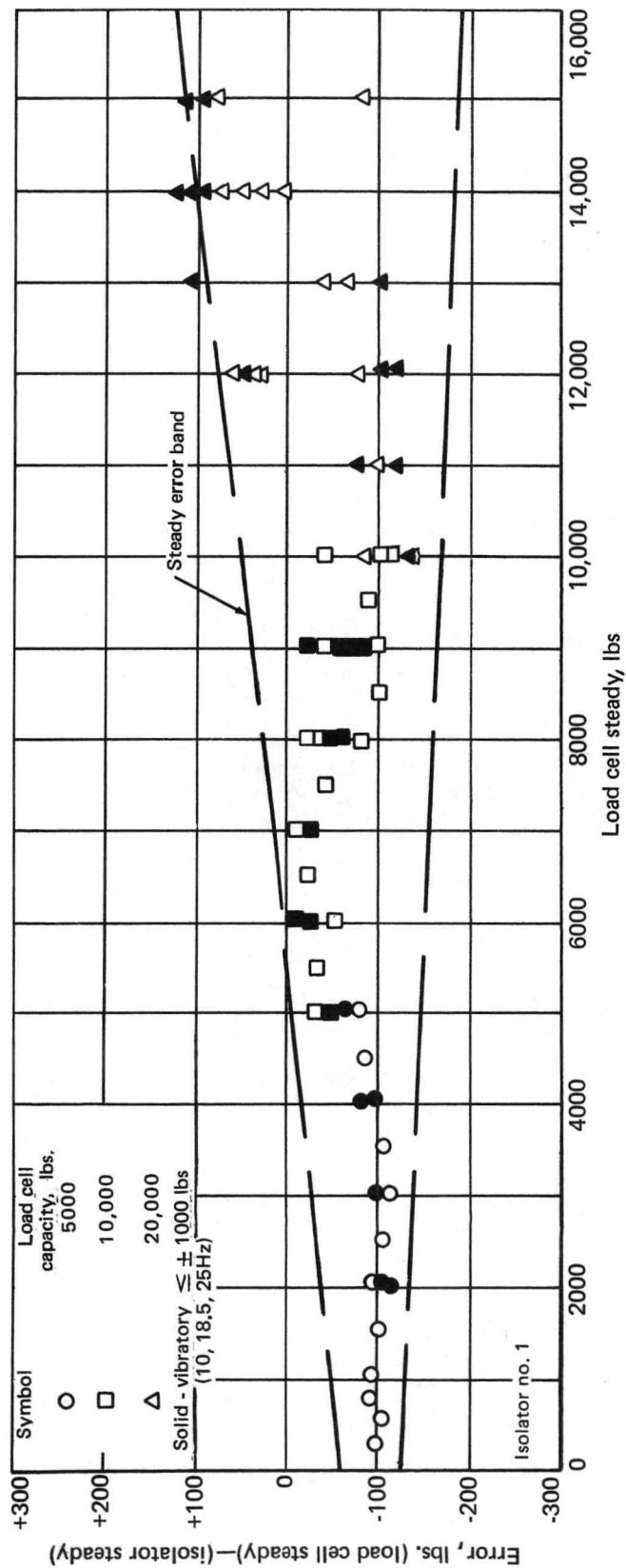


Figure 7. - Steady Load Calibration at Basic Isolator Settings - Isolator No. 1.

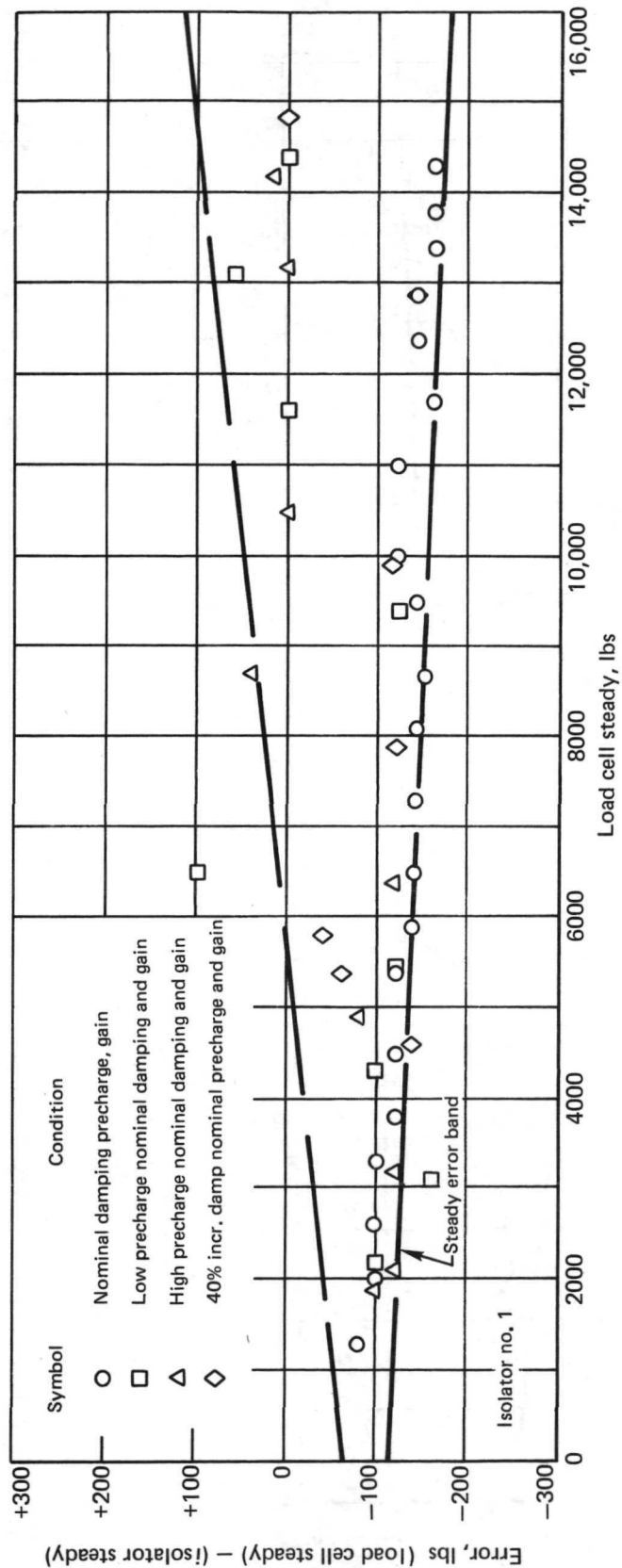


Figure 8. - Transient Load Calibration - Isolator No. 1.

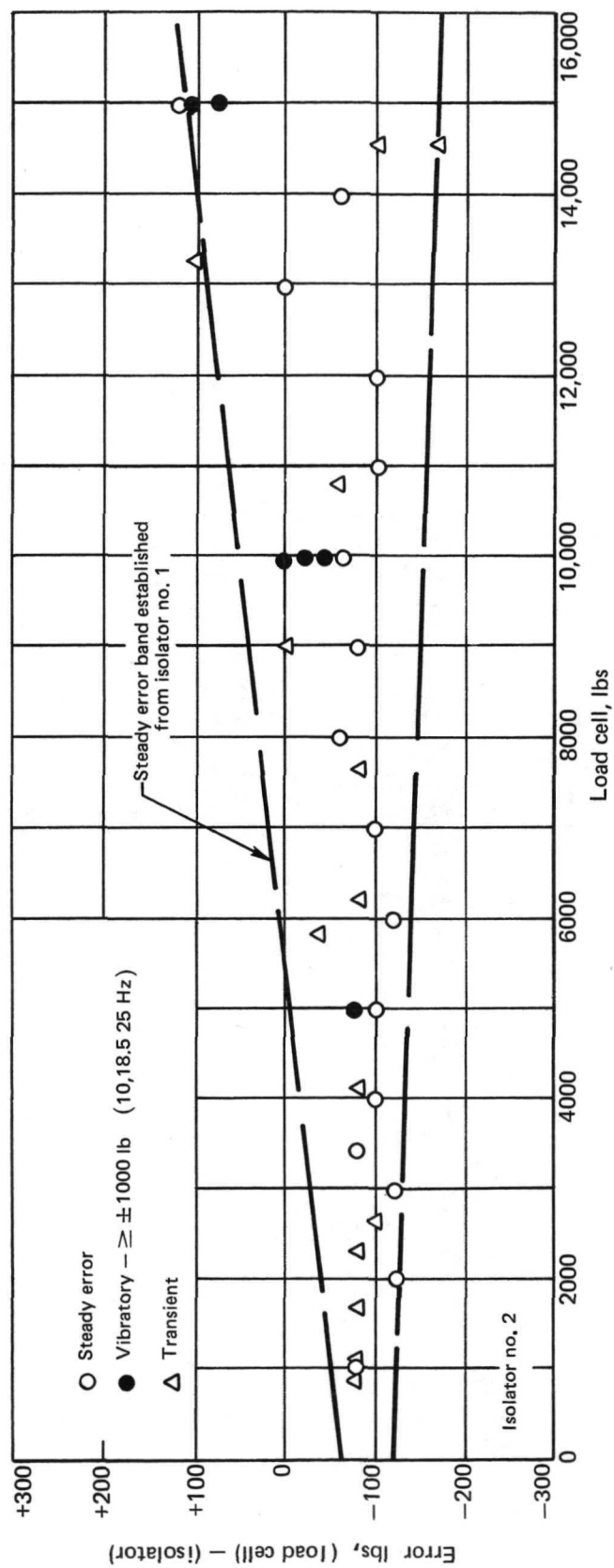


Figure 9. - Steady Load Calibration at Basic Isolator Settings -- Isolator No. 2.

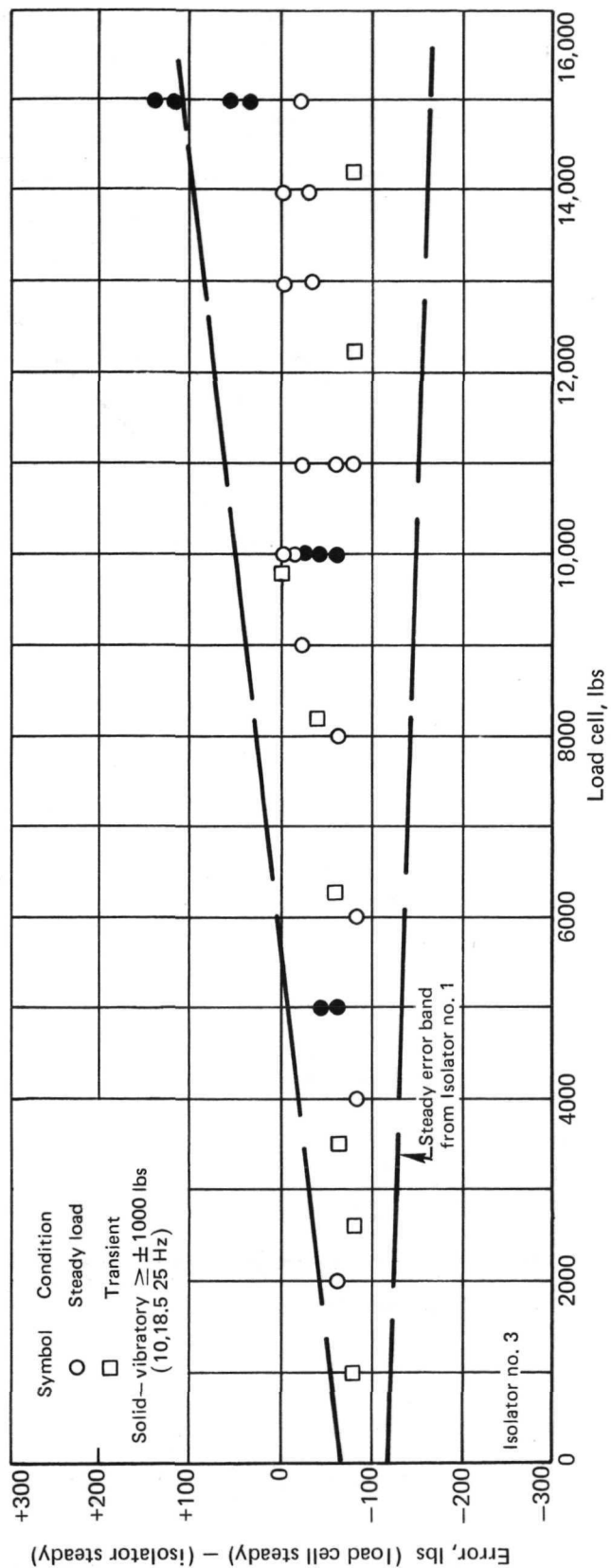


Figure 10. - Steady Load Calibration at Basic Isolator Settings - Isolator No. 3

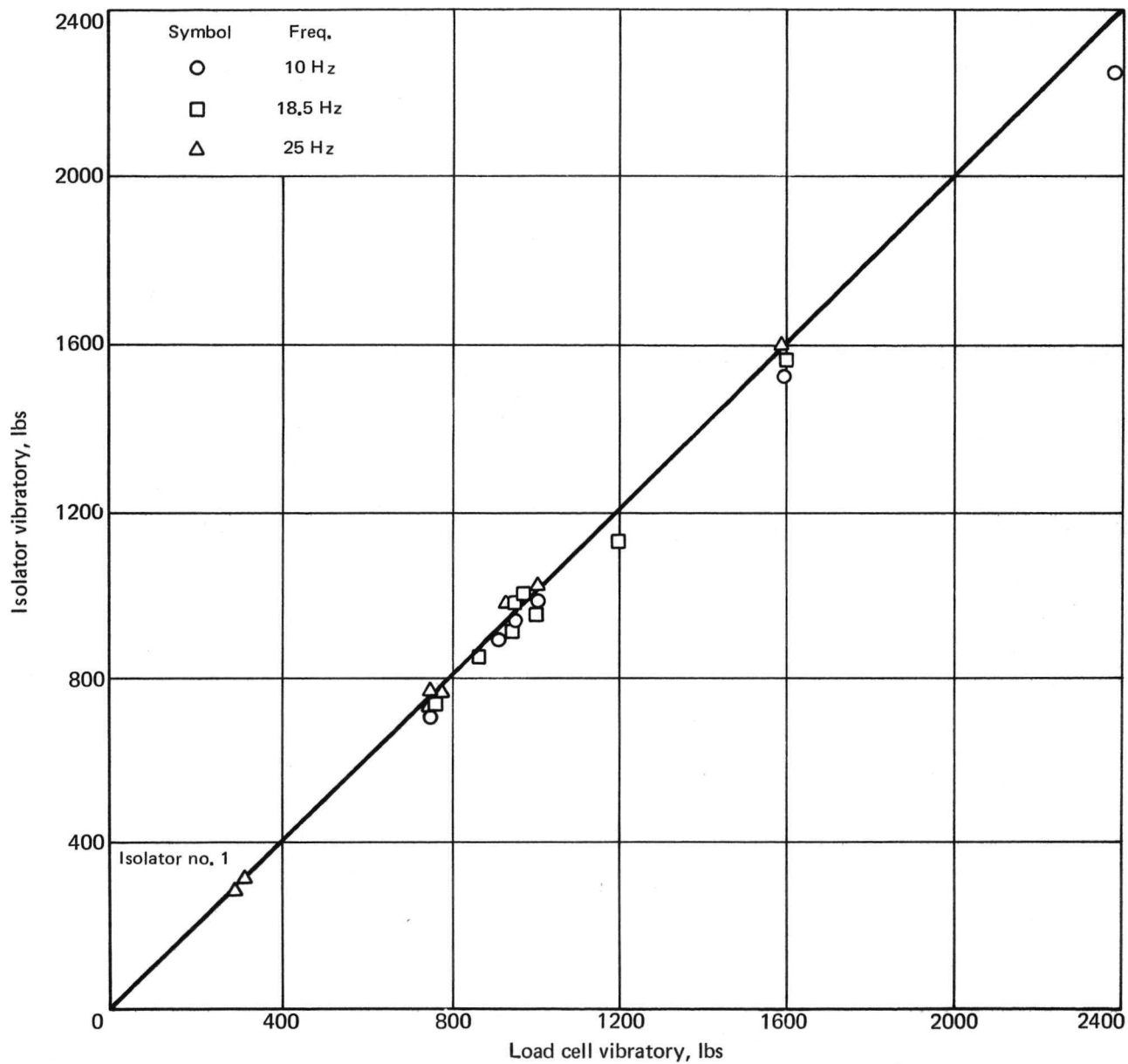


Figure 11. - Vibratory Load Calibration at Basic Isolator Settings - Isolator No. 1.

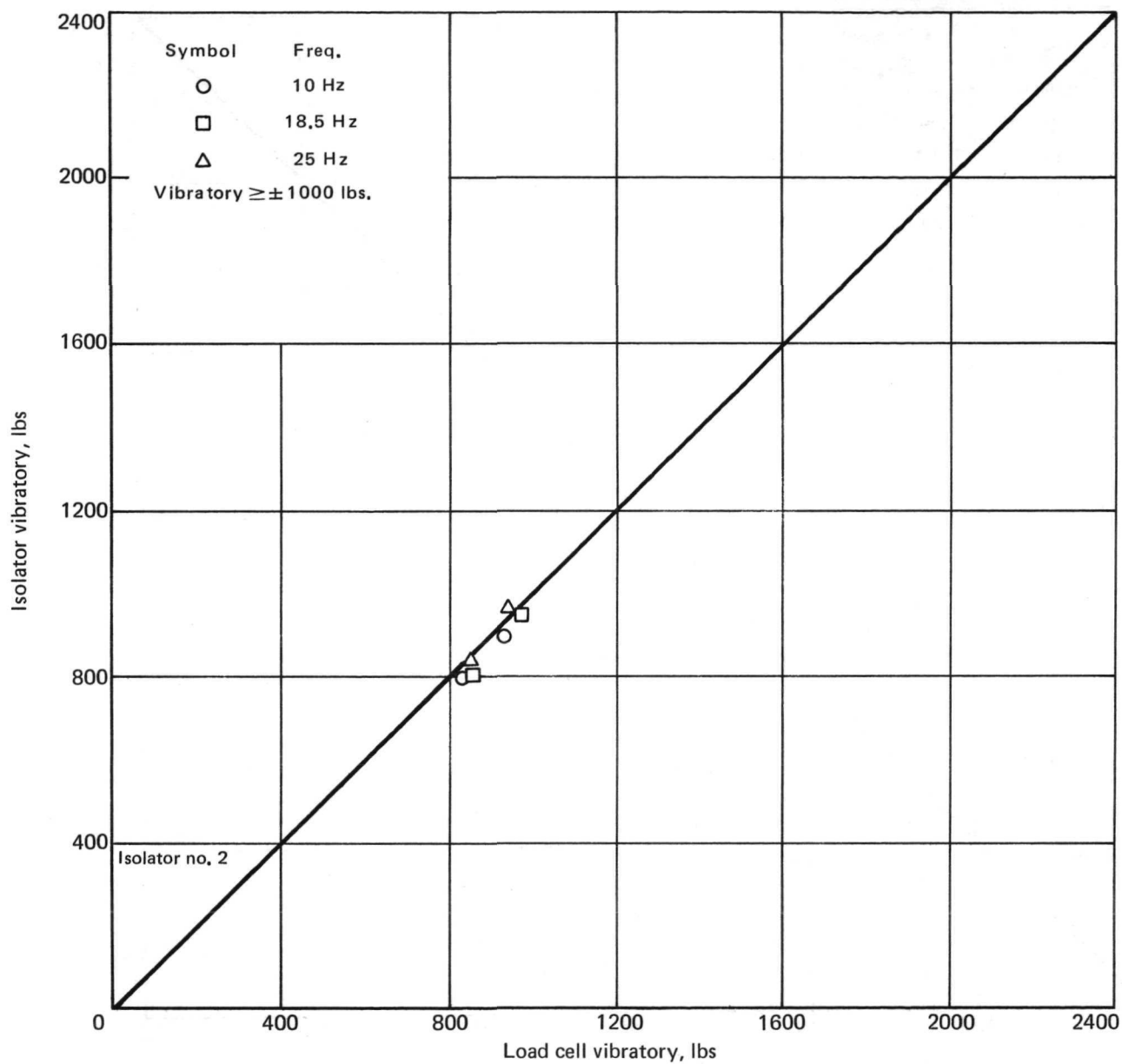


Figure 12. - Vibratory Load Calibration at Basic Isolator Settings - Isolator No. 2.

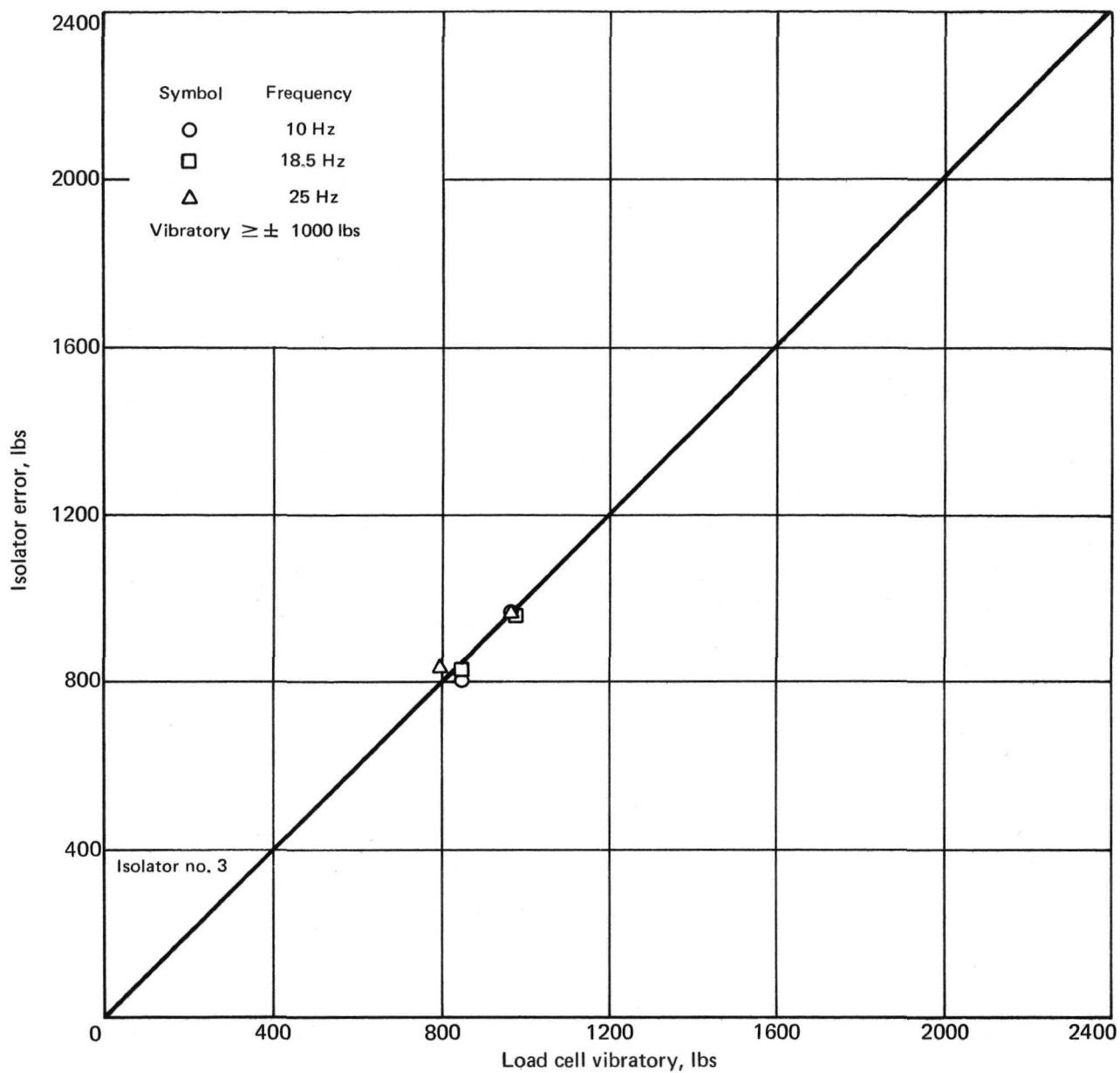


Figure 13. - Vibratory Load Calibration at Basic Isolator Settings - Isolator No. 3.



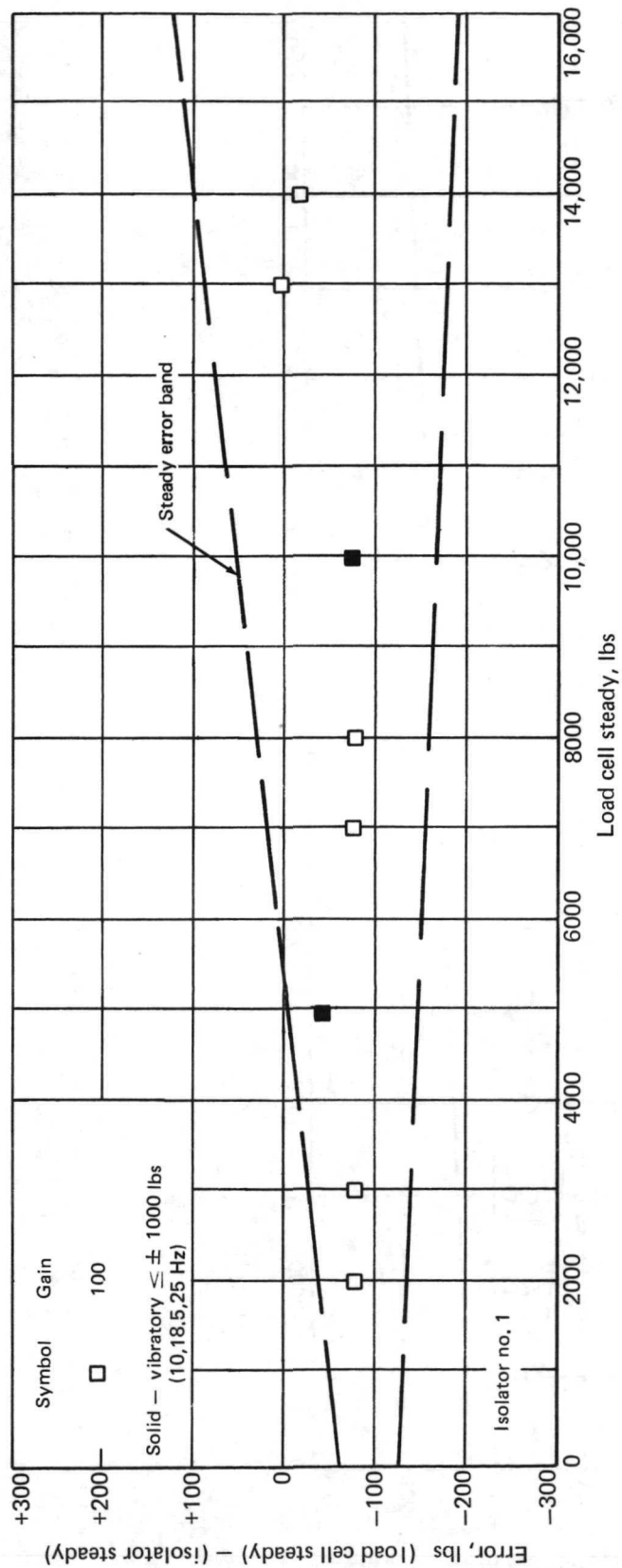


Figure 14. - Steady Load Calibration With Increased Precharge ( $P_A = 500$  psia and  $P_B = 1820$  psia) and Gain Variation - Isolator No. 1.

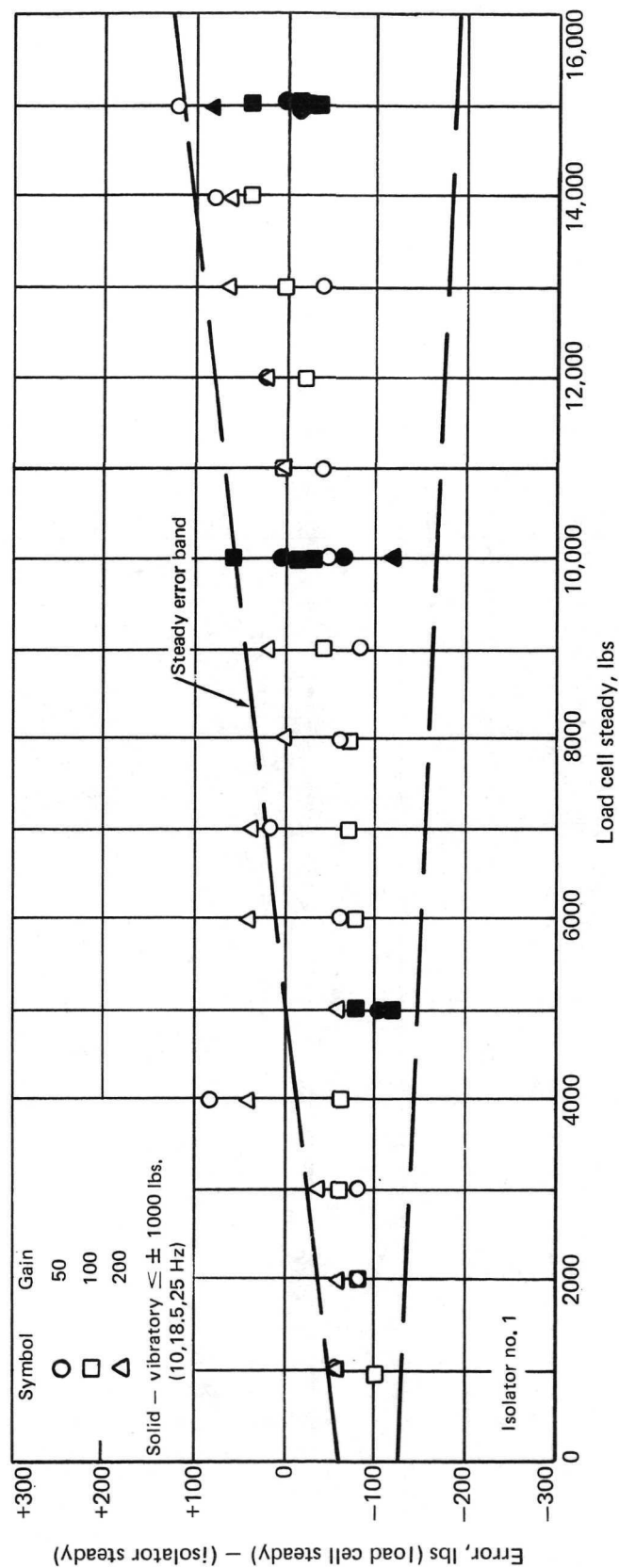


Figure 15. - Steady Load Calibration With Reduced Precharge ( $P_A = 150$  psia and  $P_B = 550$  psia) and Gain Variation Isolator No. 1.

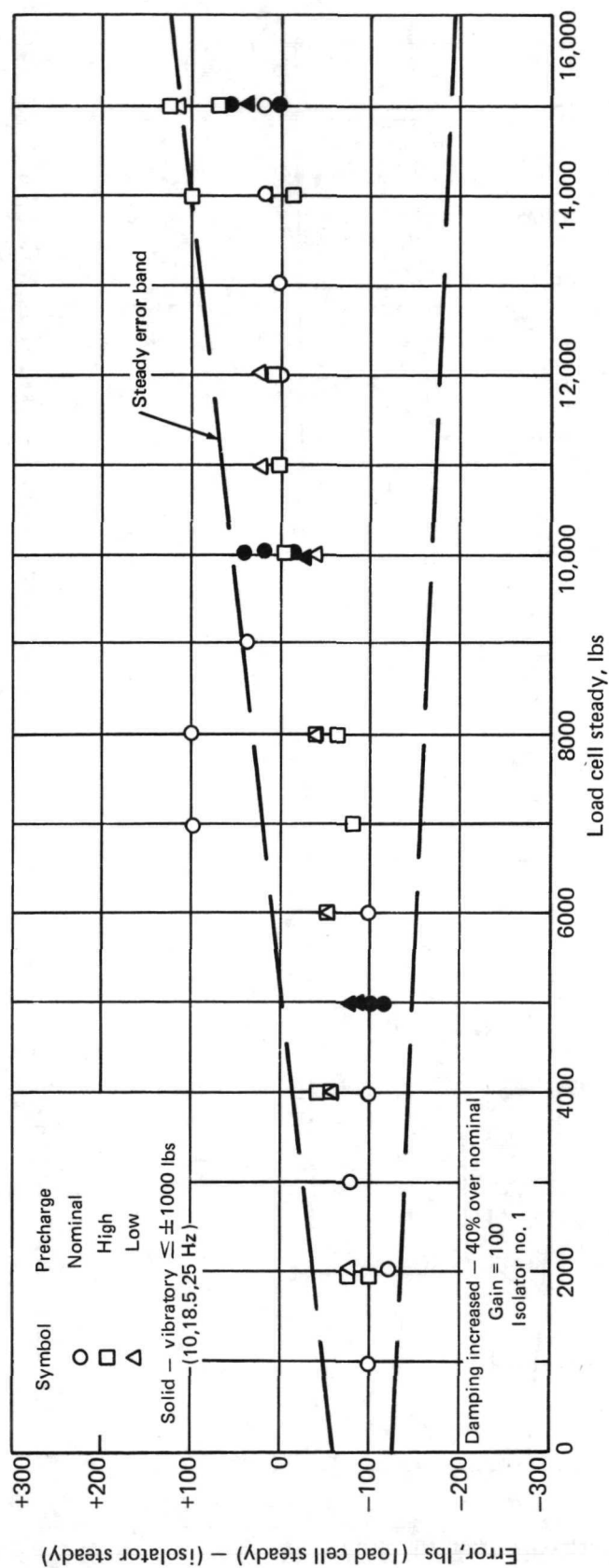
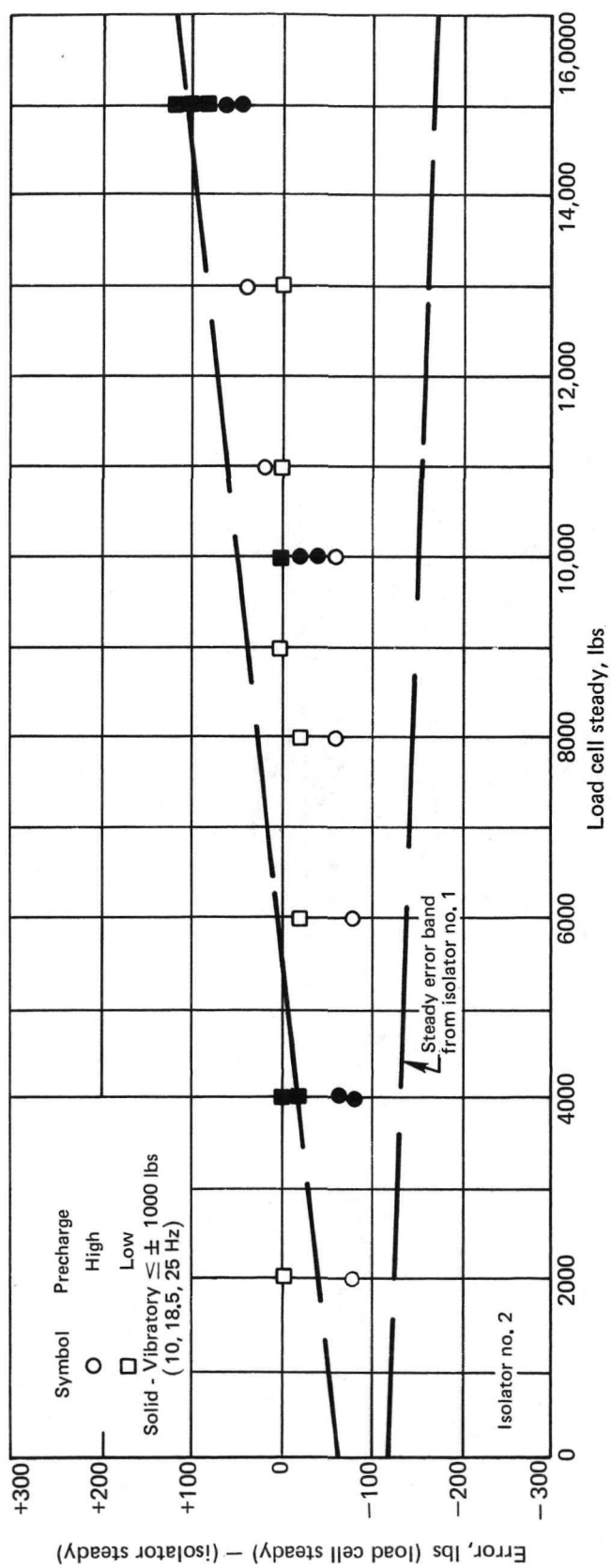


Figure 16. - Steady Load Calibration With Precharge and Damping Variations - Isolator No. 1.



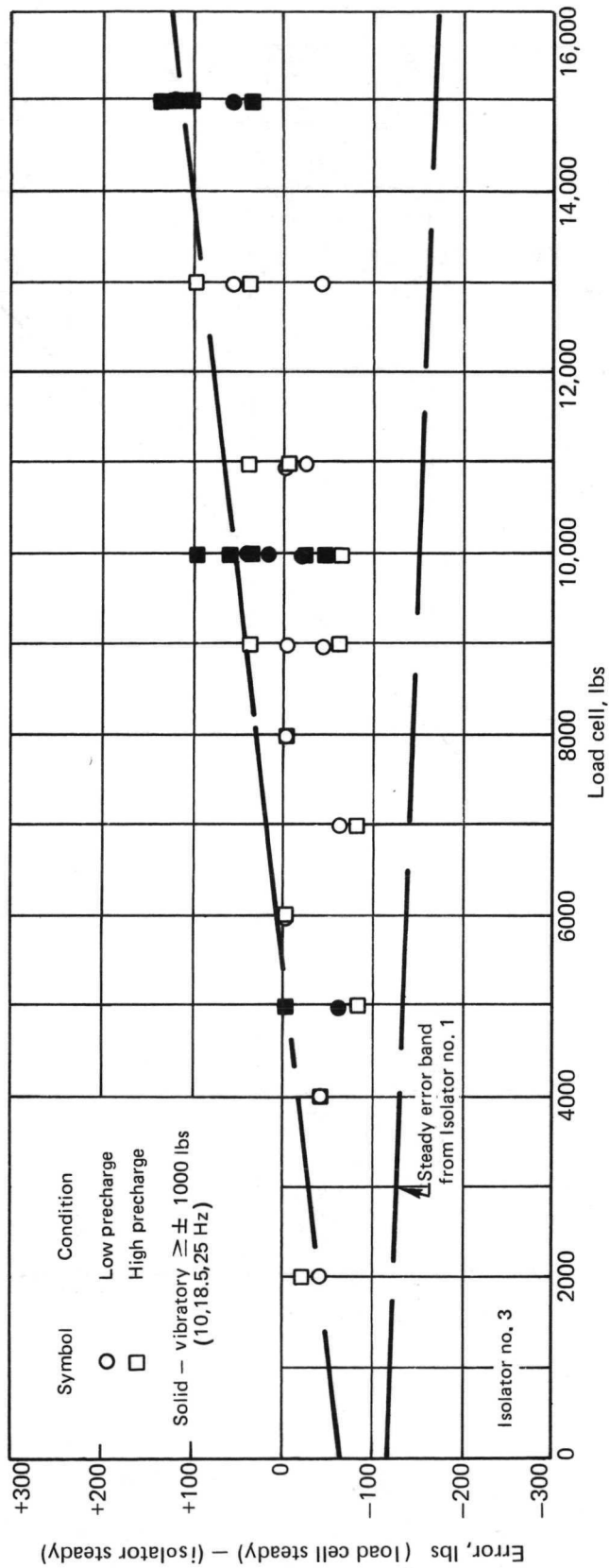


Figure 18. - Steady Load Calibration With Precharge Variations - Isolator No. 3.

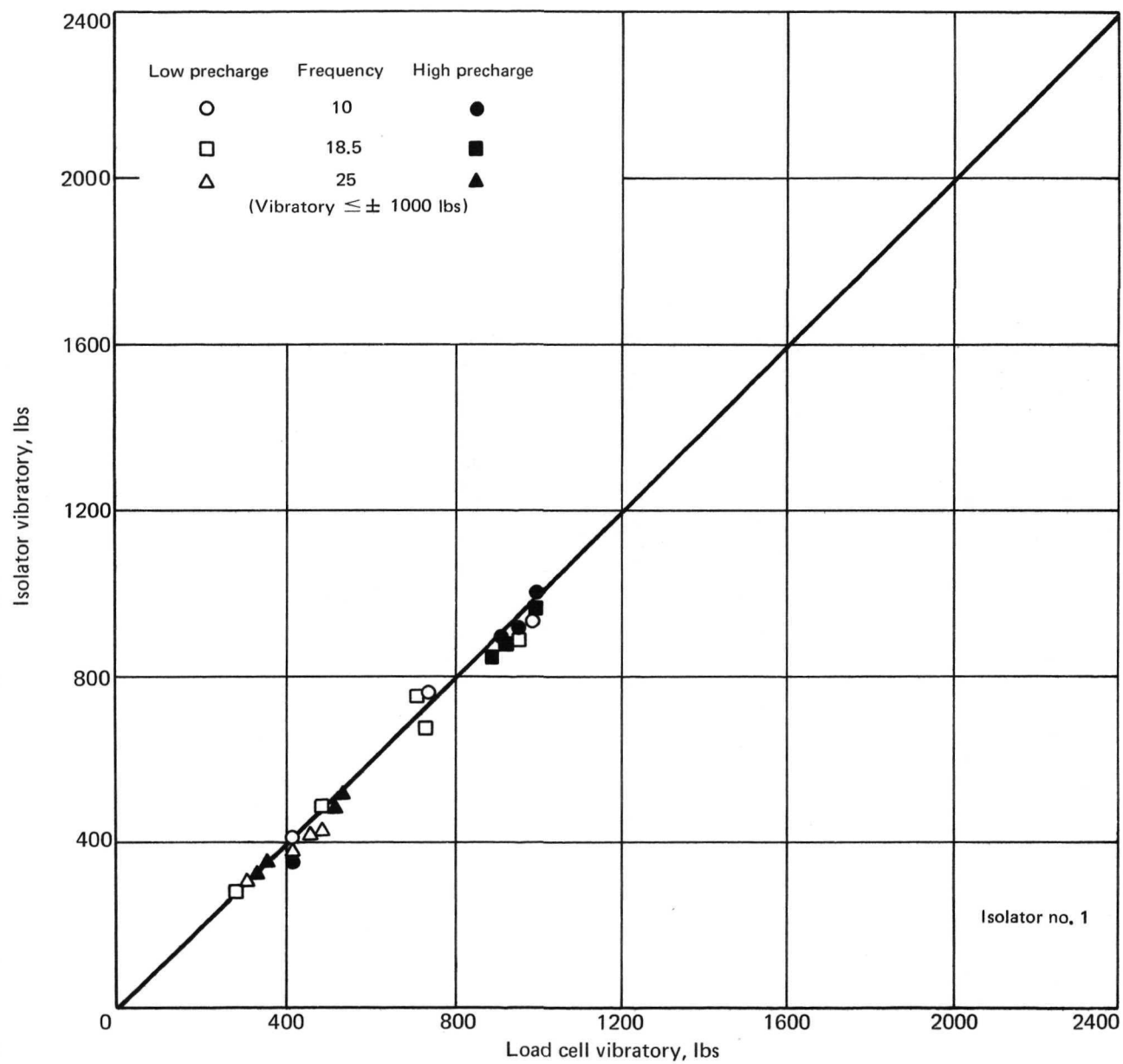


Figure 19. - Vibratory Load Calibration With Precharge Variations - Isolator No. 1.

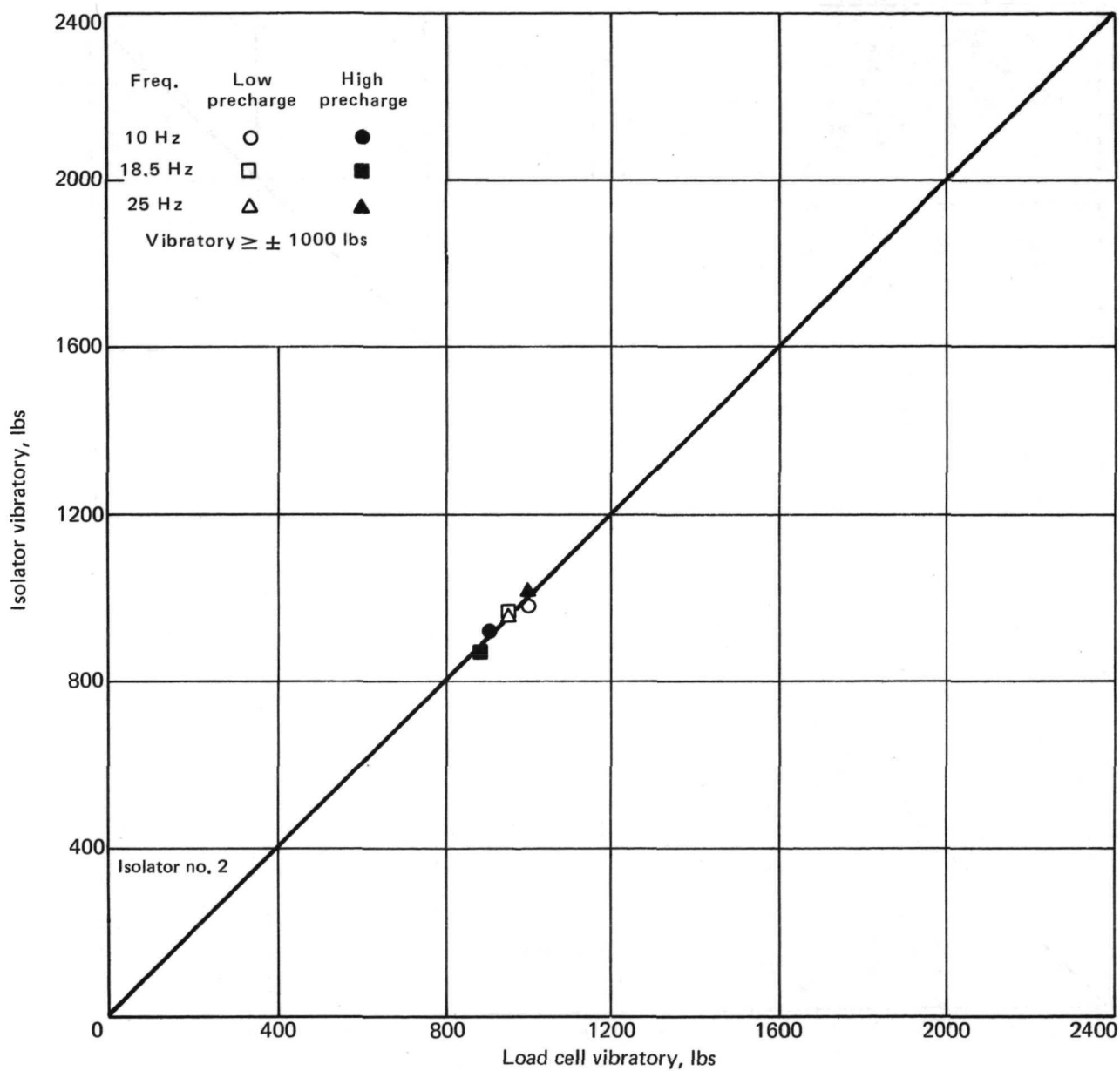


Figure 20. - Vibratory Load Calibration With Precharge Variations - Isolator No. 2.

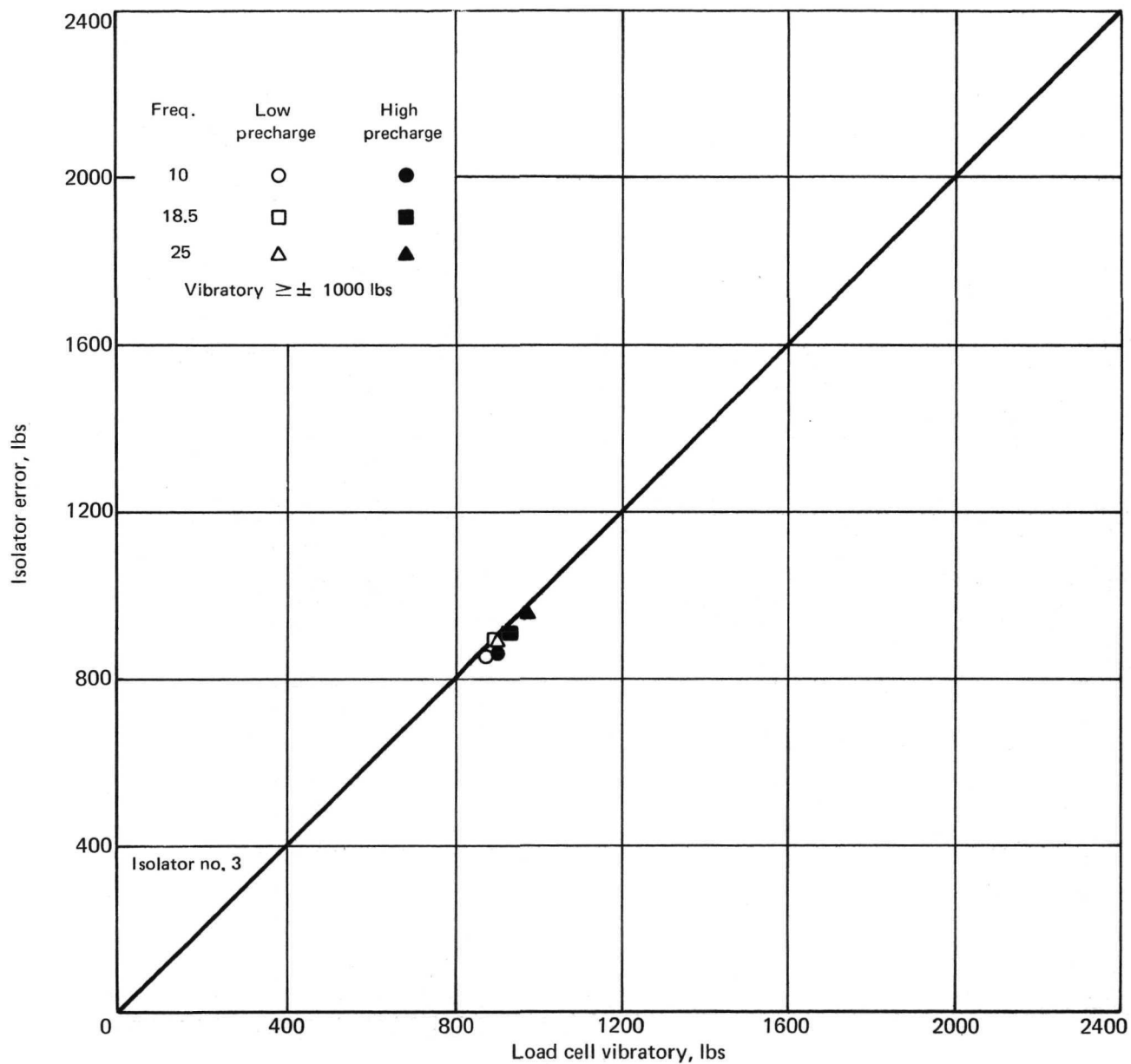


Figure 21. - Vibratory Load Calibration With Precharge Variations - Isolator No. 3.



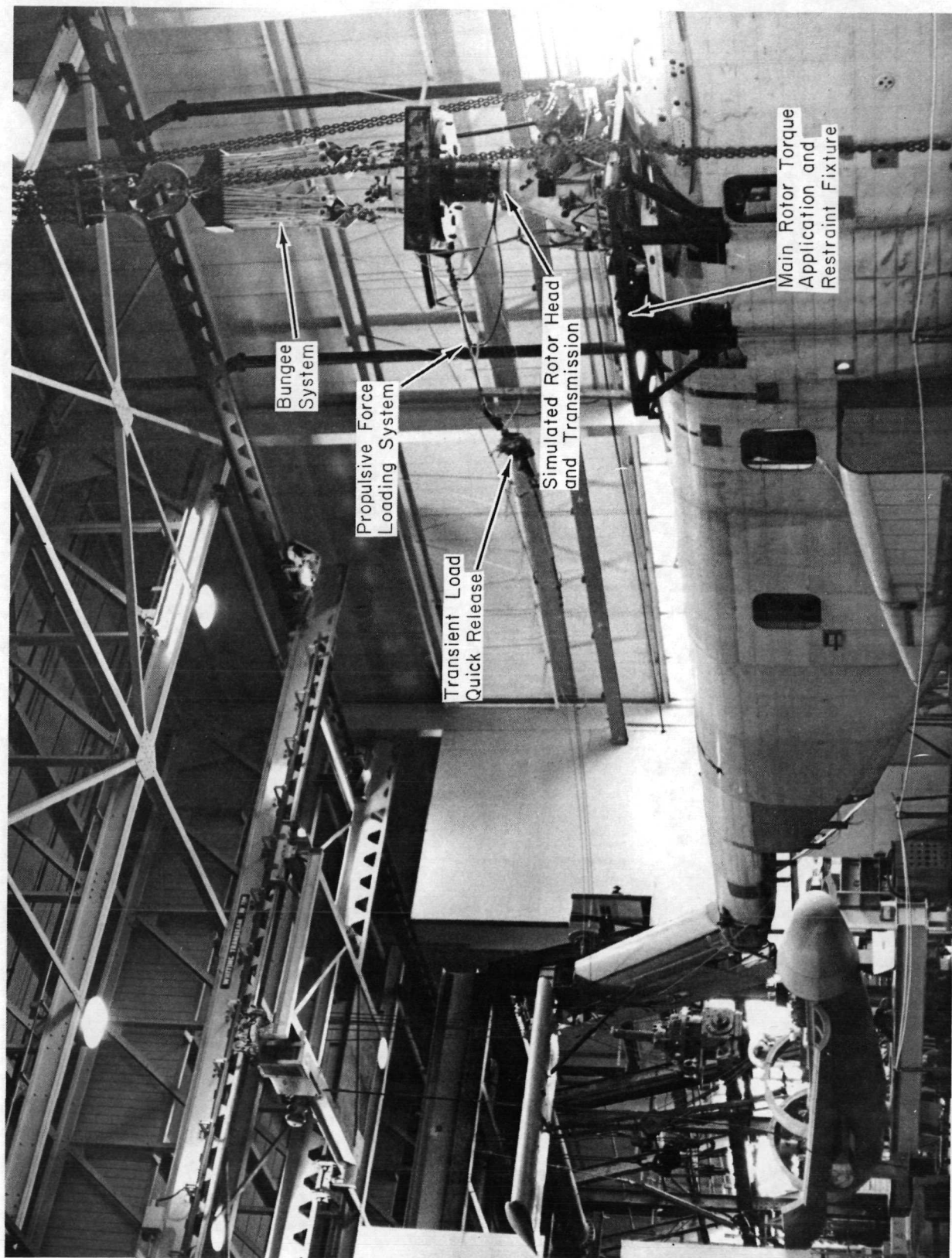


Figure 22. - Active Isolator/Rotor Force Measurement Test Facility.

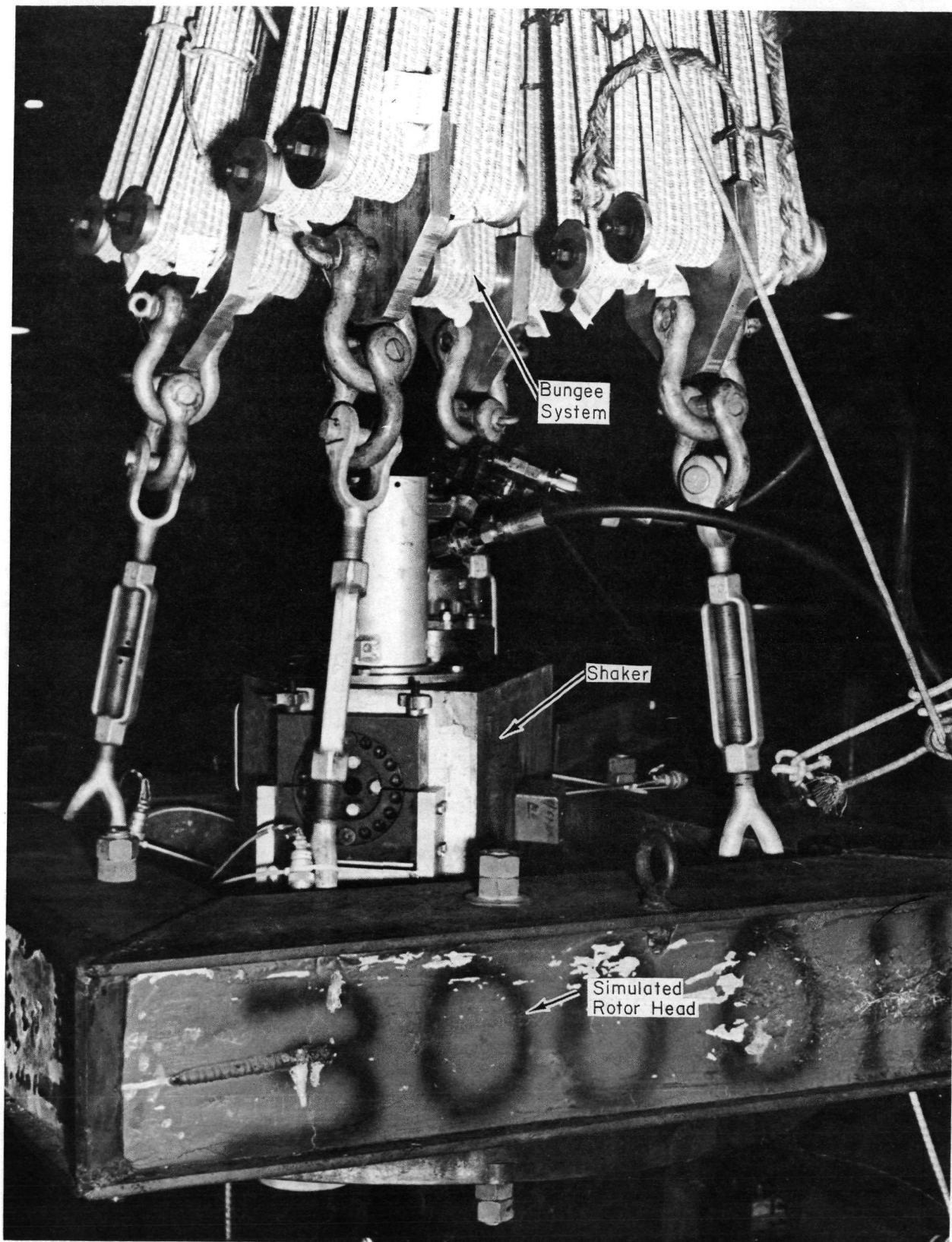


Figure 23. - Unidirectional Hydraulic Shaker Installation.

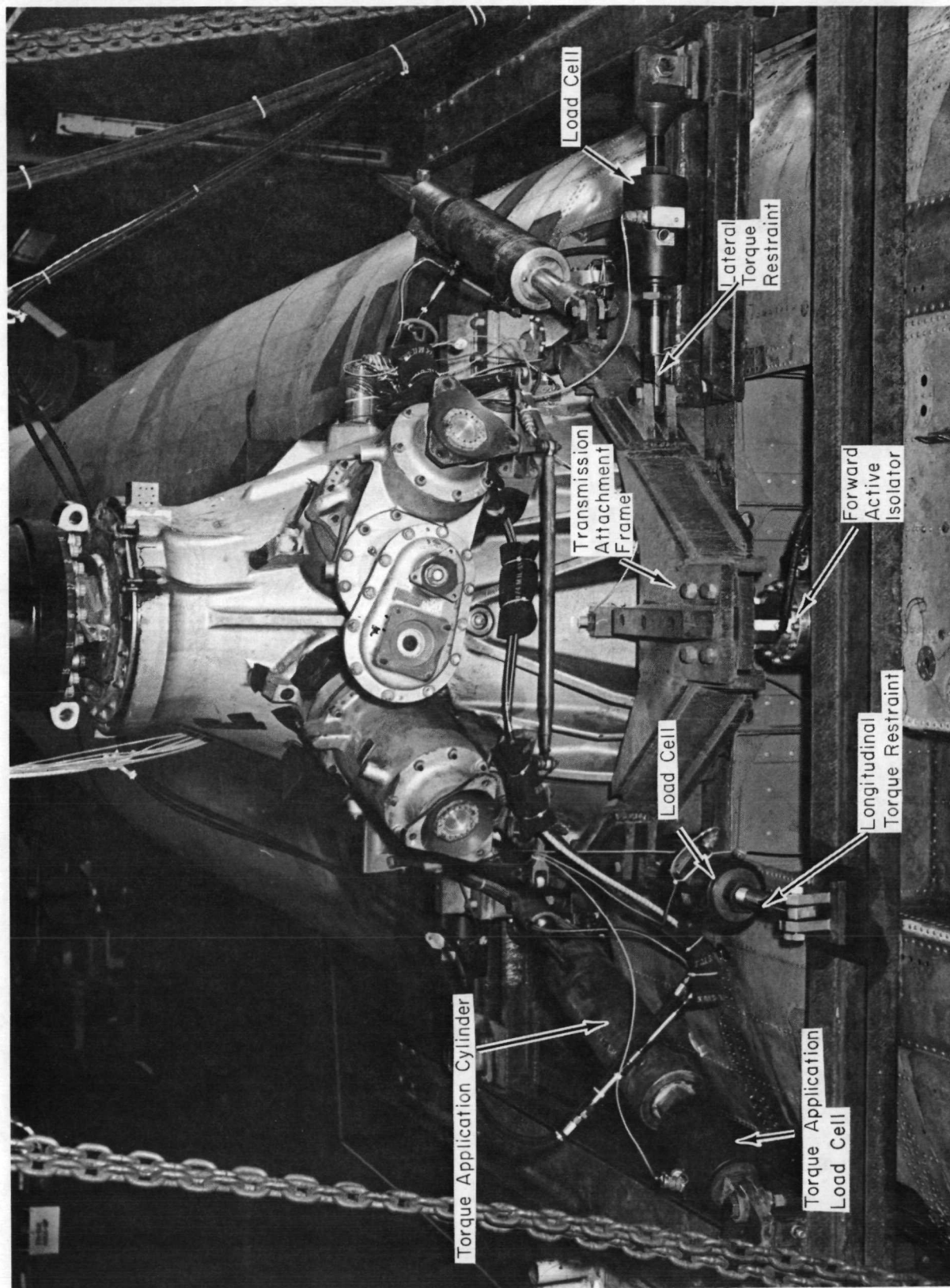


Figure 24. - Active Isolation Rotor Force Measurement System Installation.



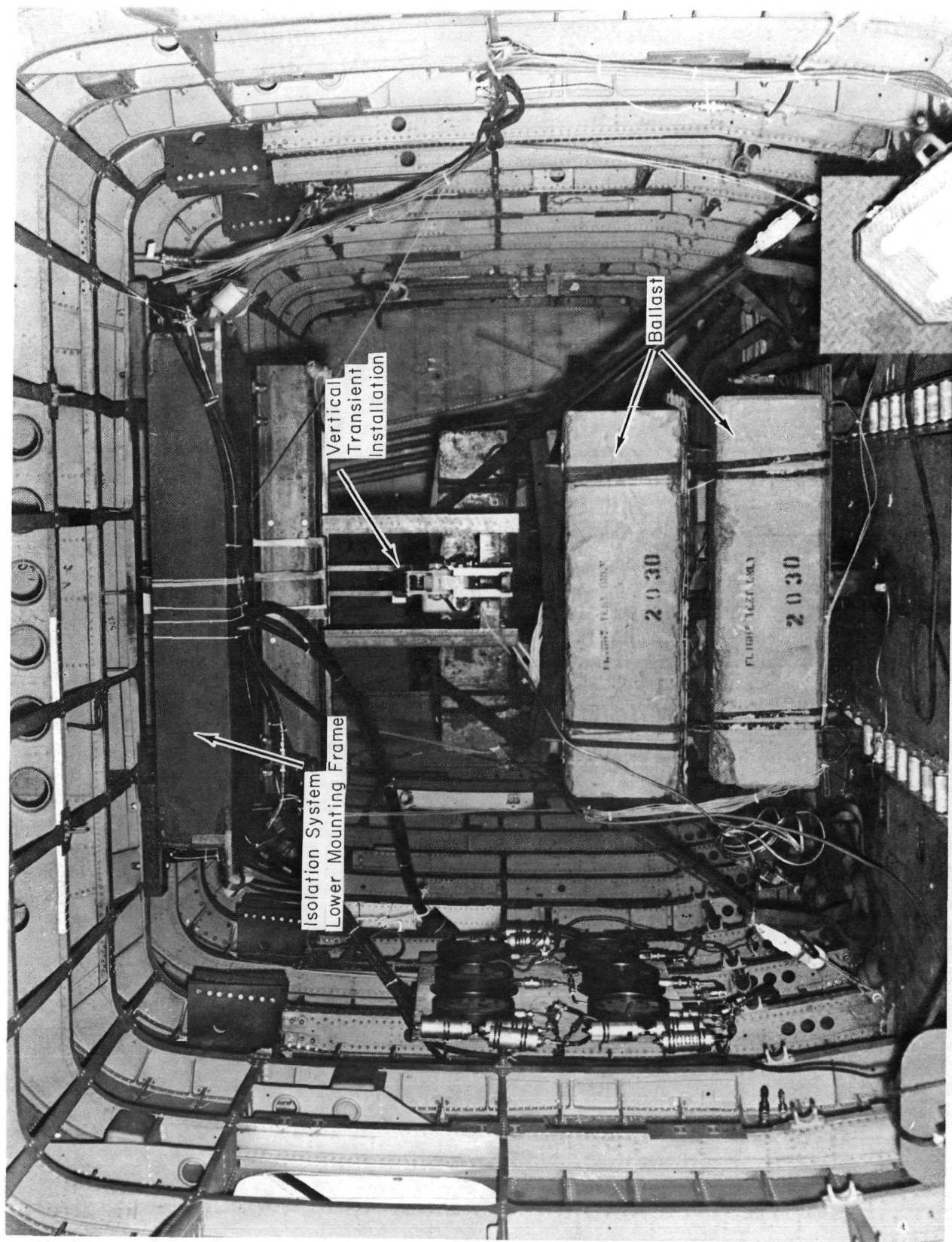


Figure 25. - Cabin Interior.

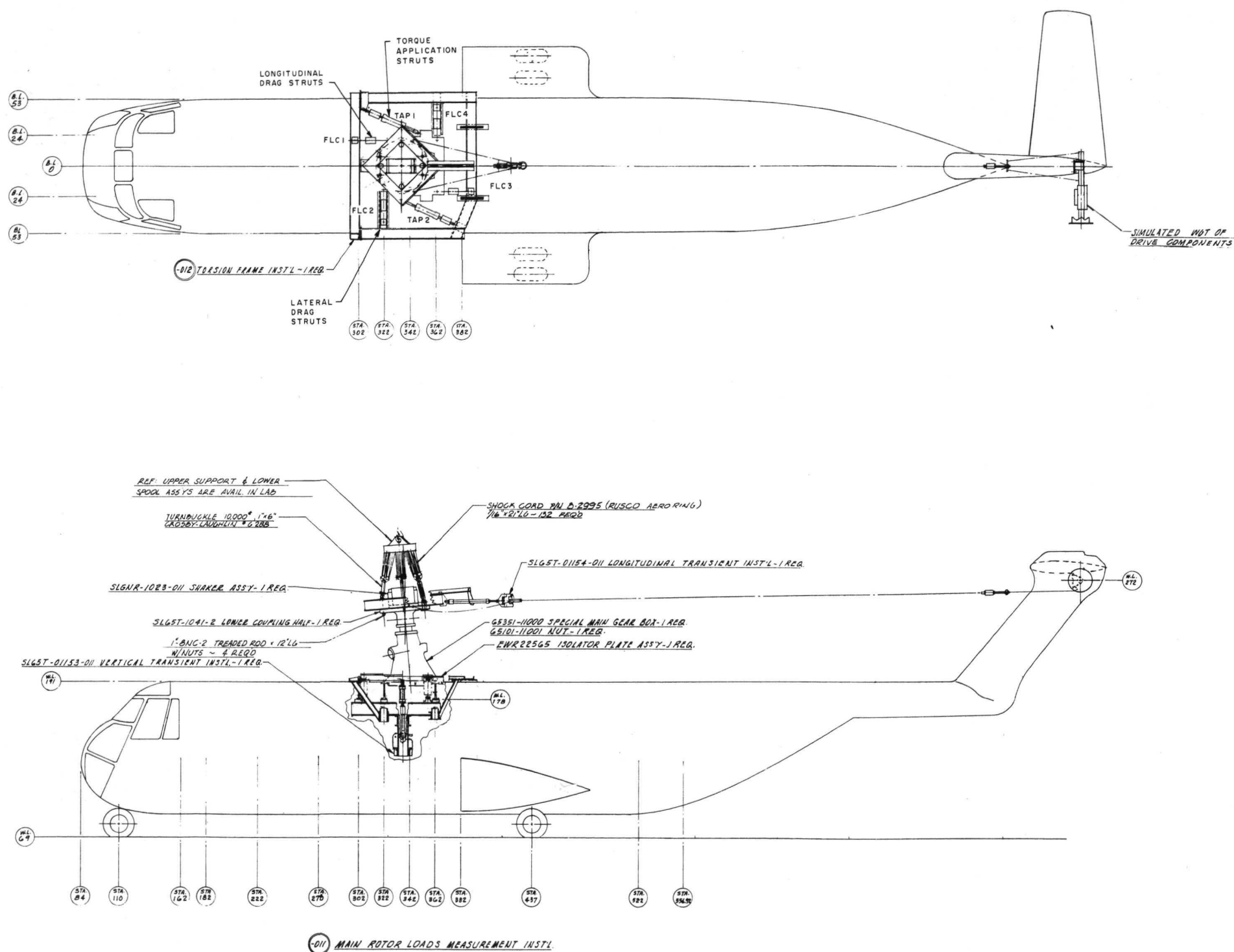


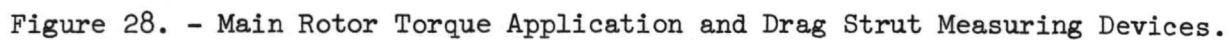
Figure 26. - Test Facility Illustration.













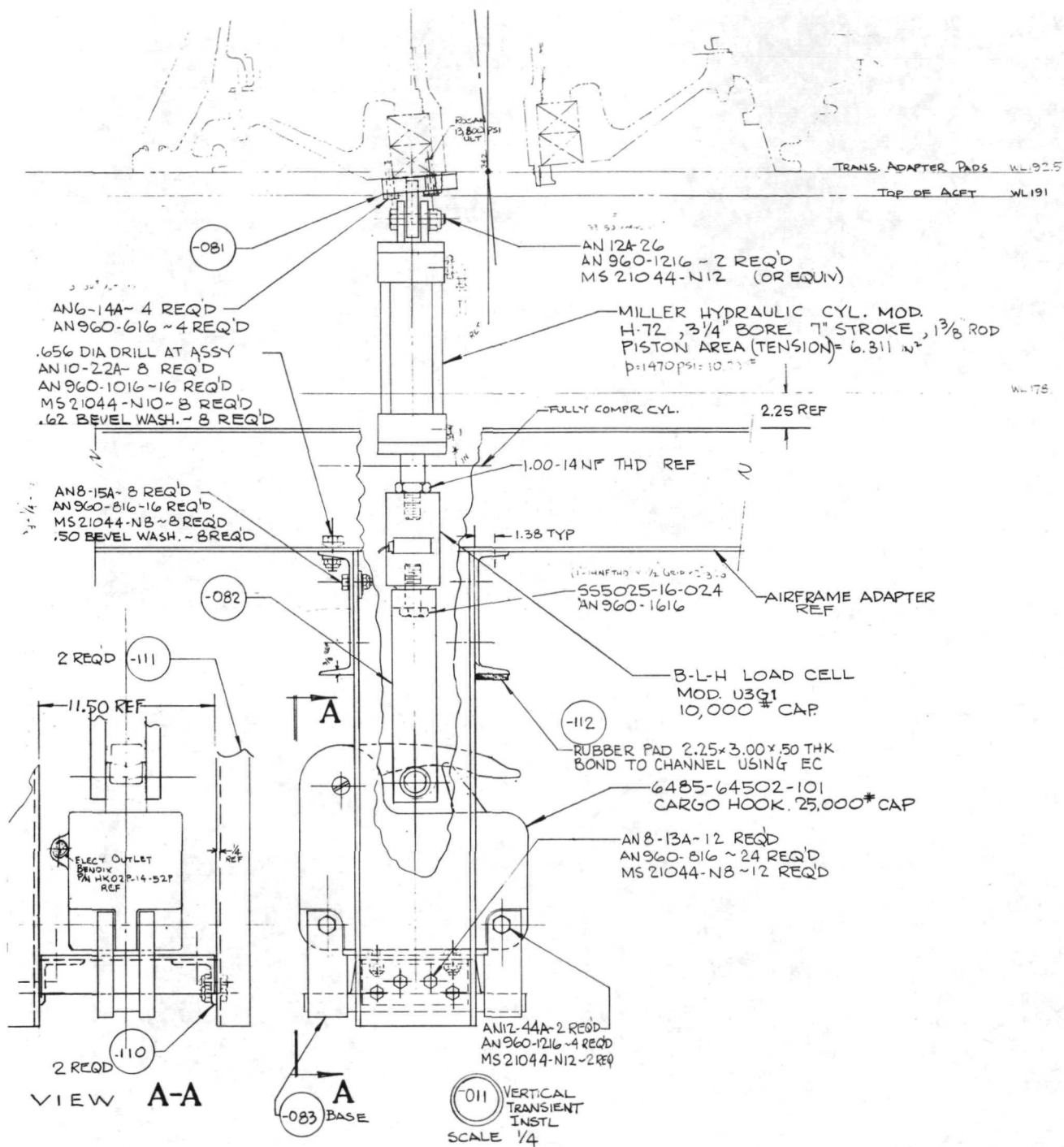


Figure 29. - Vertical Transient Loading System.

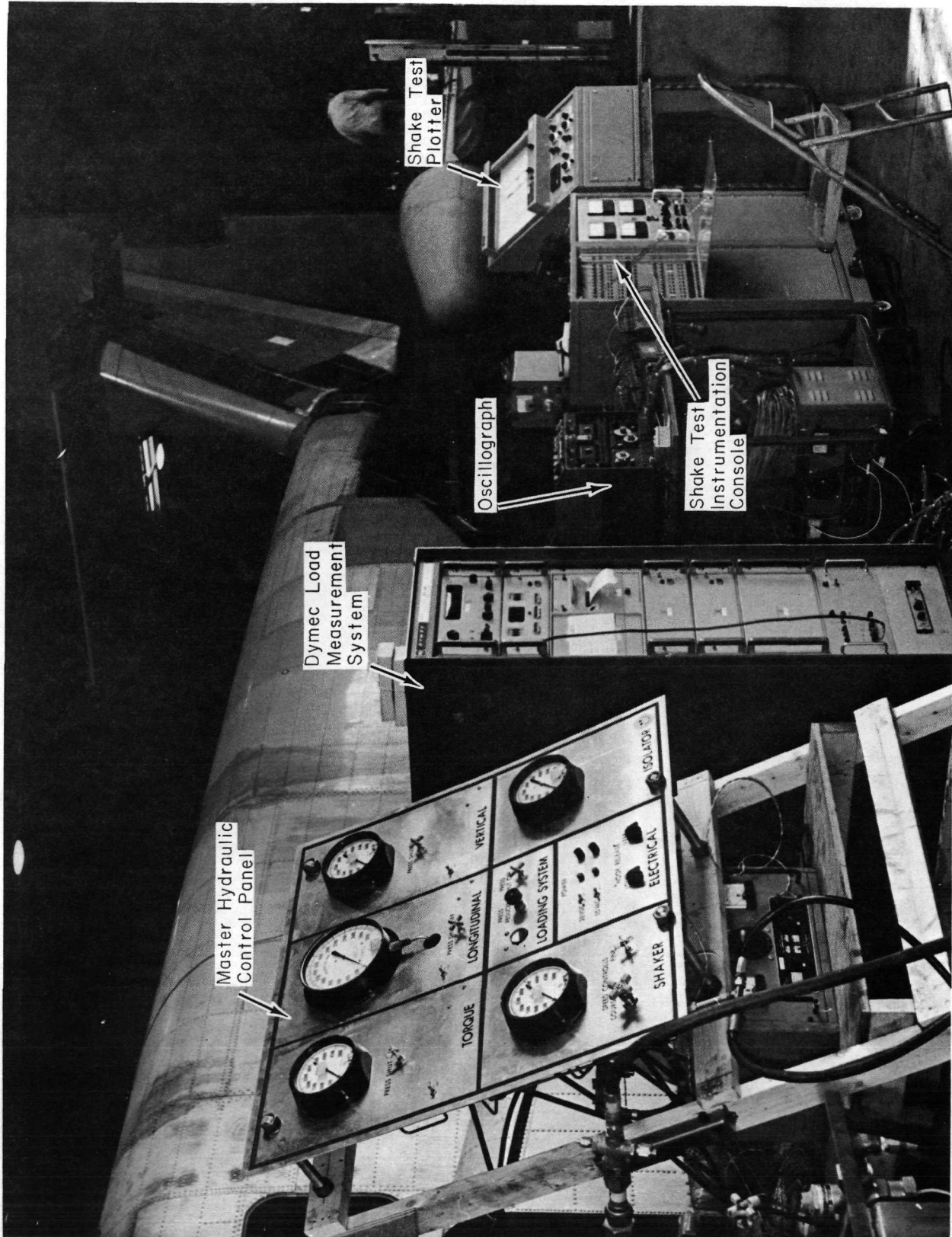


Figure 30. - Master Controls and Instrumentation for Steady and Vibratory Loads.



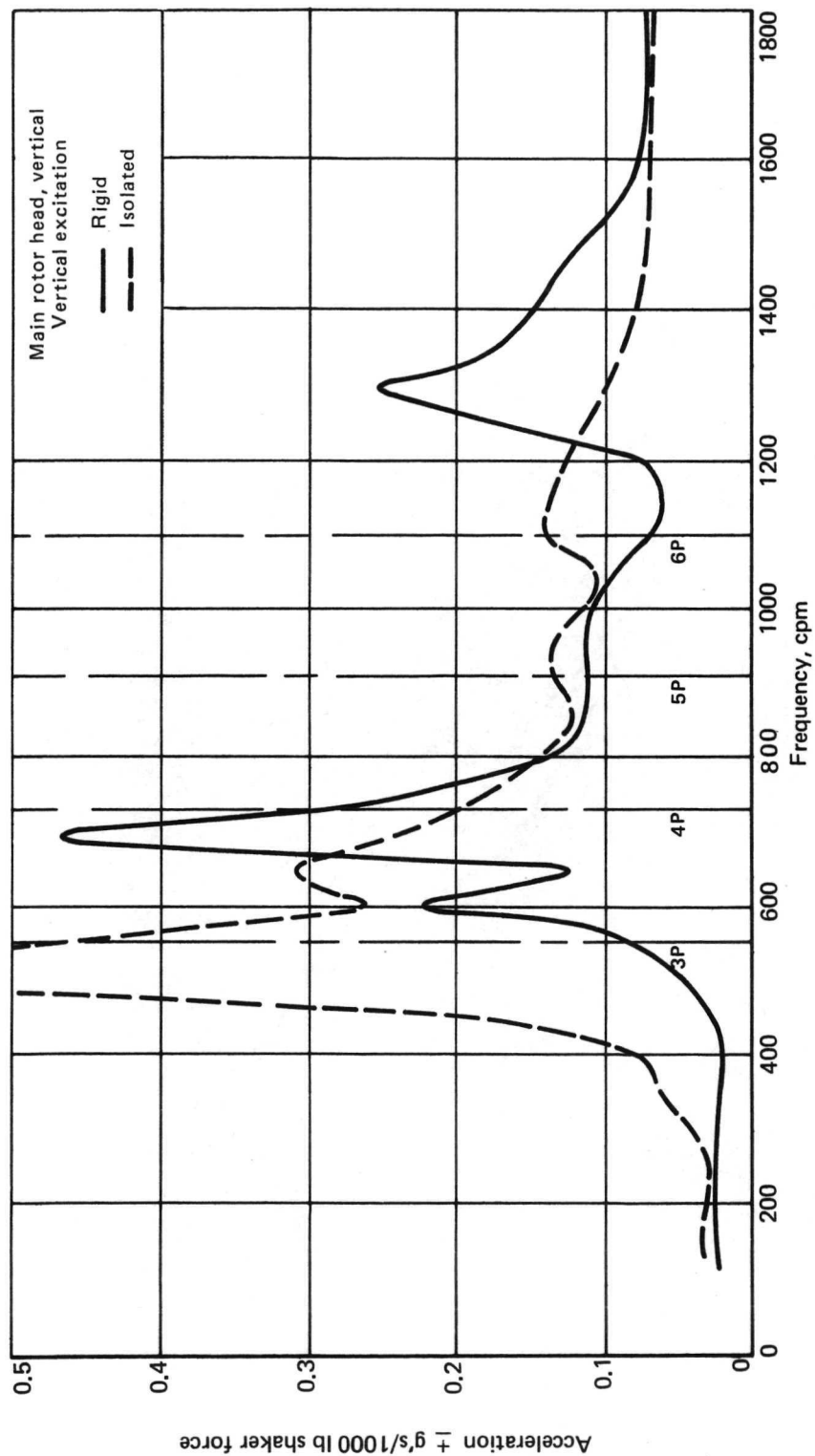


Figure 32. - Main Rotor Head-Vertical Response due to Vertical Excitation.

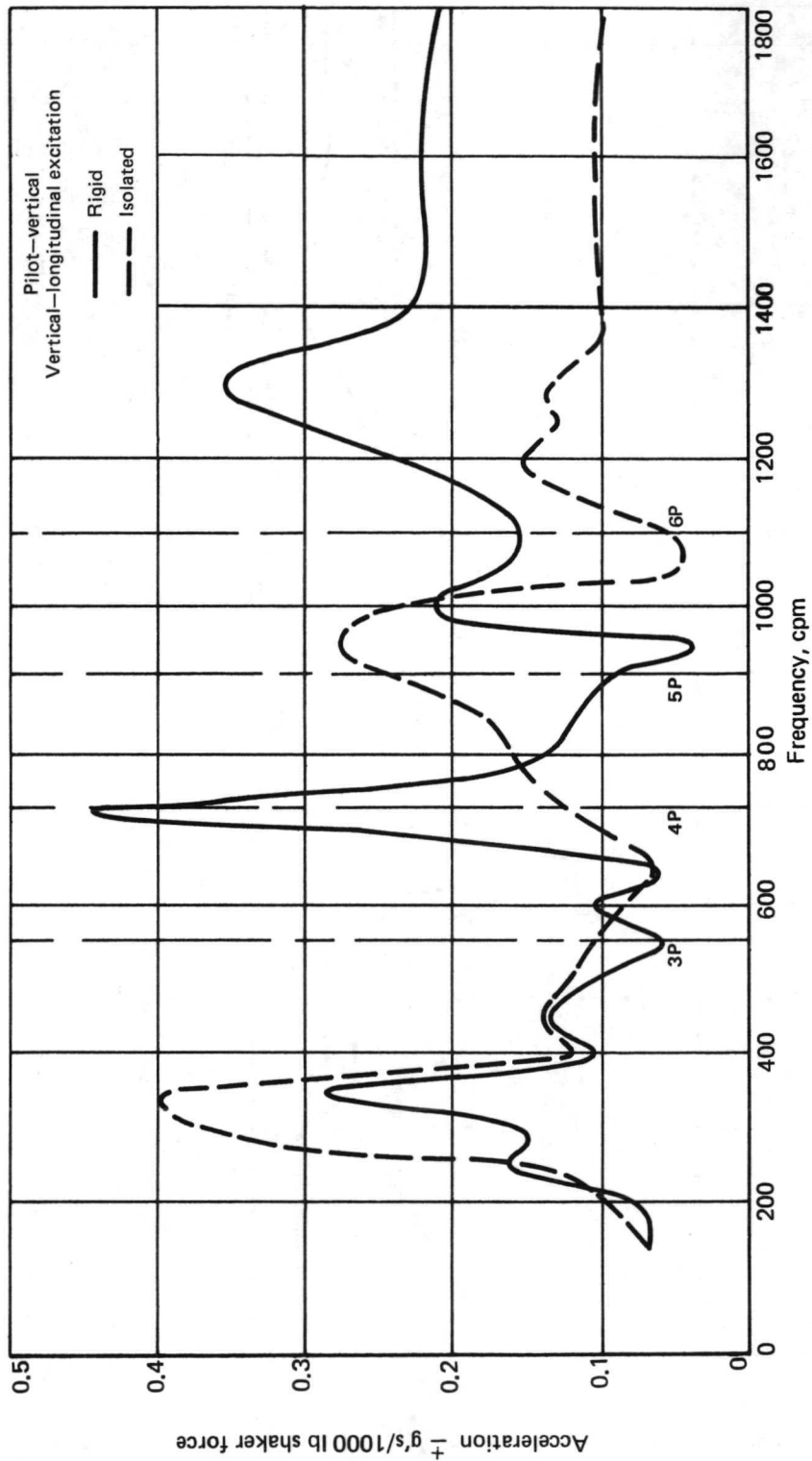


Figure 33. - Pilot Vertical Response due to Vertical-Longitudinal Excitation

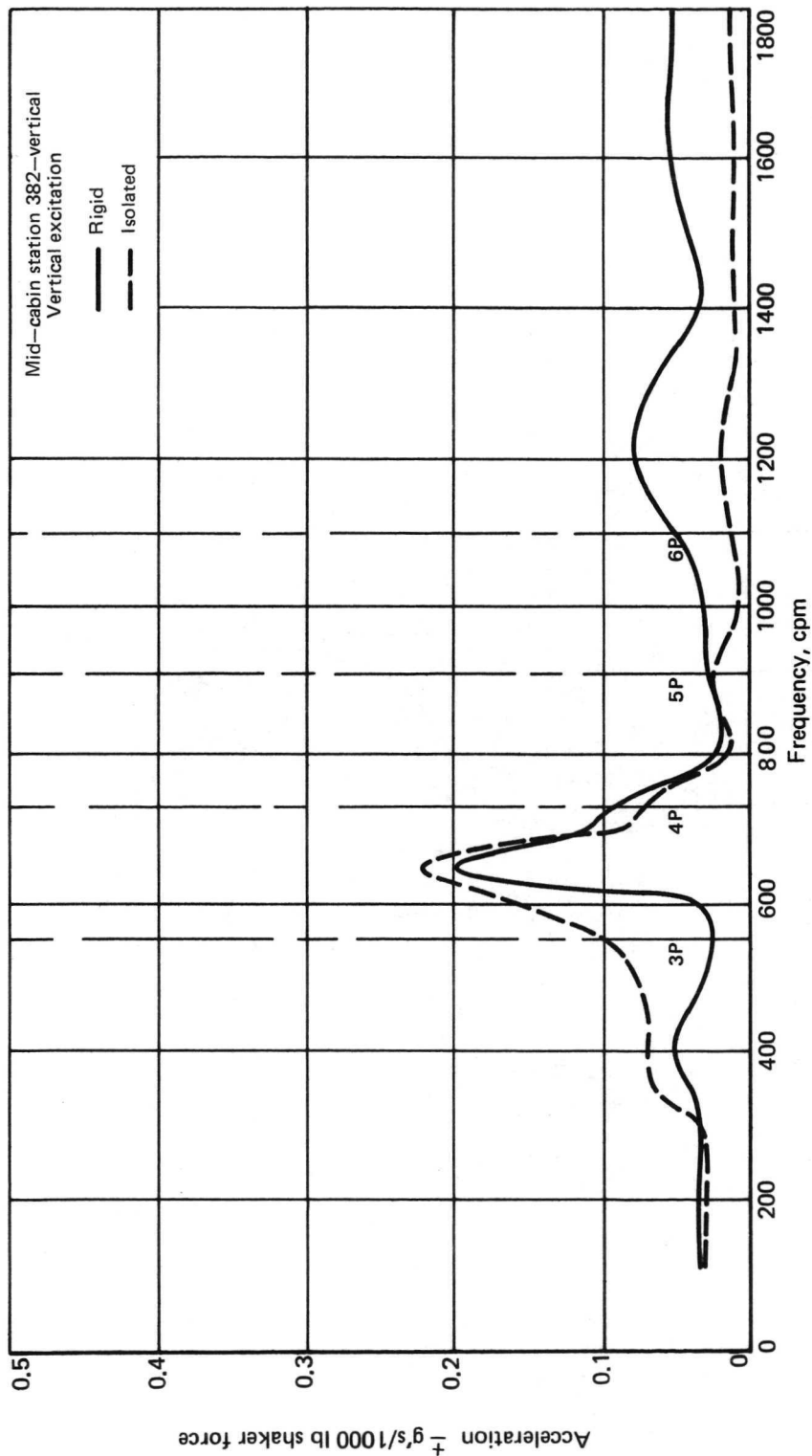


Figure 34. - Mid Cabin Vertical Response Due to Vertical Excitation.



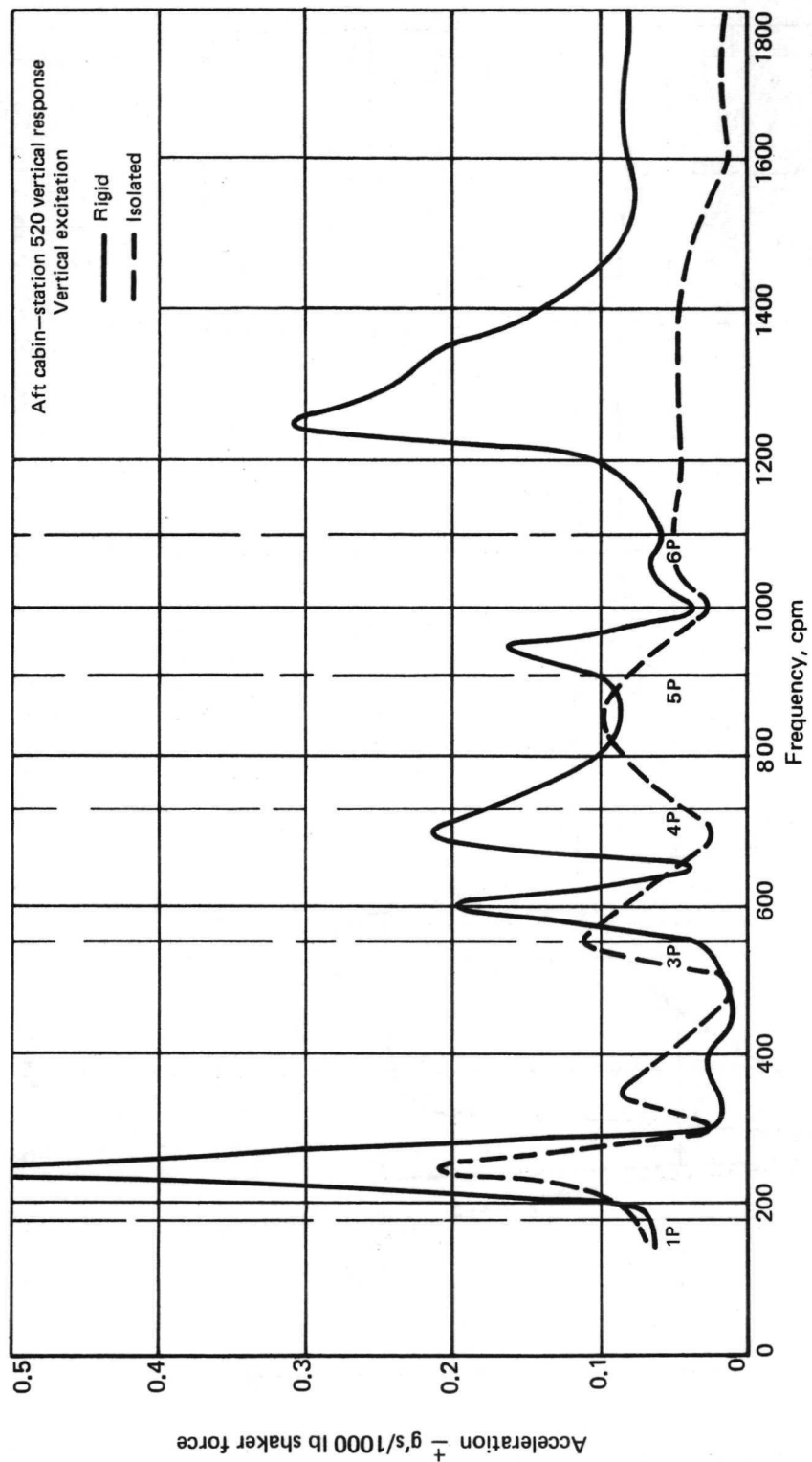


Figure 35. - Aft Cabin Vertical Response Due to Vertical Excitation.

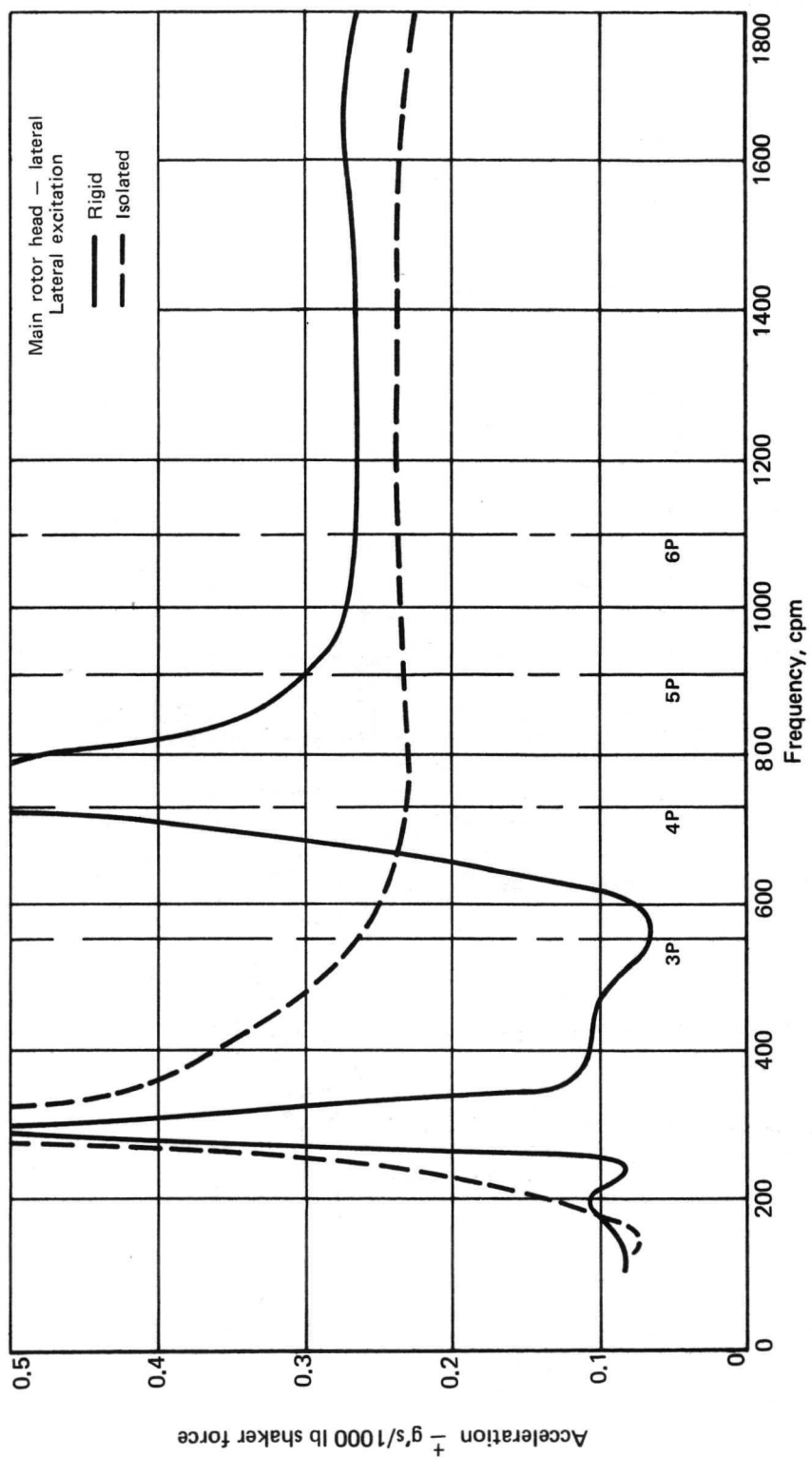


Figure 36. — Main Rotor Head-Lateral Response Due to Lateral Excitation.

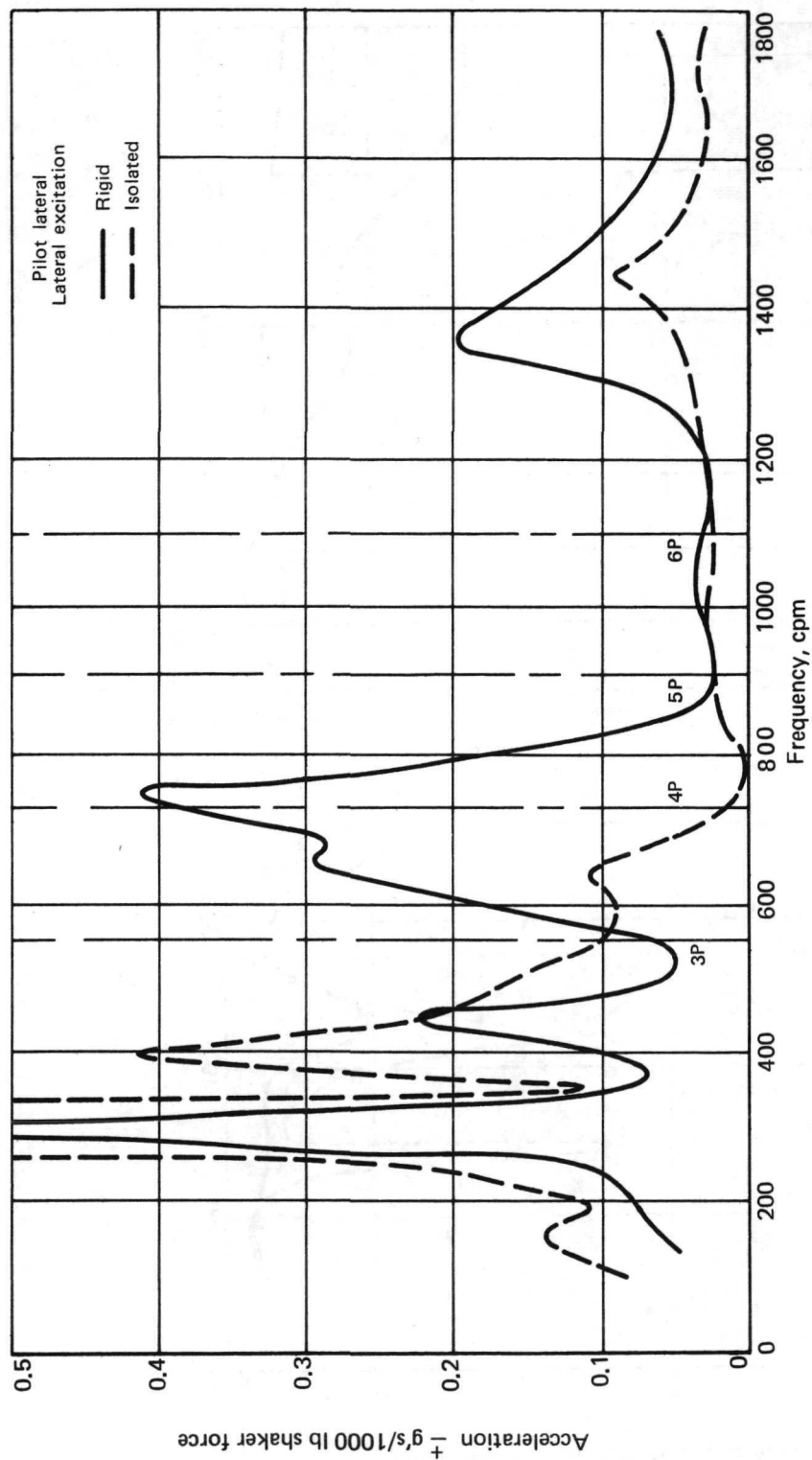


Figure 37. - Pilot Head-Lateral Response Due to Lateral Excitation.

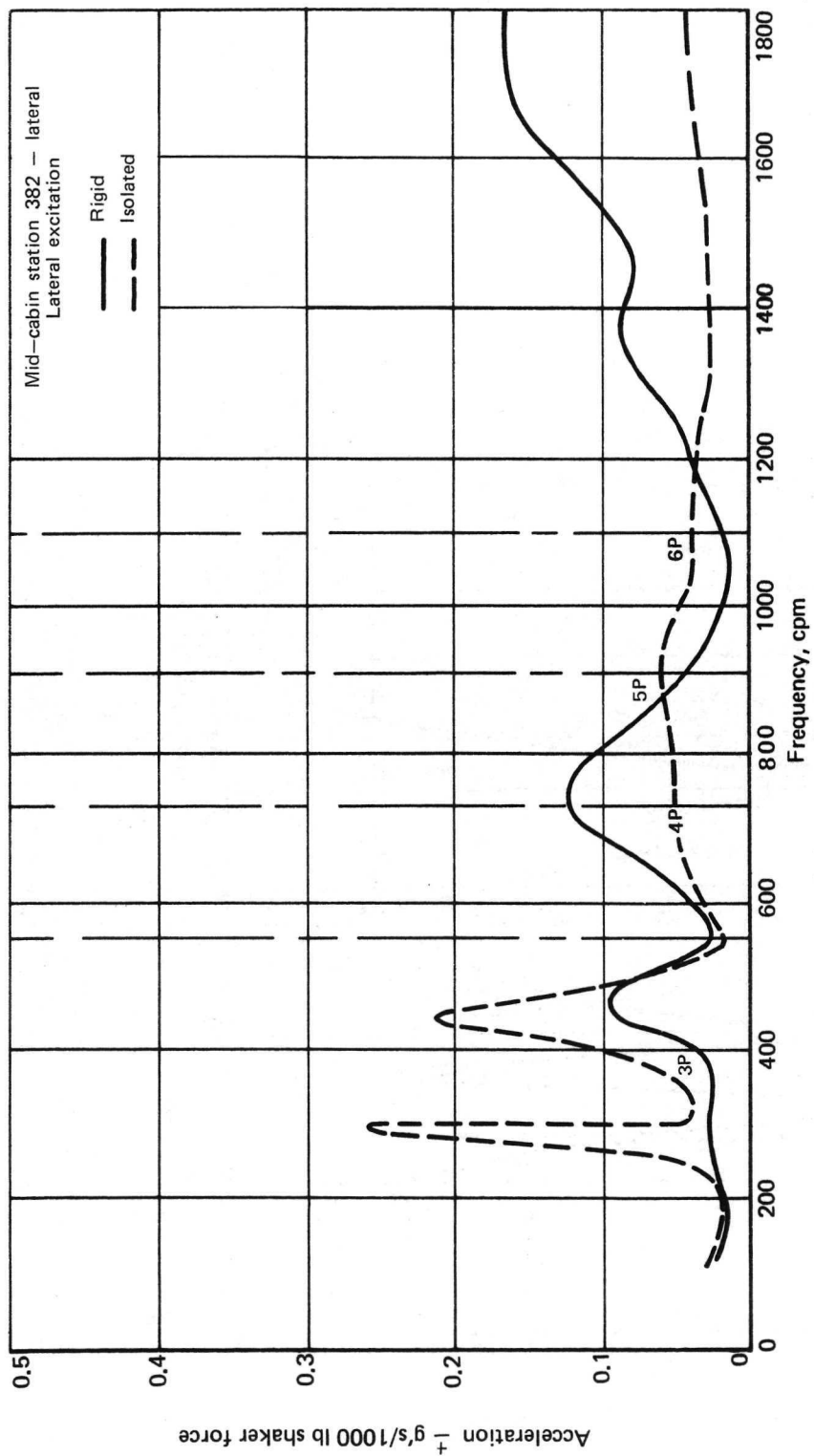


Figure 38. - Mid Cabin Head-Lateral Response Due to Lateral Excitation.

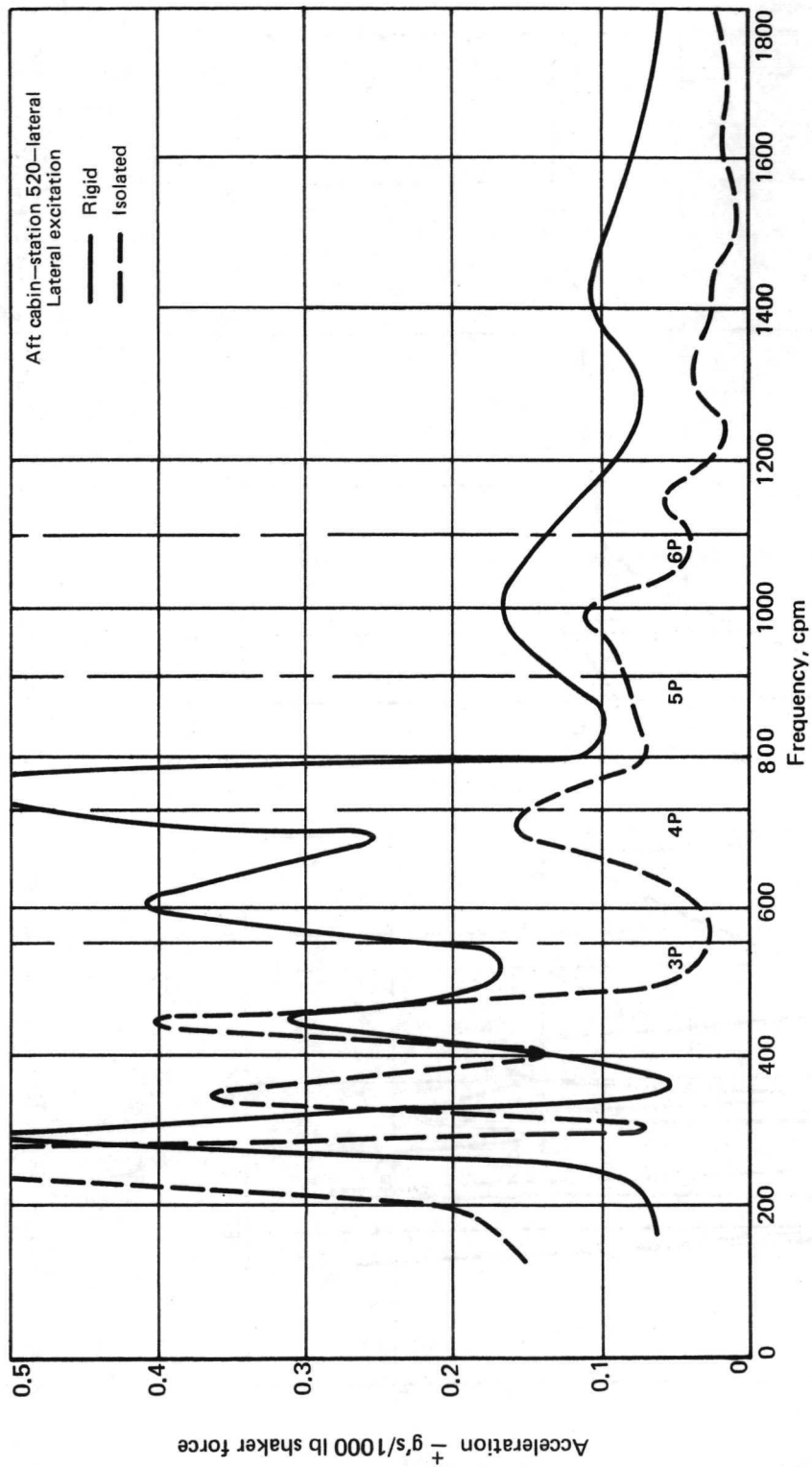


Figure 39. - Aft Cabin Head-Lateral Response due to Lateral Excitation.

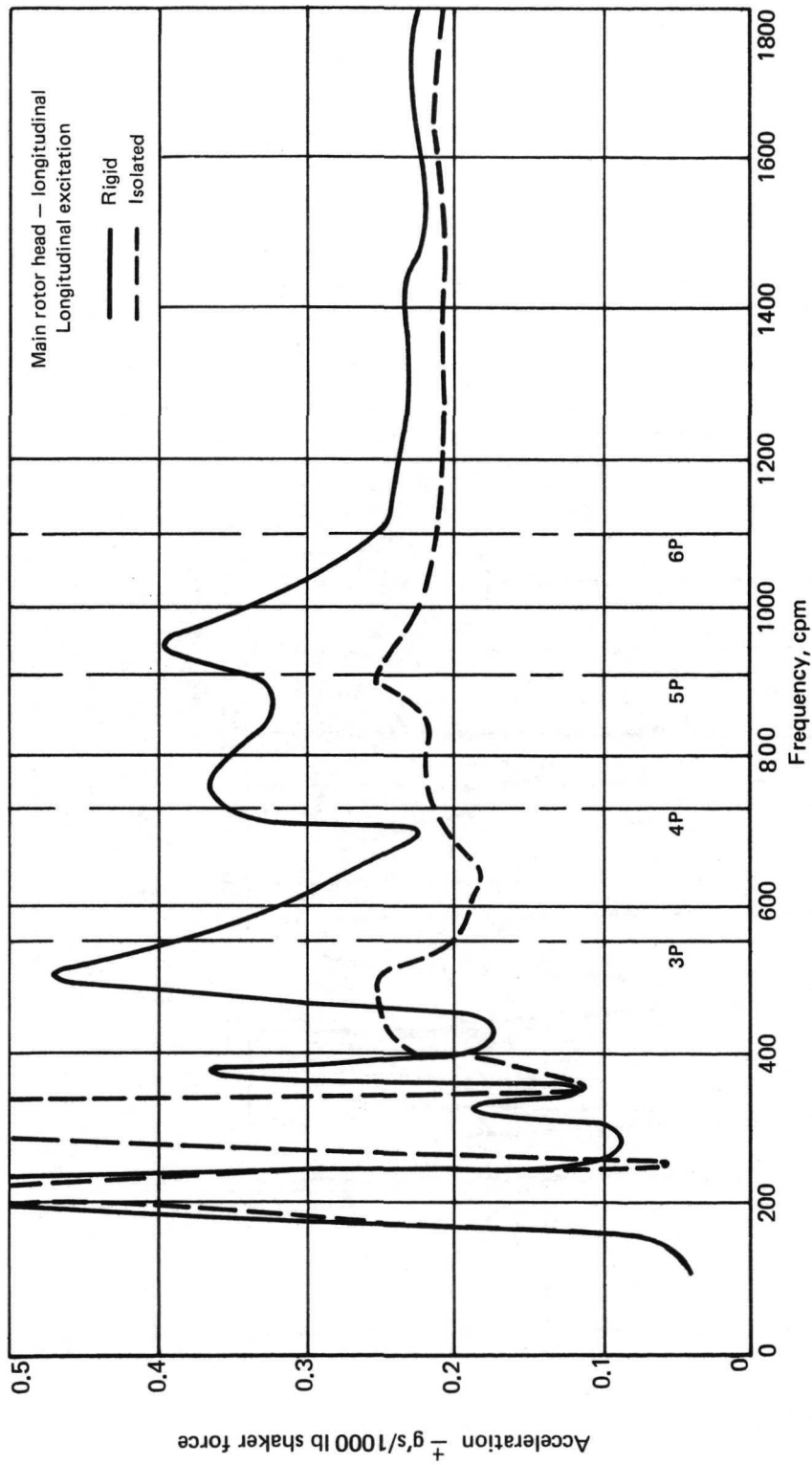


Figure 40. - Main Rotor Head-Longitudinal Response Due to Longitudinal Excitation

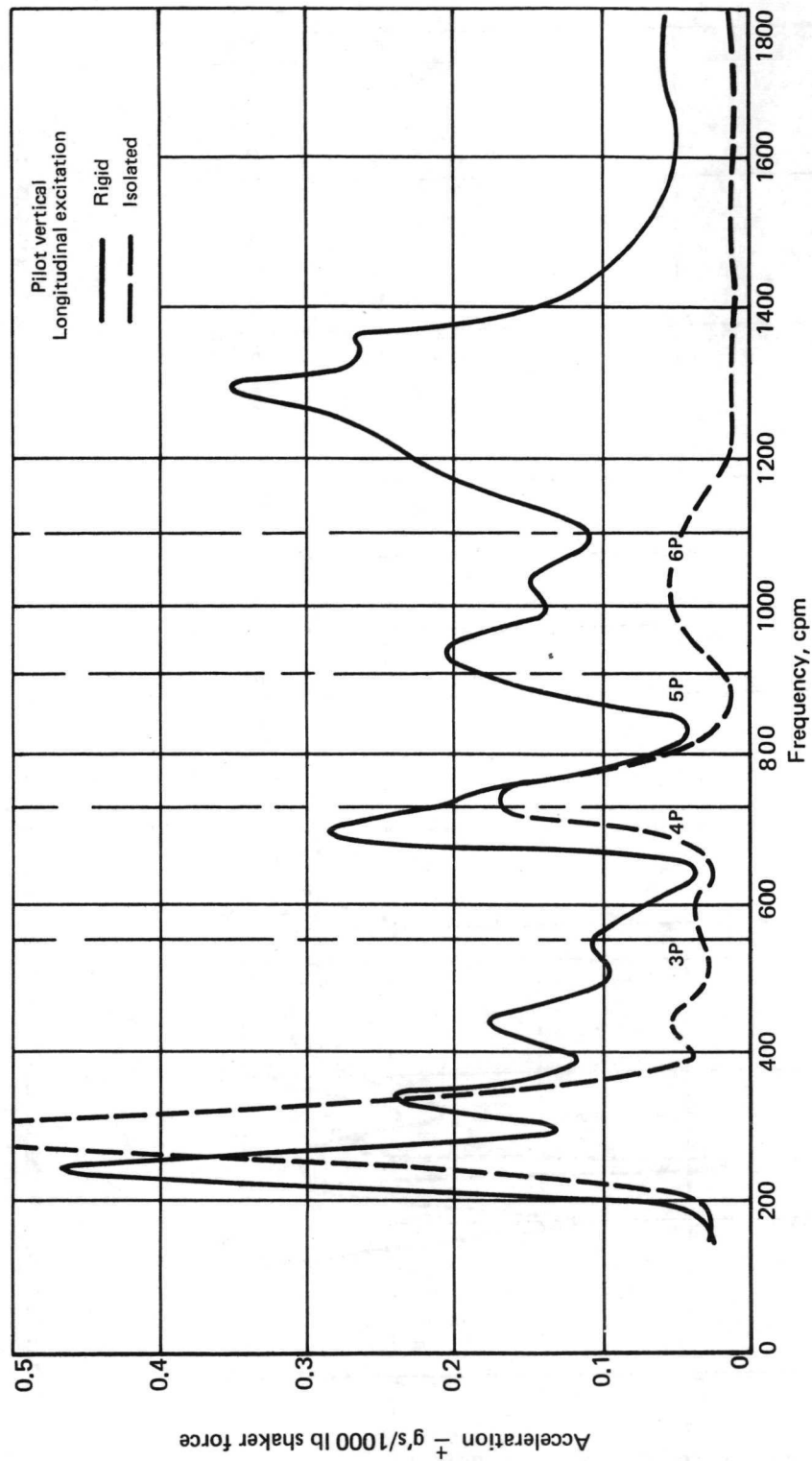


Figure 41. - Pilot Head-Longitudinal Response Due to Longitudinal Excitation.

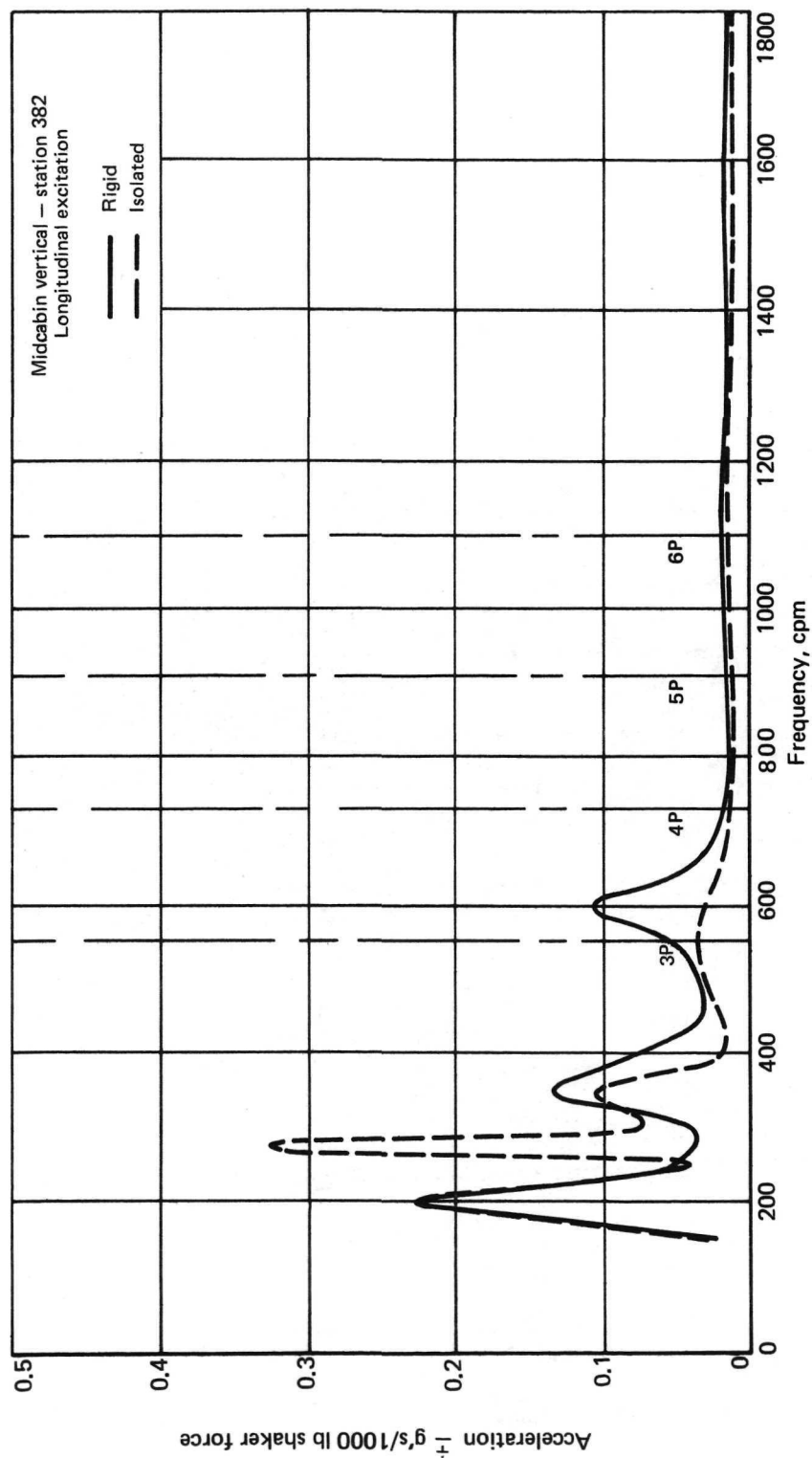


Figure 42. - Mid Cabin Head-Longitudinal Response Due to Longitudinal Excitation.



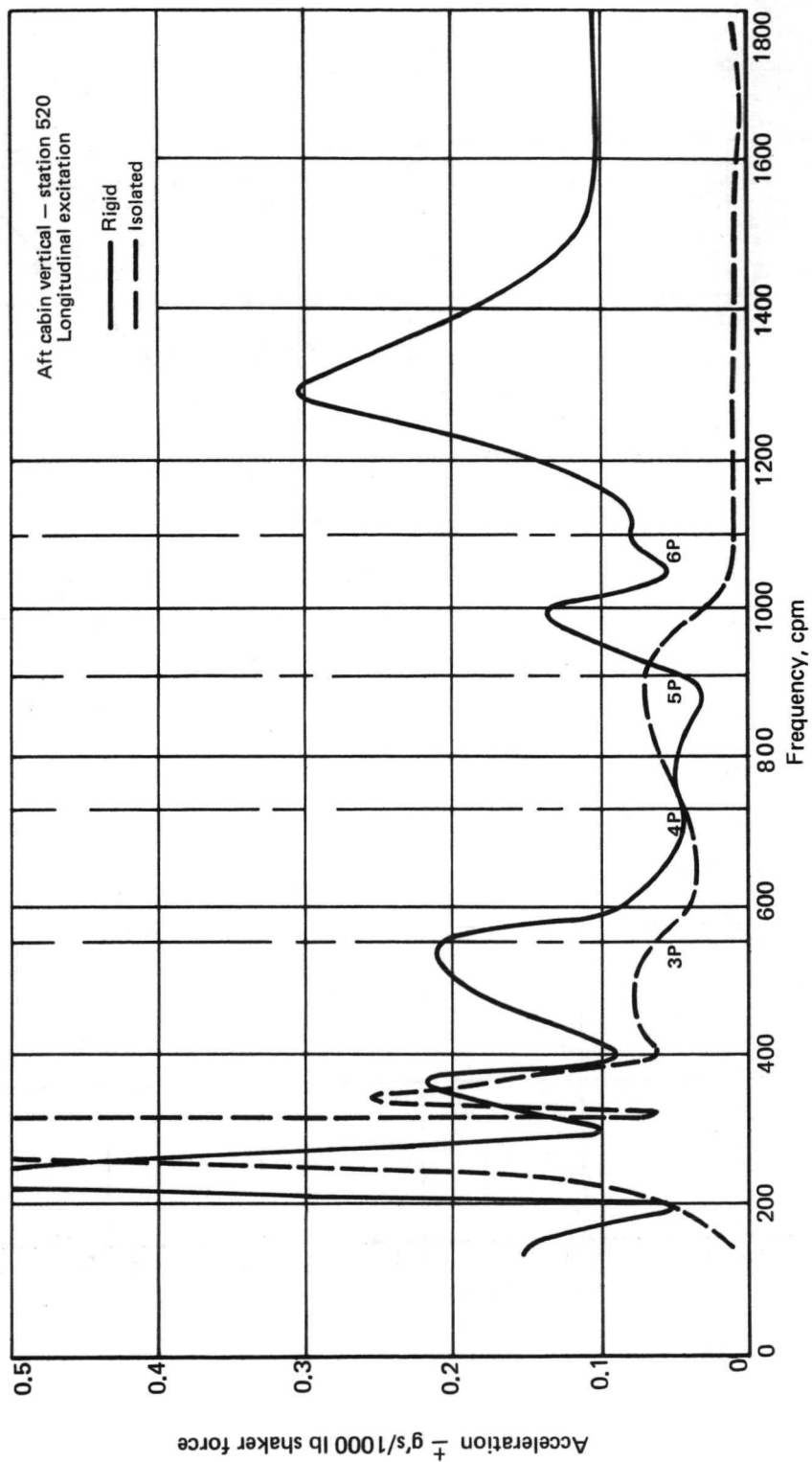


Figure 43. - Aft Cabin Head-Longitudinal Response Due to Longitudinal Excitation.

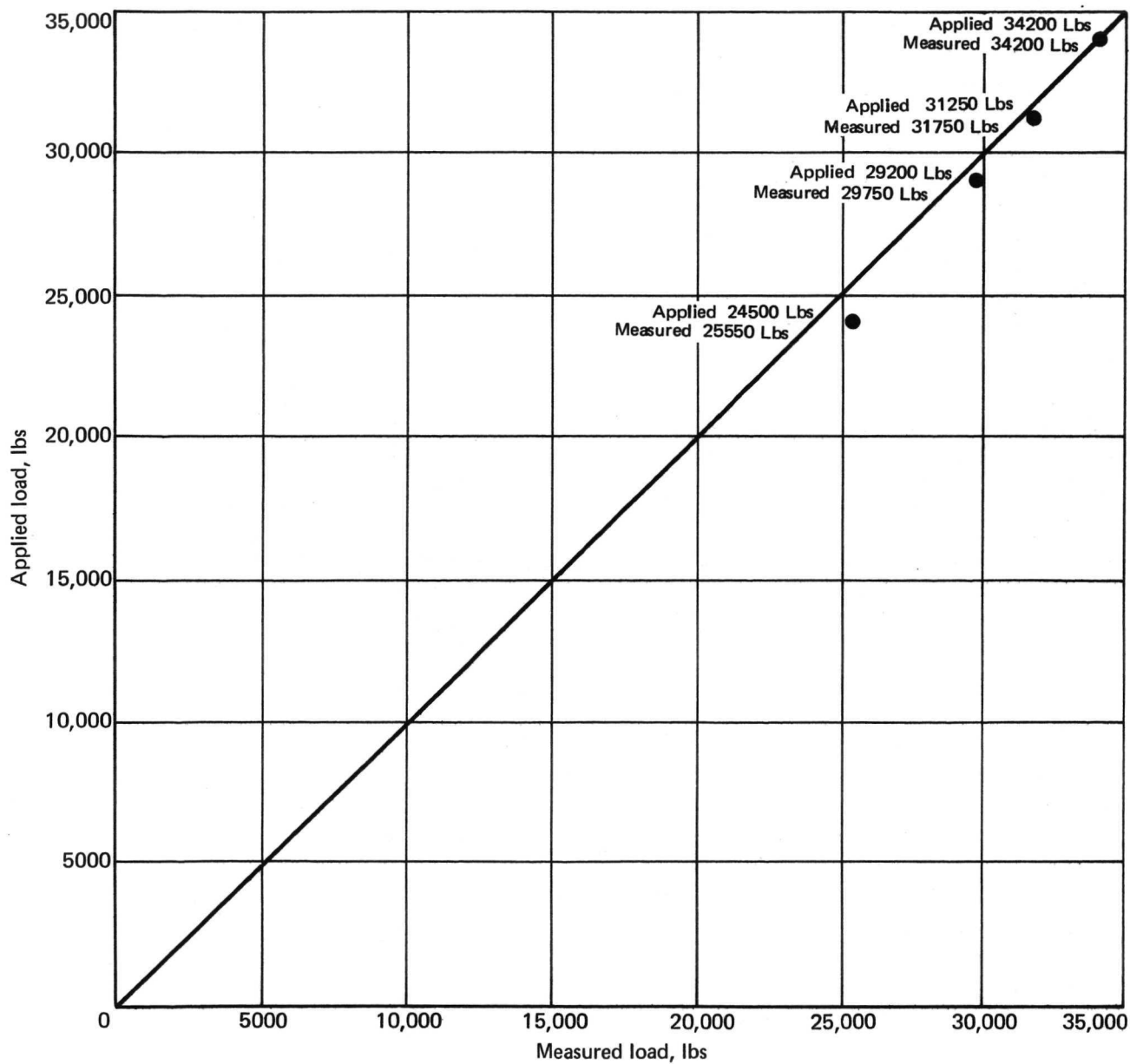


Figure 44. - Rotor Lift Calibration.

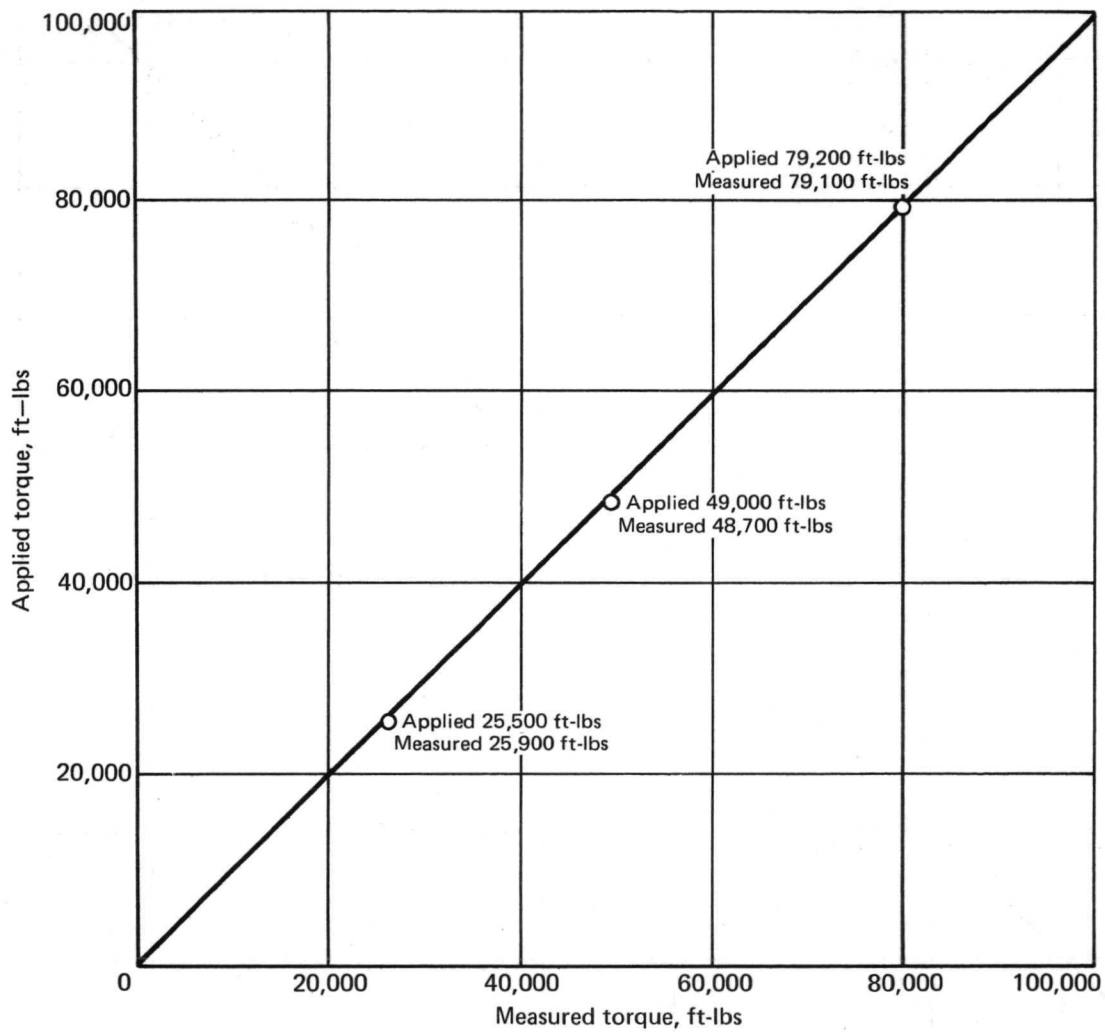


Figure 45. - Main Rotor Torque Calibration.

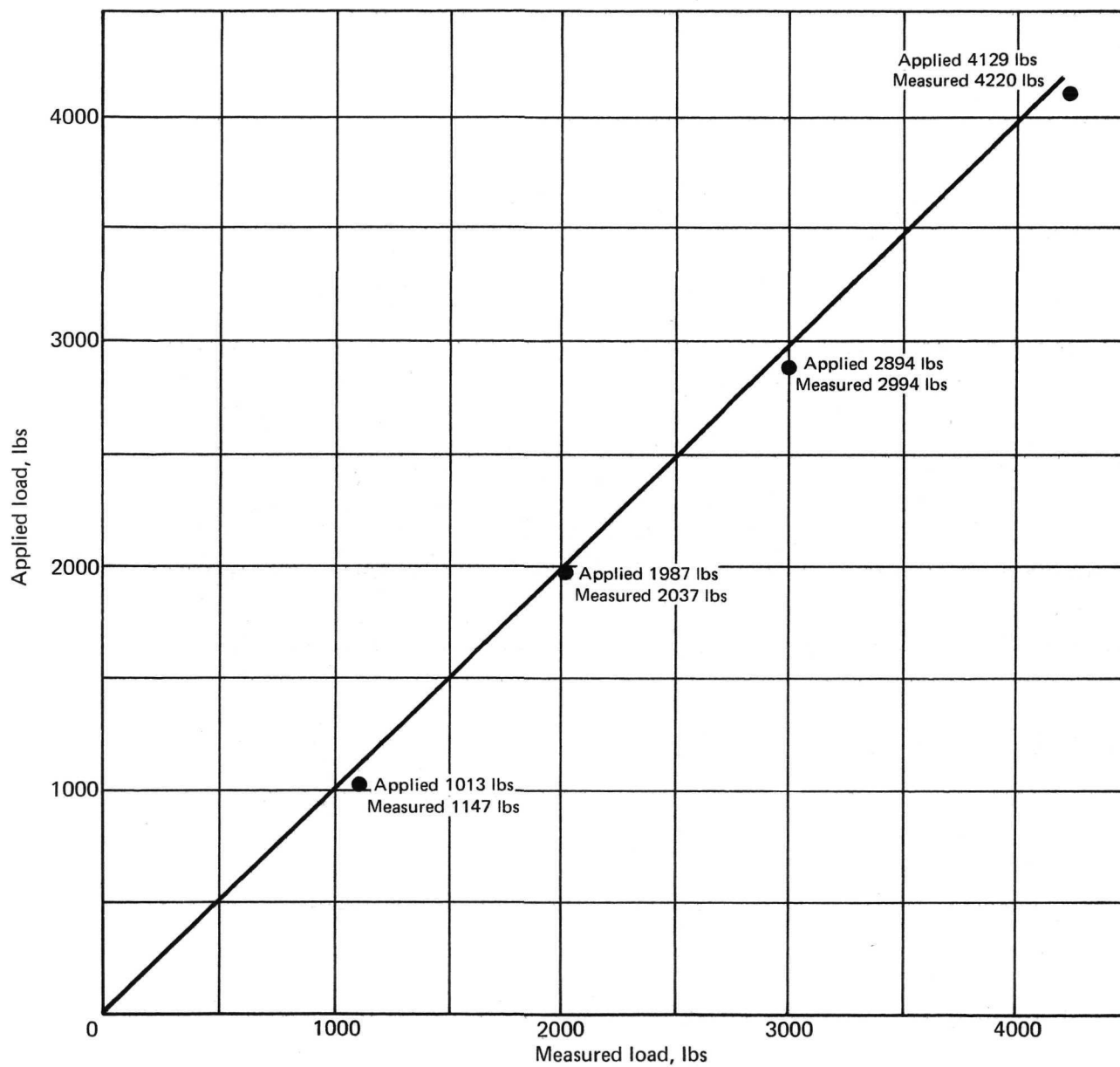


Figure 46. - Propulsive Force Calibration.

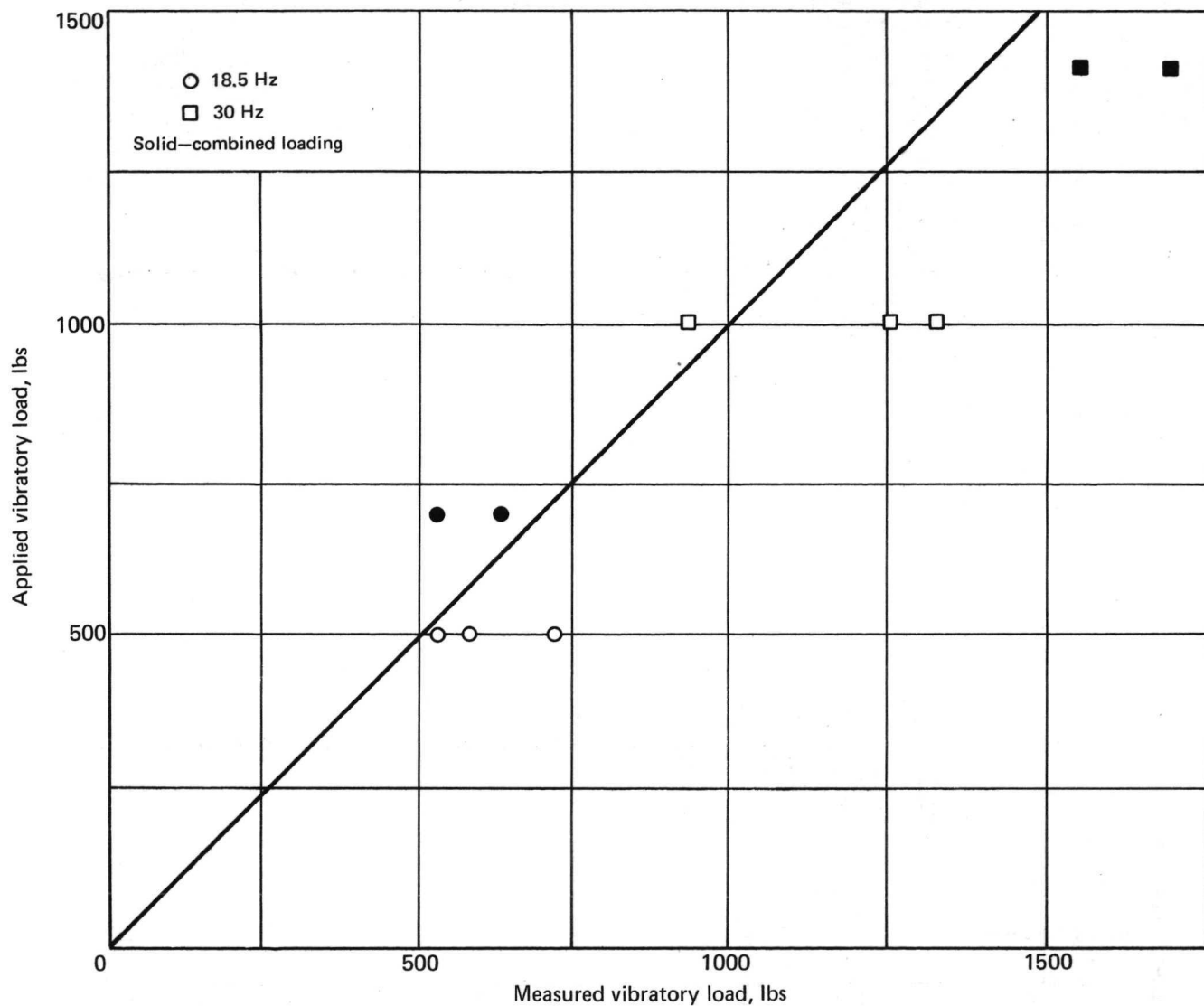


Figure 47. - Total Isolated Vibratory Load Calibration.

ACTIVE ISOLATOR FULL SCALE TEST PROGRAM

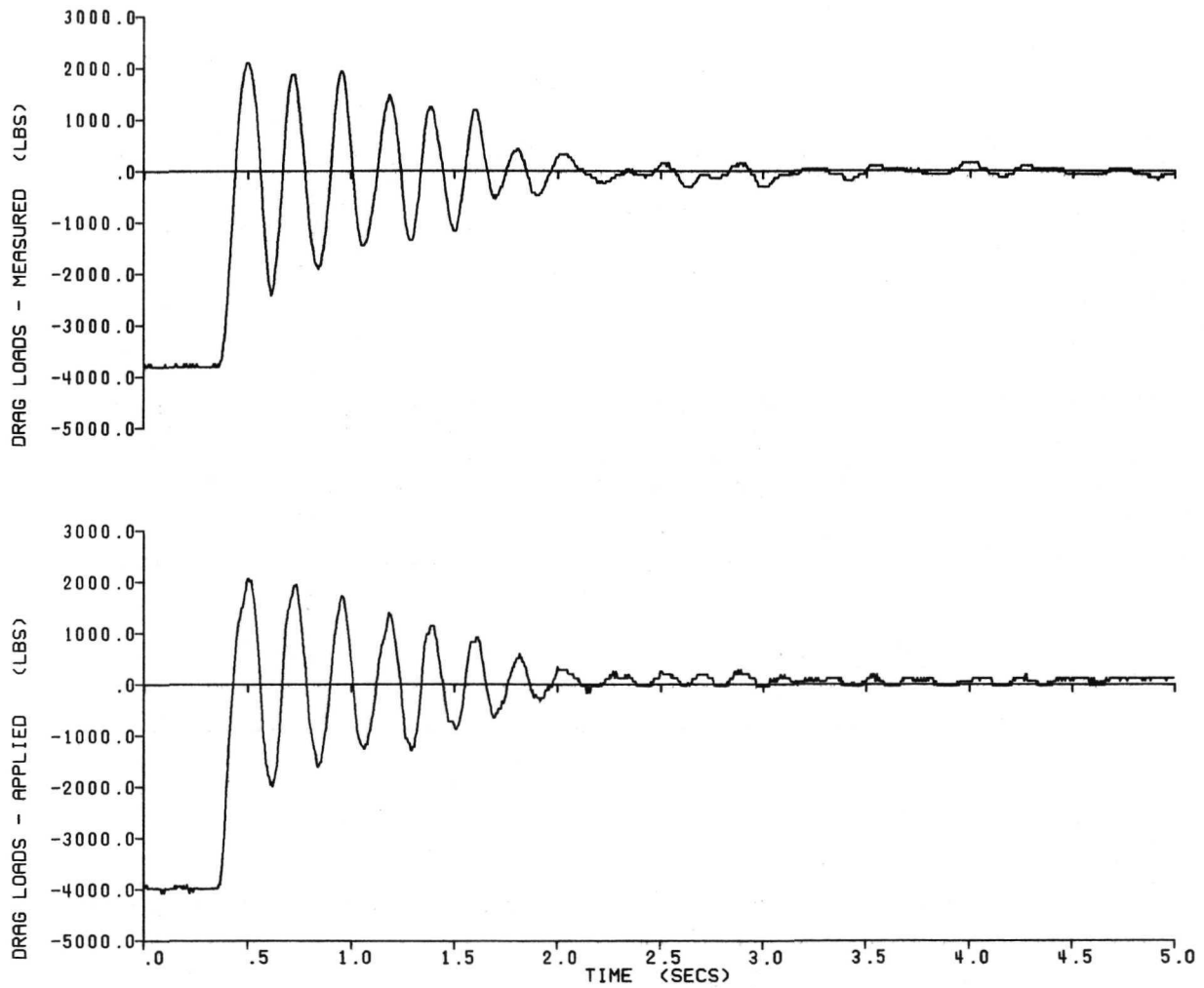


Figure 48. - Longitudinal Transient Response.

ACTIVE ISOLATOR FULL SCALE TEST PROGRAM

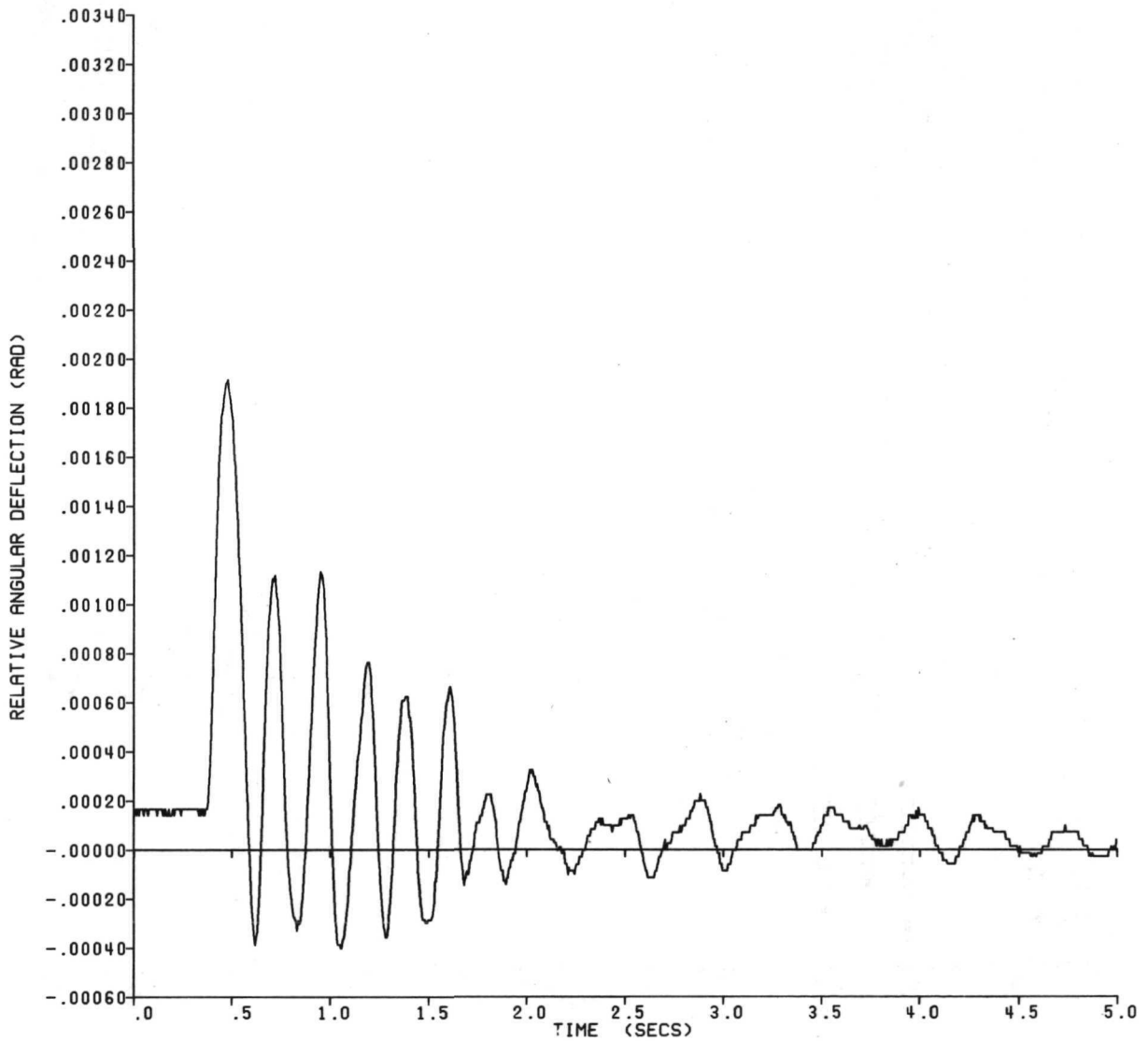


Figure 49. - Angular Deflection Between Airframe and Transmission Interface.

ACTIVE ISOLATOR FULL SCALE TEST PROGRAM

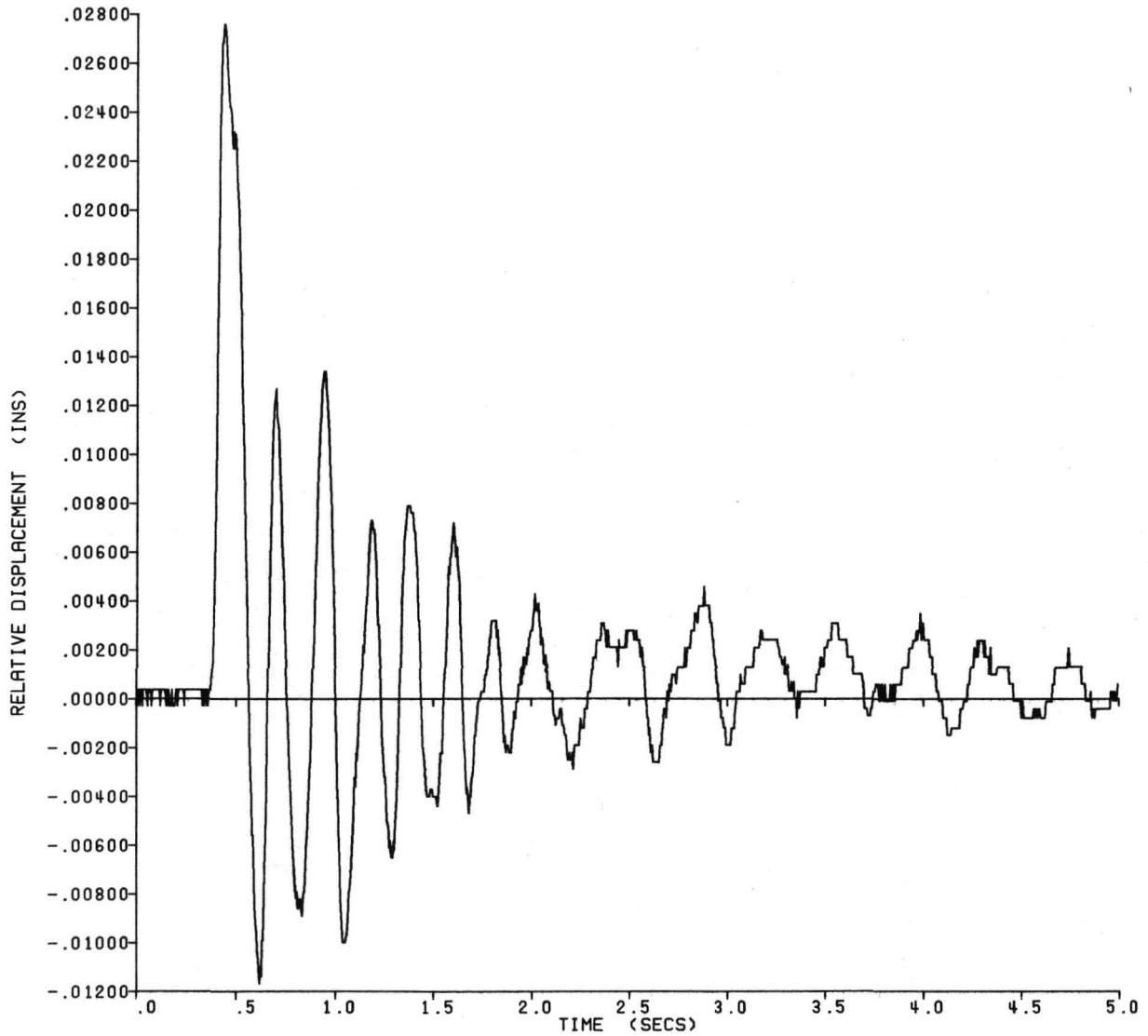


Figure 50. - Relative Deflection Between Airframe and Transmission Interface.



ACTIVE ISOLATOR FULL SCALE TEST PROGRAM

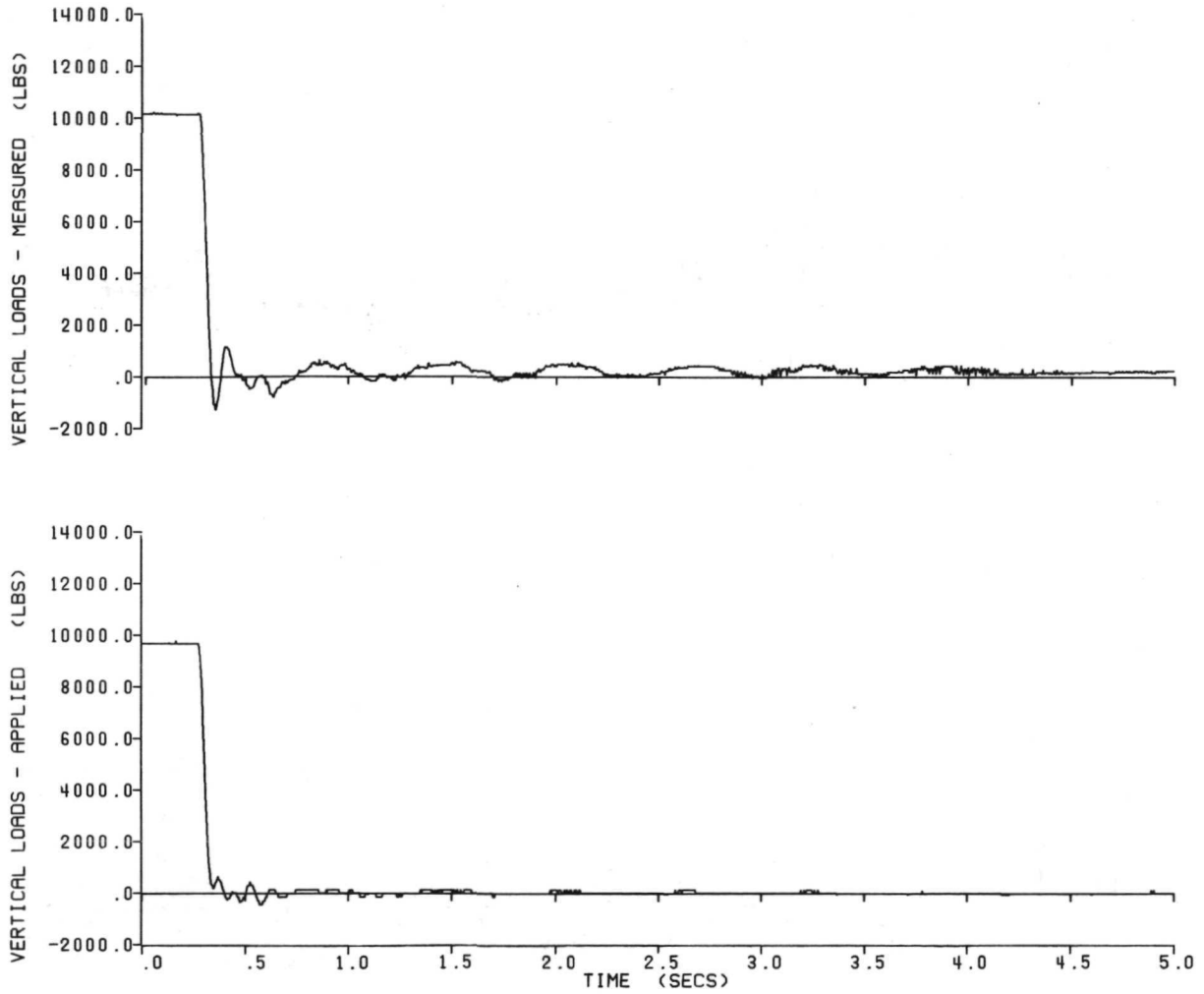


Figure 51. - Vertical Transient Response.

ACTIVE ISOLATOR FULL SCALE TEST PROGRAM

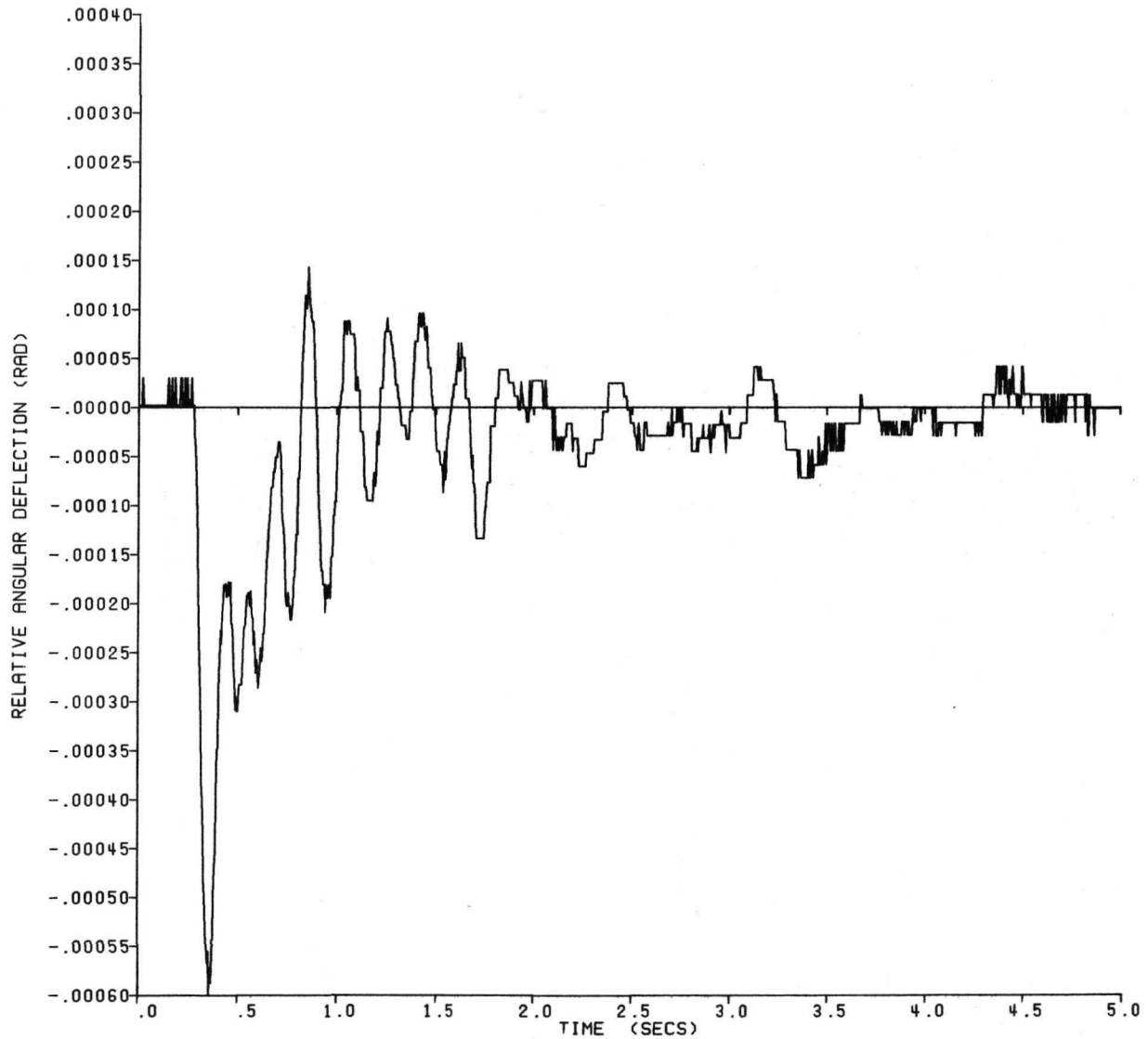


Figure 52. - Angular Deflection Between Airframe and Transmission Interface.

ACTIVE ISOLATOR FULL SCALE TEST PROGRAM

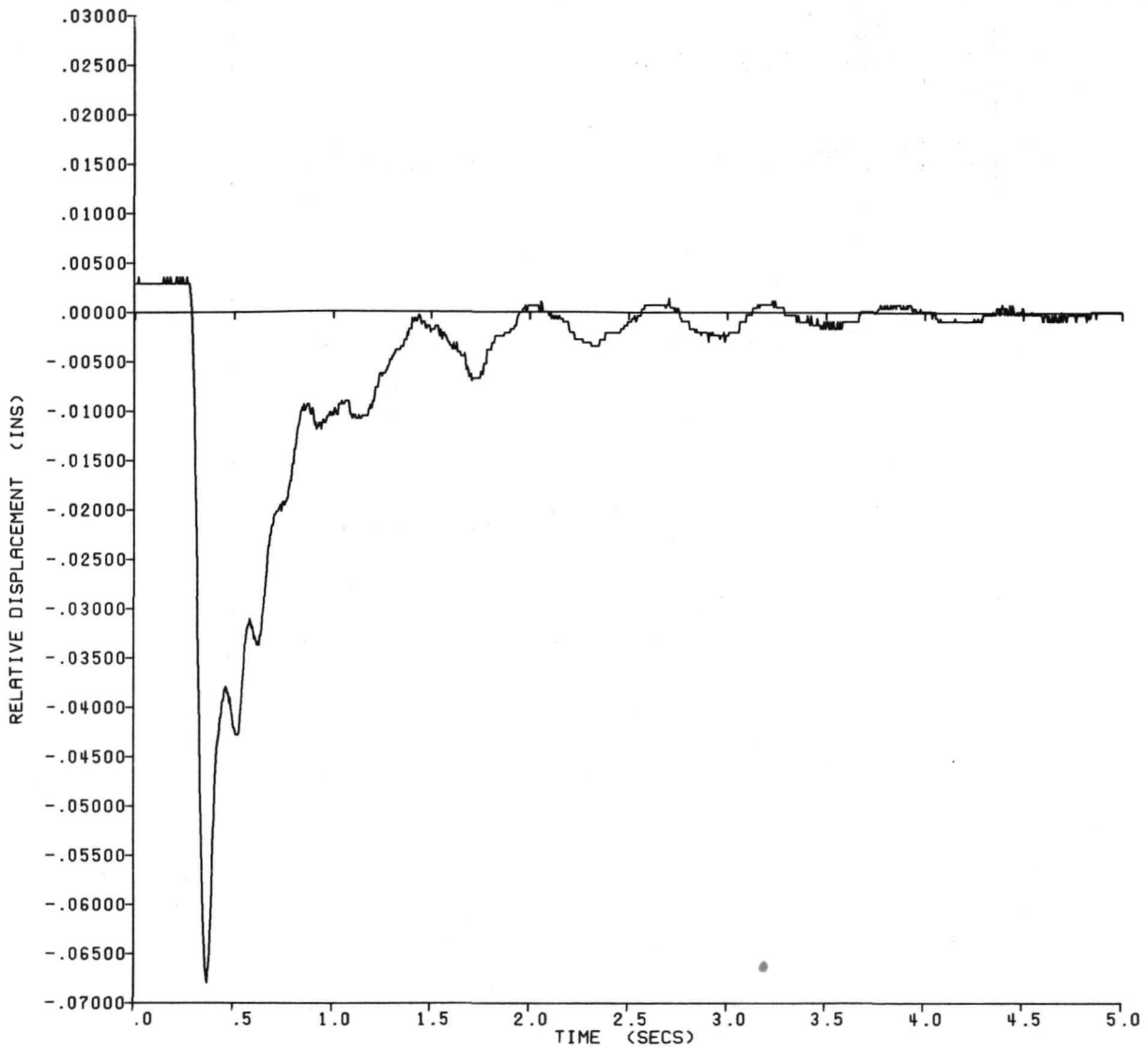


Figure 53. - Relative Deflection Between Airframe and Transmission Interface.

TABLE 1. BENCH TEST ISOLATOR PARAMETERS

<u>ISOLATOR GAIN</u>	
50	
100	(NOMINAL)
200	
<u>ISOLATOR LOW SIDE PRECHARGE PRESSURE (PSI)</u>	
150	
300	(NOMINAL)
500	
<u>ISOLATOR HIGH SIDE PRECHARGE PRESSURE (PSI)</u>	
550	
1100	(NOMINAL)
1820	
<u>FLUID DAMPING ORIFICE DIAMETER (INCHES)</u>	
.201	
.221	(NOMINAL)

TABLE II. - TEST SUMMARY STEADY APPLIED VERSUS  
MEASURED LOADS

Steady lift		Steady Longitudinal		Steady torque	
Applied	Measured	Applied	Measured	Applied	Measured
lbs	lbs	lbs	lbs	ft-lbs	ft-lbs
34200	35070	—	—	—	—
24500	25500	—	—	—	—
29200	29750	—	—	—	—
31250	31750	—	—	—	—
34000	34000	—	—	—	—
34200	34620	4130	4170	—	—
	34800	4180	4220	—	—
	34620	1013	1147	—	—
	33880	1987	2037	—	—
	32990	2894	2994	—	—
	34110	4129	4220	—	—
	34370	—	—	78800	81000
	34580	—	—	80600	83000
	34510	—	—	25500	25900
	33810	—	—	49000	48700
↓	34310	—	—	79200	79100
34200	34360	5435	3260	78300	80100

TABLE III - TEST SUMMARY OF VIBRATORY APPLIED VERSUS MEASURED LOADS

Steady lift		Steady longitudinal		Steady torque		Vibratory frequency	Vibratory loads					
Applied	Measured	Applied	Measured	Applied	Measured		Vertical		Lateral		Longitudinal	
lbs	lbs	lbs	lbs	ft-lbs	ft-lbs	Hz	Applied	Measured	Applied	Measured	Applied	Measured
							-lbs/phase	± lbs/phase	± lbs/phase	± lbs/phase	± lbs/phase	± lbs/phase
34200	33700	—	—	—	—	10	720/0°	2150/-24°	0/0°	140/54°	0/0°	410/-101°
	35120	—	—	—	—	18.5	1000/0°	935/0°	0/0°	60/0°	0/0°	270/0°
	33780	—	—	—	—	30	500/0°	730/0°	0/0°	10/0°	0/0°	20/0°
	34290	—	—	—	—	10	0/0°	280/0°	0/0°	100/0°	720/0°	1120/0°
	34280	—	—	—	—	18.5	0/0°	75/0°	0/0°	80/0°	1000/0°	1270/0°
	34240	—	—	—	—	30	0/0°	40/0°	0/0°	0/0°	500/0°	530/0°
	35080	—	—	—	—	10	0/0°	860/-154°	750/0°	1900/0.3°	0/0°	420/-138°
	34980	—	—	—	—	18.5	0/0°	10/0°	1000/0°	1350/0°	0/0°	40/0°
	34930	—	—	—	—	30	0/0°	0/0°	500/0°	580/0°	0/0°	50/0°
	35900	—	—	—	—	10	510/0°	970/-31°	0/0°	120/19°	510/0°	1300/-55°
	37200	—	—	—	—	18.5	990/0°	1110/0°	0/0°	140/0°	990/0°	1310/0°
	36100	—	—	—	—	30	495/0°	350/0°	0/0°	50/0°	495/0°	510/0°
	34900	—	—	—	—	10	510/0°	910/108°	510/0°	890/1.4°	0/0°	580/98°
	34900	—	—	—	—	18.5	990/0°	560/0°	990/0°	1460/0°	0/0°	190/0°
	34980	—	—	—	—	30	495/0°	200/0°	495/0°	530/0°	0/0°	30/0°
	34100	882	1609	38000	39300	10	360/0°	1420/123°	360/0°	910/42°	0/0°	220/131°
	34120	945	1650	40200	41100	18.5	990/0°	1020/0°	990/0°	1310/0°	0/0°	650/0°
34200	34000	856	1684	40200	41400	30	350/0°	360/0°	350/0°	340/0°	0/0°	60/0°

APPENDIX I

Active Isolator Precharge Calculations

In establishing the system precharge pressures, it was determined that the individual isolator spring rates would be set to the 10,000 pound per inch value utilized during the test of Reference 2.

The equations for both the high and low pressure side spring rates as a function of system operating conditions are developed in Reference 2. It is seen that the spring rate at high frequencies is proportional to the operating pressure, piston cross-sectional area, and inversely proportional to the operating volume between the diaphragm and flow restrictor. This equation is given as:

$$K_{a,b} = \frac{P_{a,b} A_{a,b}^2}{V_{ca',b'}}$$

The piston area has not changed, and thus spring rate is proportional to the ratio of  $P_{a,b}$  to  $V_{ca',b'}$  solely. A sample calculation of the required precharge pressures for isolator #1 is presented. Calculations for the remaining units are identical in nature. From the data for the original configuration (Table I-1), it is seen that the high pressure side spring rate is proportional to the following ratio:

$$K \propto \frac{P_b}{V_{cb'}} = \frac{1100}{16} = 68.8$$

The high side operating pressure for isolator #1 is measured at 1300 psi. Therefore, the new high pressure side operating volume  $V_{cb'}$  required to maintain the same spring rate is:

$$V_{cb'} = \frac{1300}{68.8} = 18.9$$

The total available air volume is given as:

$$V_{tb} + V_{cb_0} = 14.2 + 26.6 = 40.8$$

The total high pressure side air volume under operating conditions for isolator #1 is:

$$V_{tb} + V_{cb'} = 14.2 + 18.9 = 33.1$$

The air pressure under operating conditions is equal to the hydraulic fluid operating pressure  $P_b$ . Therefore, assuming adiabatic expansion, the required precharge pressures can be determined from the following relationship:

$$P_H (V_{tb} + V_{cb_0}) = P_b (V_{tb} + V_{cb'})$$

Therefore,

$$P_H = \frac{(1300)(33.1)}{(40.8)} = 1050 \text{ psi}$$



In a similar manner for the low pressure side of isolator #1:

$$V_{ca}' = \frac{(420)(6)}{300} = 8.4$$

$$V_{ta} + V_{ca_o} = 7.1 + 26.6 = 33.7$$

$$V_{ta} + V_{ca}' = 7.1 + 8.4 = 15.5$$

Therefore,

$$P_L = \frac{(420)(15.5)}{(33.7)} = 193 \text{ psi}$$

A summary of all measured and calculated parameters for the three units and the basic configuration is presented in Table I-1.

TABLE I-1 ACTIVE ISOLATOR MEASURED AND CALCULATED OPERATING PARAMETERS

	<u>Original Configuration</u>	<u>Isolator 1</u>	<u>Isolator 2</u>	<u>Isolator 3</u>
$V_{ca_o}$	26.6	26.6	26.6	26.6
$V_{cb_o}$	26.6	26.6	26.6	26.6
$V_{ta}$	7.1	7.1	7.1	7.1
$V_{tb}$	14.2	14.2	14.2	14.2
$P_a$	300	420	400	250
$P_b$	1100	1300	1150	900
$V_{ca}'$	6	8.4	8	5
$V_{cb}'$	16	18.9	16.7	13.1
$P_H$	810	1050	870	600
$P_L$	116	193	179	90



APPENDIX II

APPLICATIONS OF ACTIVE ISOLATION TO SLOWED ROTORS

Considering a reduction of 50 percent in main rotor operating speed, the resulting N/Rev frequency would be reduced by a factor of 2 to 555 CPM. At this frequency, the degree of vertical isolation is not considered acceptable (Figures 32 to 35). Shifting of isolator modes to maintain the desired N/Rev isolation can be obtained by lowering the existing spring rate in each isolator. By lowering the spring rate and maintaining a constant ratio of N/Rev frequency to isolator mode frequencies, the amount of isolation at 555 CPM must be comparable to that obtained with the configuration tested at a forcing frequency of 1110 CPM. The location of inplane and vertical isolator modes, with the lower spring rate and reduced rotor speed are compared with the original spectrum obtained at normal operating conditions in Figure (II-1). No amplification at 1/Rev would result since this frequency is also reduced by 50 percent.

In order to achieve the modes at the indicated frequencies (Figure II-1), the isolator spring rates would have to be reduced from the 10,000 pounds per inch tested, to 2500 pounds per inch. Assuming system operating pressures to remain constant the required air volumes and precharge pressures required to produce this spring rate are calculated as follows:

$$\text{From the data of Appendix I - } K \propto \frac{68.8}{4} = 17.2$$

resulting in a new high pressure side operating volume:

$$V_{cb}' = \frac{1300}{17.2} = 75.6$$

The total available air volume remains at  $40.8 \text{ in}^3$ , however, the new required air volume under operating conditions is:

$$V_{tb} + V_{cb}' = 14.2 + 75.6 = 89.8$$

Since the required air volume exceeds the existing volume, by  $49 \text{ in}^3$ , an additional supply is required.

Assuming additional air volume capabilities, including those existing, totaled  $100 \text{ in}^3$ , the high air precharge pressure could be calculated from the relationship derived in Appendix I or:

$$P_H = \frac{(1300)(89.8)}{100} = 1168 \text{ psi}$$

In a similar manner for the low pressure side:

$$K \propto \frac{50}{4} = 12.5 \quad \text{and}$$

$$V_{ca}' = \frac{420}{12.5} = 33.6$$

$$V_{ta} + V_{ca'} = 7.1 + 33.6 = 40.7$$

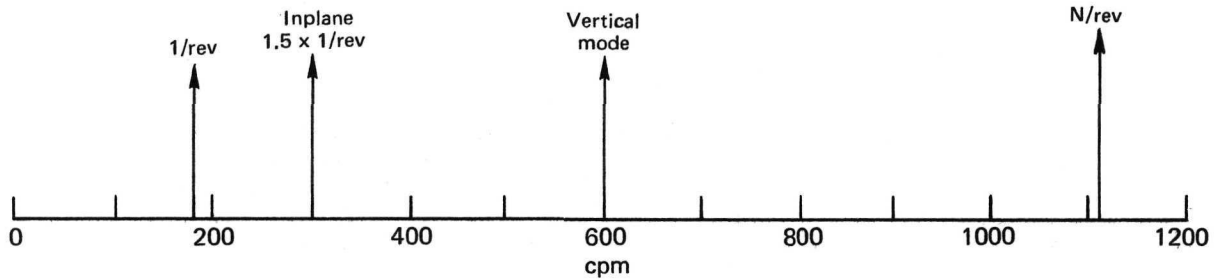
Since the total existing available air volume is 33.7, the capability for expansion is considered. Therefore, an air volume capability totaling 50 in<sup>3</sup> is used. The low air precharge pressure is found to be:

$$P_L = \frac{(420)(40.7)}{50} = 336 \text{ psi}$$

This analysis indicates that available air volumes in the prototype units is inadequate by approximately 50 cubic inches for the high pressure side and 20 cubic inches on the low pressure side.

In order to expand the existing wide band isolation capabilities, air volumes which can be utilized as required would be needed in a universal system. A conceptual schematic of such a system is presented in Figure II-2. The load measurement capability has been determined to be independant of isolator stiffness.

CH-53A  
100%  $N_R$   
K/Isolator = 10,000 lb/inches



CH-53A  
50%  $N_R$   
K/Isolator = 2500 lb/inches

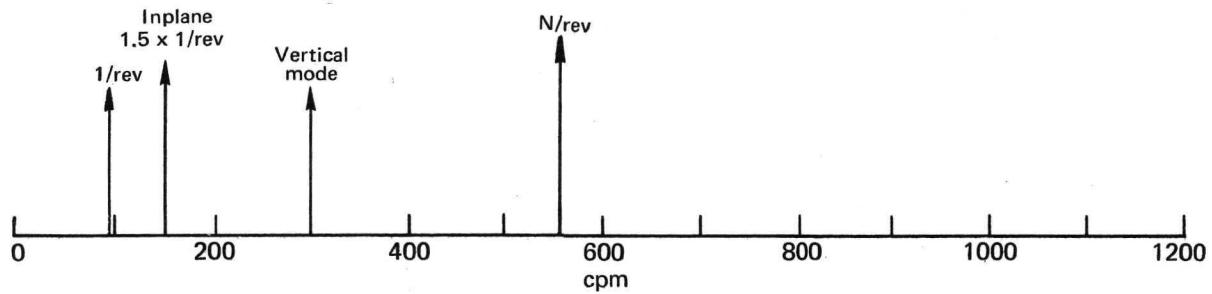


Figure II-1 - Effect of Rotor RPM on Required Isolator Spring Rates.

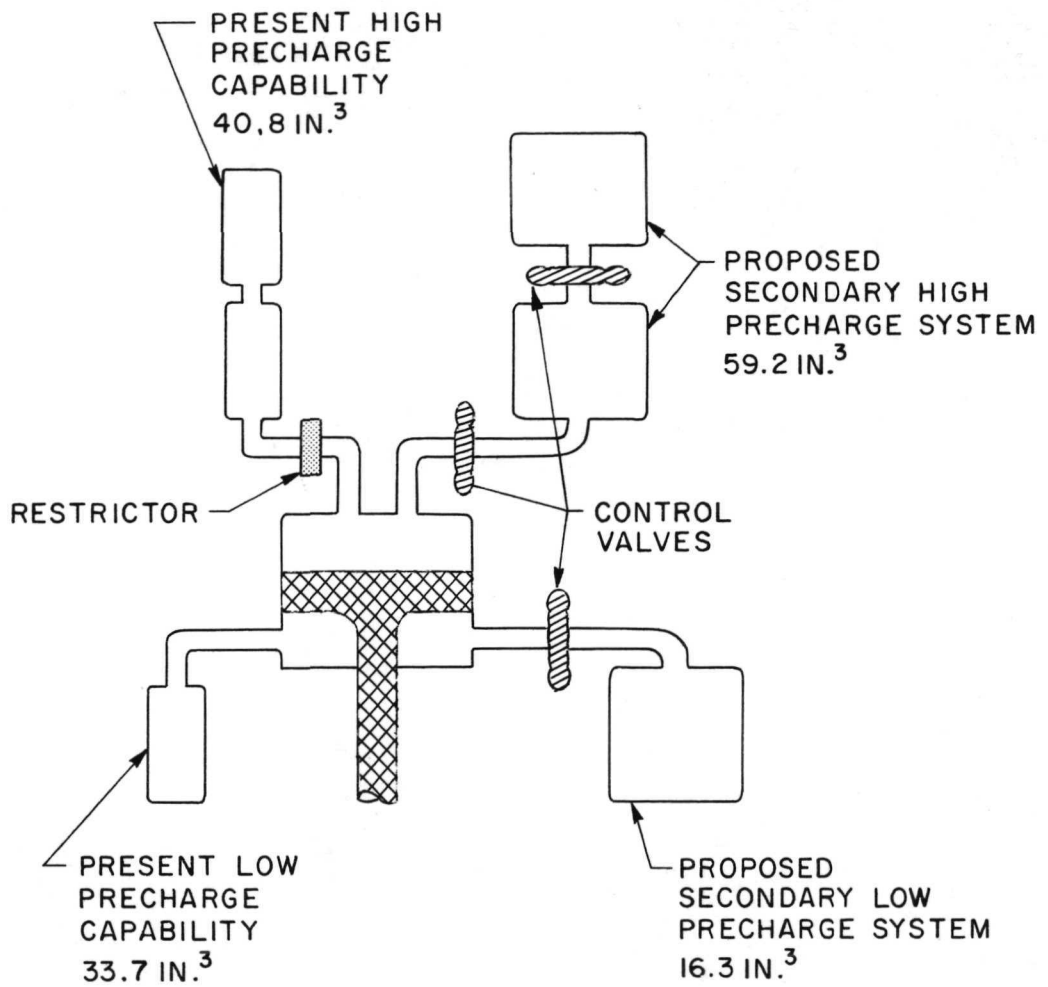


Figure II-2 - Proposed Universal Isolation System.





APPENDIX III

Computer Program Definition

### APPENDIX III-A

Sikorsky Aircraft has developed computer programs to analyze data produced by isolation and rotor loads testing. This data may be of two kinds:

- (1) That produced in a steady or vibratory test. The data produced from actual test cases is keypunched and input to program Y164A along with geometry and mass property information. From this data, the applied and measured forces acting on the system are calculated. Samples of actual test cases and a complete program listing are presented. A listing of this program is presented in Appendix III-B.
- (2) The data resulting from transient testing. The actual data is recorded on analog tape, digitized, and used as a series of time points as input to program Y164E, to calculate the forces in time history form.

The various program in the analysis package are described as follows:

#### Y164A (1108)

For steady or vibratory conditions: Takes recorded keypunched data, plus geometry and mass data in LOADER format, and isolator calibration curves in CLOAD format, and performs calculations to obtain the forces acting on the system.

This program handles the steady and vibratory case on its own. For the transient case, a sequence of programs must be run in addition to Y164A. The sequence is as follows:

#### Y164B (360)

Initializes disk area and stores basic problem description and dimensions.

#### Y164C (360)

Takes digital tape content in 9-bit words, validates, removes 'sets' inserted for reduction purposes and stores the result on disk, where one direct access record has data for all gages over 72 points.

#### Y164D (360)

Determines calibration of disk data, further validates and summarizes data ready for third-pass program, Y164E.

#### Y164E (360)

Third-pass program. This takes data from disk using subroutine GETPTS, together with LOADER and CLOAD data, and generates time histories of the forces on the system. These are written on tape.

Y164F (360)

This program takes the data recorded on tape, and uses time history plotting techniques to produce Benson-Lehner plots.



APPENDIX III-B

COMPUTER PROGRAM LISTING

```

1      2      3      4      5      6      7      8
.....0.....0.....0.....0.....0.....0.....0.....0
U1005 CD COUNT

```

U1005 CD COUNT

1	2	3	4	5	6	7	8	U1005 CD COUNT
.....0	.....0	.....0	.....0	.....0	.....0	.....0	.....0	
EQUIVALENCE	(V( 1),TYPE )	(V( 2),RIGID )					IRBS0500	00051
EQUIVALENCE	(V( 7),DEBUG )						IRBS0510	00052
							IRBS0520	00053
EQUIVALENCE	(V( 11),WUB1 )	(V( 12),XUB1 )	(V( 13),YUB1 )				IRBS0530	00054
1 (V( 14),ZUB1 )	(V( 15),IOXUB1)	(V( 16),IOYUB1)	(V( 17),WUB2 )				IRBS0540	00055
2 (V( 18),XUB2 )	(V( 19),YUB2 )	(V( 20),ZUB2 )	(V( 21),IOXUB2)				IRBS0550	00056
3 (V( 22),IOYUB2)	(V( 23),WLB )	(V( 24),XLB )	(V( 25),YLB )				IRBS0560	00057
4 (V( 26),ZLB )	(V( 27),IOXLB )	(V( 28),IOYLB )	(V( 29),XI1 )				IRBS0570	00058
5 (V( 30),YI1 )	(V( 31),XI2 )	(V( 32),YI2 )	(V( 33),XI3 )				IRBS0580	00059
6 (V( 34),YI3 )	(V( 35),XLC1 )	(V( 36),YLC1 )	(V( 37),ZLC1 )				IRBS0590	00060
7 (V( 38),XLC2 )	(V( 39),YLC2 )	(V( 40),ZLC2 )	(V( 41),XLC3 )				IRBS0600	00061
8 (V( 42),YLC3 )	(V( 43),ZLC3 )	(V( 44),XLC4 )	(V( 45),YLC4 )				IRBS0610	00062
9 (V( 46),ZLC4 )	(V( 47),XTAP1 )	(V( 48),YTAP1 )	(V( 49),ZTAP1 )				IRBS0620	00063
A (V( 50),ALPHA1)	(V( 51),XTAP2 )	(V( 52),YTAP2 )	(V( 53),ZTAP2 )				IRBS0630	00064
B (V( 54),ALPHA2)	(V( 55),XP )	(V( 56),YP )	(V( 57),ZP )				IRBS0640	00065
C (V( 58),BETA )	(V( 59),XA )	(V( 60),YA )	(V( 61),ZA )				IRBS0650	00066
D (V( 62),AH )	(V( 63),AL )	(V( 64),PA )	(V( 65),PR )				IRBS0660	00067
E (V( 66),G )	(V( 67),CV )						IRBS0670	00068
EQUIVALENCE	(V( 70),XA1 )	(V( 71),YA1 )	(V( 72),ZA1 )				IRBS0680	00069
1	(V( 73),XA2 )	(V( 74),YA2 )	(V( 75),ZA2 )				IRBS0690	00070
2	(V( 76),XA3 )	(V( 77),YA3 )	(V( 78),ZA3 )				IRBS0700	00071
3	(V( 79),XA4 )	(V( 80),YA4 )	(V( 81),ZA4 )				IRBS0710	00072
4	(V( 82),XA5 )	(V( 83),YA5 )	(V( 84),ZA5 )				IRBS0720	00073
5	(V( 85),XA6 )	(V( 86),YA6 )	(V( 87),ZA6 )				IRBS0730	00074
6	(V( 88),XA7 )	(V( 89),YA7 )	(V( 90),ZA7 )				IRBS0740	00075
7	(V( 91),XA8 )	(V( 92),YA8 )	(V( 93),ZA8 )				IRBS0750	00076
8	(V( 94),XA9 )	(V( 95),YA9 )	(V( 96),ZA9 )				IRBS0760	00077
9	(V( 97),XA10 )	(V( 98),YA10 )	(V( 99),ZA10 )				IRBS0770	00078
A	(V(100),XA11 )	(V(101),YA11 )	(V(102),ZA11 )				IRBS0780	00079
B	(V(103),XA12 )	(V(104),YA12 )	(V(105),ZA12 )				IRBS0790	00080
C	(V(106),XA13 )	(V(107),YA13 )	(V(108),ZA13 )				IRBS0800	00081
D	(V(109),SF )	(V(110),SFREQ )	(V(111),XSH )				IRBS0810	00082
E	(V(112),YSH )	(V(113),ZSH )	(V(114),THTLAT)				IRBS0820	00083
F	(V(115),THTLON)						IRBS0830	00084
EQUIVALENCE	(V(116),VERVM )	(V(117),AINPVM)					IRBS0840	00085
							IRBS0850	00086
COMMON /RUSED/	DR	TAP1	TAP2	FLC1T	FLC2T		IRBS0860	00087
1	FLC3T	FLC4T	P1H	P1L	P2H		IRBS0870	00088
2	P2L	P3H	P3L	DEL1	DEL2		IRBS0880	00089
3	DEL3	ACC1	ACC2	ACC3	ACC4		IRBS0890	00090
4	ACC5	ACC6	ACC7	ACC8	ACC9		IRBS0900	00091
5	ACC10	ACC11	ACC12	ACC13			IRBS0910	00092
DIMENSION RD(29)							IRBS0920	00093
EQUIVALENCE (RD(1),DR)							IRBS0930	00094
COMMON /RDRDRD/	DRS	TAP1S	TAP2S	FLC1TS	FLC2TS		IRBS0940	00095
1	FLC3TS	FLC4TS	P1HS	P1LS	P2HS		IRBS0950	00096
2	P2LS	P3HS	P3LS	DEL1S	DEL2S		IRBS0960	00097
3	DEL3S	ACC1S	ACC2S	ACC3S	ACC4S		IRBS0970	00098
4	ACC5S	ACC6S	ACC7S	ACC8S	ACC9S		IRBS0980	00099
							IRBS0990	00100

1	1	2	3	4	5	6	7	8	
.....	..0.....	0.....	0.....	0.....	0.....	0.....	0.....	0.....	U1005 CD COUNT
5		ACC105,ACC115,ACC125,ACC135,						IRBS0990	00101
6	1	DRV ,TAP1V ,TAP2V ,FLC1TV,FLC2TV,						IRBS1000	00102
7		FLC3TV,FLC4TV,P1HV ,P1LV ,P2HV ,						IRBS1010	00103
8		P2LV ,P3HV ,P3LV ,DEL1V ,DEL2V ,						IRBS1020	00104
9		DEL3V ,ACC1V ,ACC2V ,ACC3V ,ACC4V ,						IRBS1030	00105
A		ACC5V ,ACC6V ,ACC7V ,ACC8V ,ACC9V ,						IRBS1040	00106
B		ACC10V,ACC11V,ACC12V,ACC13V,						IRBS1050	00107
C		DRP ,TAP1P ,TAP2P ,FLC1TP,FLC2TP,						IRBS1060	00108
D		FLC3TP,FLC4TP,P1HP ,P1LP ,P2HP ,						IRBS1070	00109
E		P2LP ,P3HP ,P3LP ,DEL1P ,DEL2P ,						IRBS1080	00110
F		DEL3P ,ACC1P ,ACC2P ,ACC3P ,ACC4P ,						IRBS1090	00111
G		ACC5P ,ACC6P ,ACC7P ,ACC8P ,ACC9P ,						IRBS1100	00112
H		ACC10P,ACC11P,ACC12P,ACC13P						IRBS1110	00113
		EQUIVALENCE (DRS,R(1,1))						IRBS1120	00114
C								IRBS1130	00115
C		EXTERNAL SELLIN						IRBS1140	00116
C		READ TITLES,USERS,DATE						IRBS1150	00117
C								IRBS1160	00118
		READ (5,10)TITL1,NAME1,NAME2,DATE						IRBS1170	00119
		10 FORMAT(12A6/6A6)						IRBS1180	00120
C								IRBS1200	00121
C		READ AND PLOT ERROR CURVE DATA						IRBS1210	00122
C								IRBS1220	00123
		JSIZE =1000						IRBS1230	00124
		CALL CLOAD(NC,LOC,X,Y,Z)						IRBS1240	00125
		DO 20 I=1,3						IRBS1250	00126
		CALL CPLOT(NC(I),NC,LOC,X,Y,Z)						IRBS1260	00127
		20 CONTINUE						IRBS1270	00128
C								IRBS1280	00129
C		READ LOADER DATA						IRBS1290	00130
C								IRBS1300	00131
		DO 30 I=1,200						IRBS1310	00132
		30 V(I)=0.						IRBS1320	00133
		40 CONTINUE						IRBS1330	00134
		READ (5,50)TITL2						IRBS1340	00135
		50 FORMAT(12A6)						IRBS1350	00136
		ALPHA1 = ALPHA1 * 57.29578						IRBS1360	00137
		ALPHA2 = ALPHA2 * 57.29578						IRBS1370	00138
		BETA = BETA * 57.29578						IRBS1380	00139
		CALL LOADER(V)						IRBS1390	00140
		IDEBUG = DEBUG						IRBS1400	00141
		IVERVM = VERVM						IRBS1410	00142
		INPVM = AINPVM						IRBS1420	00143
C									00144
C								IRBS1430	00145
C		CALCULATE MASS DATA						IRBS1440	00146
C								IRBS1450	00147
		WUB = WUB1+WUB2						IRBS1460	00148
									00149
									00150



1	1	2	3	4	5	6	7	8	
.....0	.....0	.....0	.....0	.....0	.....0	.....0	.....0	.....0	U1005 CD COUNT
	XUB = (WUB1*XUB1+WUB2*XUB2)/WUB							IRBS1470	00151
	YUB = (WUB1*YUB1+WUB2*YUB2)/WUB							IRBS1480	00152
	ZUB = (WUB1*ZUB1+WUB2*ZUB2)/WUB							IRBS1490	00153
	IOXUB = IOXUB1+IOXUB2 + WUB1*((YUB1-YUB)**2 + (ZUB1-ZUB)**2)/386.							IRBS1500	00154
1								IRBS1510	00155
	IOYUB = IOYUB1+IOYUB2 + WUB1*((XUB1-XUB)**2 + (ZUB1-ZUB)**2)/386.							IRBS1520	00156
1								IRBS1530	00157
C								IRBS1540	00158
	WAC = WUB+WLB							IRBS1550	00159
	XAC = (WUB*XUB+WLB*XLB)/WAC							IRBS1560	00160
	YAC = (WUB*YUB+WLB*YLB)/WAC							IRBS1570	00161
	ZAC = (WUB*ZUB+WLB*ZLB)/WAC							IRBS1580	00162
	IXAC = IOXUB + IOXLB + WUB*((YUB-YAC)**2 + (ZUB-ZAC)**2)/386.							IRBS1590	00163
1								IRBS1600	00164
	IYAC = IOYUB + IOYLB + WUB*((XUB-XAC)**2 + (ZUB-ZAC)**2)/386.							IRBS1610	00165
1								IRBS1620	00166
C								IRBS1630	00167
C	PRINT TITLE INFORMATION							IRBS1640	00168
C								IRBS1650	00169
	WRITE (6,60)TITL1,NAME1,NAME2,DATE,TITL2							IRBS1660	00170
60	FORMAT(1H1,29X,12A6/30X,72('-')///							IRBS1670	00171
1	40X,'PREPARED BY ',2A6 //							IRBS1680	00172
2	40X,'CHECKED BY ',2A6 //							IRBS1690	00173
3	40X,'DATE ',2A6 ///							IRBS1700	00174
4	30X, 12A6/30X,72('-') /)							IRBS1710	00175
C								IRBS1720	00176
C	PRINT INPUT, MASS AND GEOMETRY							IRBS1730	00177
C								IRBS1740	00178
	WRITE (6,520)							IRBS1750	00179
	WRITE (6,530)WUB1,WUB2,WUB,WLB,WAC,XUB1,XUB2,XUB,XLB,XAC,YUB1,YUB2							IRBS1760	00180
1	YUB,YLB,YAC,ZUB1,ZUB2,ZUB,ZLB,ZAC,IOXUB1,IOXUB2,IOXUB,IOXLB,IXAC,							IRBS1770	00181
2	IOYUB1,IOYUB2,IOYUB,IOYLB,IYAC							IRBS1780	00182
	WRITE (6,540)							IRBS1790	00183
	WRITE (6,550)(V(I),I=29,34)							IRBS1800	00184
	WRITE (6,560)							IRBS1810	00185
	WRITE (6,570)(V(I),I=35,46)							IRBS1820	00186
	WRITE (6,580)							IRBS1830	00187
	WRITE (6,590)(V(I),I=47,54)							IRBS1840	00188
	WRITE (6,600)							IRBS1850	00189
	DO 70 J=1,13							IRBS1860	00190
	J1 = 3*(J-1)+70							IRBS1870	00191
	J2 = 3*J +69							IRBS1880	00192
	WRITE (6,620)J,(V(I),I=J1,J2)							IRBS1890	00193
70	CONTINUE							IRBS1900	00194
	WRITE (6,610)							IRBS1910	00195
	WRITE (6,630)(V(I),I=55,58)							IRBS1920	00196
	WRITE (6,640)XSH,YSH,ZSH							IRBS1930	00197
	WRITE (6,650)(V(I),I=59,61)							IRBS1940	00198
	WRITE (6,660)PA,AH,PB,AL,G,CV							IRBS1950	00199
	WRITE (6,670)SF,THTLON,SFREQ,THTLAT							IRBS1960	00200

1	2	3	4	5	6	7	8	
.....0.....0.....0.....0.....0.....0.....0.....0								U1005 CD COUNT
C	CONVERT ANGLES TO RADIANS							IRBS1970 00201
	ALPHA1 = ALPHA1/57.29578							IRBS1980 00202
	ALPHA2 = ALPHA2/57.29578							IRBS1990 00203
	BETA = BETA /57.29578							IRBS2000 00204
C								IRBS2010 00205
C	CALCULATE GEOMETRY							IRBS2020 00206
C								IRBS2030 00207
	D( 1) = YI2 -YA							IRBS2040 00208
	D( 2) = YA -YI3							IRBS2050 00209
	D( 3) = ZA -ZLC1							IRBS2060 00210
	D( 4) = YUB -YA							IRBS2070 00211
	D( 5) = XA -XI1							IRBS2080 00212
	D( 6) = XI2 -XA							IRBS2090 00213
	D( 7) = XA -XUB							IRBS2100 00214
	D( 8) = YA -YLC1							IRBS2110 00215
	D( 9) = XA -XLC2							IRBS2120 00216
	D(10) = YLC3 -YA							IRBS2130 00217
	D(11) = XLC4 -XA							IRBS2140 00218
	D(12) = XA -XTAP1							IRBS2150 00219
	D(13) = YTAP1-YA							IRBS2160 00220
	D(14) = XTAP2-XA							IRBS2170 00221
	D(15) = YA -YTAP2							IRBS2180 00222
	D(16) = XP -XA							IRBS2190 00223
	D(17) = ZP -ZA							IRBS2200 00224
C	PRINT CALCULATED GEOMETRY TEMPORARILY							IRBS2210 00225
	WRITE (6,80)D							IRBS2220 00226
	80 FORMAT(///, '*** D ***'/(5F15.3/))							IRBS2230 00227
C								IRBS2240 00228
C	READ CARD INPUT OF RECORDED DATA, 29 CARDS ,NUMBERED 1 - 29 ,							IRBS2250 00229
C								IRBS2260 00230
C	3 NUMBERS PER CARD							IRBS2270 00231
C								IRBS2280 00232
	90 CONTINUE							IRBS2290 00233
	READ (5,100)TITL3,IEND							IRBS2300 00234
100	FORMAT(12A6,6X,I2)							IRBS2310 00235
	DO 170 I=1,29							IRBS2320 00236
	READ (5,110)J,(IR(I,K),K=1,3)							IRBS2330 00237
	IF (J.NE.I) WRITE (6,120)J,I							IRBS2340 00238
110	FORMAT(I5,3I10)							IRBS2350 00239
120	FORMAT(' *** ERROR IN INDEX OF RECORDED DATA ,'/							IRBS2360 00240
	1 ' INDEX READ ', I5/							IRBS2370 00241
	2 ' INDEX SHOULD BE ', I5)							IRBS2380 00242
C								IRBS2390 00243
C	PERFORM VARIOUS DATA CHANGES							IRBS2400 00244
C								IRBS2410 00245
	DO 130 K=1,3							IRBS2420 00246
130	R(I,K)=IR(I,K)							IRBS2430 00247
C	DIVIDE DEFLECTION DATA BY 1000.							IRBS2440 00248
	IF (I.LT.14.OR.I.GT.16) GO TO 150							IRBS2450 00249
	DO 140 K=1,3							IRBS2460 00250



1	2	3	4	5	6	7	8	
.....0.....0.....0.....0.....0.....0.....0.....0.....0								U1005 CD COUNT
C								IRBS2970 00301
C								IRBS2980 00302
	WRITE (6,220)TITL3							IRBS2990 00303
	220 FORMAT(1H1,29X,12A6/30X,72(1-' '))							IRBS3000 00304
C								IRBS3010 00305
C	PRINT EXTERNAL LOADS AND TRUE ISOLATOR LOADS							IRBS3020 00306
	WRITE (6,680)VA,SA,DA,MXA,MYA,MZA,(FIT(I),I=1,3)							IRBS3030 00307
C								IRBS3040 00308
C	CALCULATE MEASURING SYSTEM LOADS AT A ,							IRBS3050 00309
C	APPLIED SYSTEM LOADS AT A ,							IRBS3060 00310
C	CALCULATED LOADS AT A							IRBS3070 00311
C								IRBS3080 00312
	ALPHA = ALPHA1							IRBS3090 00313
	SALPHA=SIN(ALPHA)							IRBS3100 00314
	CALPHA=COS(ALPHA)							IRBS3110 00315
C								IRBS3120 00316
	VAM = FIT(1)+FIT(2)+FIT(3)							IRBS3130 00317
	VAA = VA-WUB							IRBS3140 00318
	SAM = FLC4T - FLC2T							IRBS3150 00319
	SAA = (TAP1-TAP2)*CALPHA +SA							IRBS3160 00320
	DAM = FLC1T - FLC3T							IRBS3170 00321
	DAA = DA + (TAP1-TAP2)*SALPHA							IRBS3180 00322
	MXAM = FIT(2)*D(1) - FIT(3)*D(2) +(FLC4T-FLC2T)*D(3)							IRBS3190 00323
	MXAA = MXA + D(3)*CALPHA*(TAP1-TAP2) - WUB*D(4)							IRBS3200 00324
	MYAM = FIT(1)*D(5) -(FIT(2)+FIT(3))*D(6)+(FLC3T-FLC1T)*D(3)							IRBS3210 00325
	MYAA = MYA - WUB*D(7) +D(3)*SALPHA*(TAP2-TAP1)							IRBS3220 00326
	MZAM = FLC1T*D(8) +FLC2T*D(9)+FLC3T*D(10)+FLC4T*D(11)							IRBS3230 00327
	MZAA = MZA - TAP1*CALPHA*D(12)-TAP1*SALPHA*D(13)							IRBS3240 00328
	1 - TAP2*CALPHA*D(14)-TAP2*SALPHA*D(15)							IRBS3250 00329
C								IRBS3260 00330
	TAP12 = TAP1-TAP2							IRBS3270 00331
	VAC = VAM+WUB							IRBS3280 00332
	SAC = SAM- TAP12*CALPHA							IRBS3290 00333
	DAC = DAM- TAP12*SALPHA							IRBS3300 00334
	MXAC = MXAM- TAP12*CALPHA*D(3)+WUB*D(4)							IRBS3310 00335
	MYAC = MYAM+ TAP12*SALPHA*D(3)+WUB*D(7)							IRBS3320 00336
	MZAC = MZAM+ TAP1 *CALPHA*D(12)+ TAP1*SALPHA*D(13)							IRBS3330 00337
	1 + TAP2 *CALPHA*D(14)+ TAP2*SALPHA*D(15)							IRBS3340 00338
C								IRBS3350 00339
C	PRINT OUT RESULTS							IRBS3360 00340
C								IRBS3370 00341
	ITYPE=TYPE							IRBS3380 00342
	IF (ITYPE.NE.0) GO TO 230							IRBS3390 00343
	WRITE (6,690)VAM,SAM,DAM,MXAM,MYAM,MZAM,VAA,SAA,DAA,MXAA,MYAA,MZAA							IRBS3400 00344
	1,VAC,SAC,DAC,MXAC,MYAC,MZAC							IRBS3410 00345
	IF (IEND.EQ.0) GO TO 90							IRBS3420 00346
	IF (IEND.EQ.1) GO TO 40							IRBS3430 00347
	IF(IEND.EQ.-1)STOP							IRBS3440 00348
230	CONTINUE							IRBS3450 00349
	IRIGID = RIGID							IRBS3460 00350



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C									
	IF (IRIGID.EQ.1) GO TO 260								IRBS3970 00401
	XCOM3 = ACC5V*SIN(ACC5P) - ACC4V*SIN(ACC4P)								IRBS3980 00402
	YCOM3 = ACC5V*COS(ACC5P) - ACC4V*COS(ACC4P)								IRBS3990 00403
	RAUB1P = ATAN2(XCOM3,YCOM3)								IRBS4000 00404
	RAUB1V = SQRT(XCOM3**2 + YCOM3**2) / (YA5-YA4)*386.								IRBS4010 00405
	RAUB1S = (ACC5S-ACC4S) / (YA5-YA4)*386.								IRBS4020 00406
260	CONTINUE								IRBS4030 00407
	XCOM4 = ACC11V*SIN(ACC11P)-ACC10V*SIN(ACC10P)								IRBS4040 00408
	YCOM4 = ACC11V*COS(ACC11P)-ACC10V*COS(ACC10P)								IRBS4050 00409
	RAUB2P = ATAN2(XCOM4,YCOM4)								IRBS4060 00410
	RAUB2V = SQRT(XCOM4**2 + YCOM4**2) / (YA11-YA10)*386.								IRBS4070 00411
	RAUB2S = (ACC11S-ACC10S) / (YA11-YA10)*386.								IRBS4080 00412
	IF (IDEBUG.EQ.0) GO TO 270								IRBS4090 00413
	WRITE (6,280)XCOM1,YCOM1								IRBS4140 00414
	WRITE (6,290)XCOM2,YCOM2								IRBS4150 00415
	WRITE (6,300)XCOM3,YCOM3								IRBS4160 00416
	WRITE (6,310)XCOM4,YCOM4								IRBS4170 00417
270	CONTINUE								IRBS4180 00418
280	FORMAT(' XCOM1= ',F12.4,' YCOM1=',F12.4)								IRBS4190 00419
290	FORMAT(' XCOM2= ',F12.4,' YCOM2=',F12.4)								IRBS4200 00420
300	FORMAT(' XCOM3= ',F12.4,' YCOM3=',F12.4)								IRBS4210 00421
310	FORMAT(' XCOM4= ',F12.4,' YCOM4=',F12.4)								IRBS4220 00422
C									IRBS4230 00423
C	MODIFICATIONS TO TAKE INTO ACCOUNT NON-ALIGNMENT OF ACCELEROMETERS.								00424
C									00425
C	JUST FOR NON-RIGID CASE								00426
C									00427
C	IF(IRIGID.EQ.1) GO TO 271								00428
C									00429
C	UPPER PART OF UPPER BODY. PITCH.								00430
C									00431
	IF(ZA3-ZA2) 2000,3000,2000								00432
3000	IF(ZA3-ZUB1)2000,2001,2000								00433
C									00434
2000	CONTINUE								00435
	DELT3 = XA3 - XUB1								00436
	DELT2 = XUB1- XA2								00437
	DELTZ3 = ZA3 - ZUB1								00438
	DELTZ2 = ZUB1- ZA2								00439
	ANGLE3 = ATAN2(DELTZ3,DELT3)								00440
	ANGLE2 = ATAN2(DELTZ2,DELT2)								00441
	RAD3 = SQRT(DELT3**2 + DELTZ3**2)								00442
	RAD2 = SQRT(DELT2**2 + DELTZ2**2)								00443
	PAUB1V = 386. * SQRT(XCOM1**2 + YCOM1**2) / (RAD2* COS(ANGLE2)								00444
	* + RAD3* COS(ANGLE3))								00445
	PAUB1S = 386. * (ACC2S - ACC3S) / (RAD2*COS(ANGLE2) +								00446
	* RAD3*COS(ANGLE3) )								00447
C									00448
									00449
									00450

1	1	2	3	4	5	6	7	8	
.....0	.....0	.....0	.....0	.....0	.....0	.....0	.....0	.....0	U1005 CD COUNT
2001	CONTINUE								00451
	IF (ZA6-ZUB1) 3001,2002,3001								00452
3001	CONTINUE								00453
	DELTX6 = XA6 - XUB1								00454
	DELTZ6 = ZA6 - ZUB1								00455
	ANGLE6 = ATAN2(DELTZ6,DELTX6)								00456
	RAD6 = SQRT(DELTZ6**2 + DELTX6**2)								00457
	XDSUB1 = ACC6S - (RAD6*PAUB1S*SIN(ANGLE6)) / 386.								00458
	XDUREU = ACC6V*COS(ACC6P) - (RAD6*PAUB1V*SIN(ANGLE6))								00459
	* COS(PAUB1P) / 386.								00460
	* XDUIMU = ACC6V*SIN(ACC6P) - (RAD6*PAUB1V*SIN(ANGLE6))								00461
	* SIN(PAUB1P) / 386.								00462
	XDPUB1 = ATAN2(XDUIMU,XDUREU)*57.29578								00463
	XDVUB1 = SQRT(XDUREU**2 + XDUIMU**2)								00464
C									00465
C	ROLL								00466
C									00467
2002	CONTINUE								00468
	IF (ZA5-ZA4) 2003,3002,2003								00469
3002	IF (ZA5-ZUB1) 2003,2004,2003								00470
2003	CONTINUE								00471
C									00472
	DEITY5 = YA5 -YUB1								00473
	DEITY4 = YUB1-YA4								00474
	DELTZ5 = ZA5 -ZUB1								00475
	DELTZ4 = ZUB1-ZA4								00476
	ANGLE5 = ATAN2(DELTZ5,DEITY5)								00477
	ANGLE4 = ATAN2(DELTZ4,DEITY4)								00478
	RAD5 = SQRT(DEITY5**2 + DELTZ5**2)								00479
	RAD4 = SQRT(DEITY4**2 + DELTZ4**2)								00480
	RAUB1V = 386. * SQRT(XCOM3**2 + YCOM3**2) / (RAD5*COS(ANGLE5)								00481
	* + RAD4*COS(ANGLE4) )								00482
	* RAUB1S = 386. * (ACC5S - ACC4S) / (RAD5*COS(ANGLE5)								00483
	* + RAD4*COS(ANGLE4) )								00484
C									00485
2004	CONTINUE								00486
	IF (ZA7-ZUB1) 3003,2005,3003								00487
3003	CONTINUE								00488
	DEITY7 = YA7 - YUB1								00489
	DELTZ7 = ZA7 - ZUB1								00490
	ANGLE7 = ATAN2(DELTZ7,DEITY7)								00491
	RAD7 = SQRT(DEITY7**2 + DELTZ7**2)								00492
	YDSUB1 = ACC7S + RAD7*RAUB1S*SIN(ANGLE7) / 386.								00493
	YDUREU = ACC7V*COS(ACC7P) + (RAD7*RAUB1V* SIN(ANGLE7))								00494
	* COS(RAUB1P) / 386.								00495
	* YDUIMU = ACC7V*SIN(ACC7P) + (RAD7*RAUB1V*SIN(ANGLE7))								00496
	* SIN(RAUB1P) / 386.								00497
	YDPUB1 = ATAN2(YDUIMU,YDUREU)*57.29578								00498
	YDVUB1 = SQRT(YDUREU**2 + YDUIMU **2)								00499
2005	CONTINUE								00500

1	2	3	4	5	6	7	8	
.....0.....0.....0.....0.....0.....0.....0.....0								U1005 CD COUNT
C								00501
C	LOWER PART OF UPPER BODY .	PITCH.						00502
C								00503
C								00504
	IF(ZA9-ZA8)	2006,4000,2006						00505
4000	IF(ZA9-ZUB1)	2006,2007,2006						00506
C								00507
2006	CONTINUE							00508
	DELTX9 =	XA9 - XUB2						00509
	DELTX8 =	XUB2- XA8						00510
	DELTZ9 =	ZA9 - ZUB2						00511
	DELTZ8 =	ZUB2- ZA8						00512
	ANGLE9 =	ATAN2(DELTZ9,DELTX9)						00513
	ANGLE8 =	ATAN2(DELTZ8,DELTX8)						00514
	RAD9 =	SQRT(DELTZ9**2 + DELTX9**2)						00515
	RAD8 =	SQRT(DELTZ8**2 + DELTX8**2)						00516
	PAUB2V =	386.* SQRT(XCOM2**2 + YCOM2**2) / (RAD8*COS(ANGLE8)						00517
	*	+ RAD9*COS(ANGLE9) )						00518
	PAUB2S =	386.* (ACC8S-ACC9S) / (RAD8*COS(ANGLE8)						00519
	*	+ RAD9*COS(ANGLE9) )						00520
C								00521
2007	CONTINUE							00522
	IF(ZA12 - ZUB2)	4001,2008,4001						00523
4001	CONTINUE							00524
	DELX12 =	XA12 - XUB2						00525
	DELZ12 =	ZA12 - ZUB2						00526
	ANGL12 =	ATAN2(DELZ12,DELX12)						00527
	RAD12 =	SQRT(DELZ12**2 + DELX12**2)						00528
	XDSUB2 =	ACC12S - (RAD12 *PAUB2S * SIN(ANGL12) ) / 386.						00529
	XDUREL =	ACC12V * COS(ACC12P) - (RAD12*PAUB2V * SIN(ANGL12))						00530
	*	* COS(PAUB2P) / 386.						00531
	XDUIML =	ACC12V * SIN(ACC12P) - (RAD12*PAUB2V * SIN(ANGL12))						00532
	*	* SIN(PAUB2P) / 386.						00533
	XDPUB2 =	ATAN2(XDUIML,XDUREL)*57.29578						00534
	XDVUB2 =	SQRT(XDUREL**2 + XDUIML **2 )						00535
C								00536
C	ROLL							00537
C								00538
2008	CONTINUE							00539
	IF(ZA11- ZA10)	2009,4002,2009						00540
4002	IF(ZA11- ZUB2)	2009,2010,2009						00541
2009	CONTINUE							00542
C								00543
	DELY11 =	YA11 - YUB2						00544
	DELY10 =	YUB2 - YA10						00545
	DELZ11 =	ZA11 - ZUB2						00546
	DELZ10 =	ZUB2 - ZA10						00547
	ANGL11 =	ATAN2(DELZ11,DELY11)						00548
	ANGL10 =	ATAN2(DELZ10,DELY10)						00549
	RAD11 =	SQRT(DELY11**2 + DELZ11**2)						00550



1	2	3	4	5	6	7	8	
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RAD10 = SQRT(DELY10**2 + DELZ10**2)								00551
RAUB2V = 386.* SQRT(XCOM4**2 + YCOM4 **2)/ (RAD11 *COS(ANGL11)								00552
* RAD10 *COS(ANGL10) )								00553
RAUB2S = 386.* (ACC11S - ACC10S) / (RAD11* COS(ANGL11)								00554
* RAD10* COS(ANGL10) )								00555
C								00556
2010	CONTINUE							00557
	IF(ZA13 - ZUB2) 4003,2011,4003							00558
4003	CONTINUE							00559
C								00560
	DELY13 = YA13 - YUB2							00561
	DELZ13 = ZA13 - ZUB2							00562
	ANGL13 = ATAN2(DELY13,DELY13)							00563
	RAD13 = SQRT(DELY13**2 + DELZ13**2)							00564
	YDSUB2 = ACC13S + (RAD13* RAUB2S * SIN(ANGL13)) /386.							00565
	YDUREL = ACC13V * COS(ACC13P) + (RAD13* RAUB2V * SIN(ANGL13))							00566
	* COS(RAUB2P) / 386.							00567
	YDUIML = ACC13V * SIN(ACC13P) + (RAD13* RAUB2V * SIN(ANGL13))							00568
	* SIN(RAUB2P) / 386.							00569
	YDPUB2 = ATAN2(YDUIML,YDUREL)*57.29578							00570
	YDVUB2 = SQRT(YDUREL**2 + YDUIML **2)							00571
C								00572
2011	CONTINUE							00573
C								00574
271	CONTINUE							00575
	RAUB1P =RAUB1P * 57.29578							IRBS4100 00576
	PAUB1P =PAUB1P * 57.29578							IRBS4110 00577
	RAUB2P= RAUB2P * 57.29578							IRBS4120 00578
	PAUB2P= PAUB2P * 57.29578							IRBS4130 00579
C								IRBS4240 00580
	IF (IRIGID.EQ.0) GO TO 320							IRBS4250 00581
C								IRBS4260 00582
C	R I G I D A C C E L E R A T I O N S							IRBS4270 00583
C								IRBS4280 00584
	ZDSUB = ACC1S							IRBS4290 00585
	ZDVUB = ACC1V							IRBS4300 00586
	ZDPUB = ACC1P							IRBS4310 00587
	YDSUB = ACC7S							IRBS4320 00588
	YDVUB = ACC7V							IRBS4330 00589
	YDPUB = ACC7P							IRBS4340 00590
	XDSUB = ACC6S							IRBS4350 00591
	XDVUB = ACC6V							IRBS4360 00592
	XDPUB = ACC6P							IRBS4370 00593
	PAUBS = PAUB2S							IRBS4380 00594
	PAUBV = PAUB2V							IRBS4390 00595
	PAUBP = PAUB2P							IRBS4400 00596
	RAUBS = RAUB2S							IRBS4410 00597
	RAUBV = RAUB2V							IRBS4420 00598
	RAUBP = RAUB2P							IRBS4430 00599
	XDPUB = XDPUB * 57.29578							IRBS4440 00600

1	1	2	3	4	5	6	7	8	
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	YDPUB = YDPUB * 57.29578							IRBS4450	00601
	ZDPUB = ZDPUB * 57.29578							IRBS4460	00602
	RAUBP = RAUBP * 57.29578							IRBS4470	00603
	PAUBP = PAUBP * 57.29578							IRBS4480	00604
C	320 CONTINUE							IRBS4490	00605
C	PRINT ACCELERATIONS							IRBS4500	00606
C								IRBS4510	00607
C								IRBS4520	00608
	IF (IDEBUG.EQ.0) GO TO 330							IRBS4530	00609
	IF (IRIGID.EQ.0) WRITE (6,700)ZDSUB1,ZDVUB1,ZDPUB1,ZDSUB2,ZDVUB2,ZIRBS4550							IRBS4540	00610
	1DPUB2,YDSUB1,YDVUB1,YDPUB1,YDSUB2,YDVUB2,YDPUB2,XDSUB1,XDVUB1,XDPUIRBS4560							IRBS4550	00611
	2B1,XDSUB2,XDVUB2,XDPUB2,PAUB1P,PAUB1V,PAUB1S,PAUB2P,PAUB2V,PAUB2S,IRBS4570							IRBS4560	00612
	3RAUB1P,RAUB1V,RAUB1S,RAUB2P,RAUB2V,RAUB2S							IRBS4580	00613
	IF (IRIGID.EQ.1) WRITE (6,710)ZDSUB,ZDVUB,ZDPUB,YDSUB,YDVUB,YDPUB,IRBS4590							IRBS4570	00614
	1XDSUB,XDVUB,XDPUB,PAUBS,PAUBV,PAUBP,RAUBS,RAUBV,RAUBP							IRBS4600	00615
	330 CONTINUE							IRBS4610	00616
C	CALCULATE INERTIA FORCES							IRBS4620	00617
C								IRBS4630	00618
C	IF (IRIGID.EQ.1) GO TO 340							IRBS4640	00619
C	NON-RIGID							IRBS4650	00620
C								IRBS4660	00621
C	UPPER MASS OF UPPER BODY							IRBS4670	00622
C	VIUB1S = WUB1*ZDSUB1							IRBS4680	00623
	VIUB1V = WUB1*ZDVUB1							IRBS4690	00624
	VIUB1P = ZDPUB1 + 180.							IRBS4700	00625
	SIUB1S = WUB1*YDSUB1							IRBS4710	00626
	SIUB1V = WUB1*YDVUB1							IRBS4720	00627
	SIUB1P = YDPUB1 + 180.							IRBS4730	00628
	DIUB1S = WUB1*XDSUB1							IRBS4740	00629
	DIUB1V = WUB1*XDVUB1							IRBS4750	00630
	DIUB1P = XDPUB1 + 180.							IRBS4760	00631
	RIUB1S = IOXUB1*RAUB1S							IRBS4770	00632
	RIUB1V = IOXUB1*RAUB1V							IRBS4780	00633
	RIUB1P = RAUB1P + 180.							IRBS4790	00634
	PIUB1S = IOYUB1*PAUB1S							IRBS4800	00635
	PIUB1V = IOYUB1*PAUB1V							IRBS4810	00636
	PIUB1P = PAUB1P + 180.							IRBS4820	00637
C	LOWER MASS OF UPPER BODY							IRBS4830	00638
	VIUB2S = WUB2*ZDSUB2							IRBS4840	00639
	VIUB2V = WUB2*ZDVUB2							IRBS4850	00640
	VIUB2P = ZDPUB2 + 180.							IRBS4860	00641
	SIUB2S = WUB2*YDSUB2							IRBS4870	00642
	SIUB2V = WUB2*YDVUB2							IRBS4880	00643
	SIUB2P = YDPUB2 + 180.							IRBS4890	00644
	DIUB2S = WUB2*XDSUB2							IRBS4900	00645
	DIUB2V = WUB2*XDVUB2							IRBS4910	00646
	DIUB2P = XDPUB2 + 180.							IRBS4920	00647
								IRBS4930	00648
								IRBS4940	00649
									00650

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RIUB2S = IOXUB2* RAUB2S								IRBS4950 00651
RIUB2V = IOXUB2* RAUB2V								IRBS4960 00652
RIUB2P = RAUB2P + 180.								IRBS4970 00653
PIUB2S = IOYUB2* PAUB2S								IRBS4980 00654
PIUB2V = IOYUB2* PAUB2V								IRBS4990 00655
PIUB2P = PAUB2P + 180.								IRBS5000 00656
C ALLOW VIBRATORY MASS TO BE USED , NOT STEADY MASS								00657
IF(IVERVM.NE.0) VIUB2V=VERVM * ZDVUB2								00658
IF(INPVM .NE.0) SIUB2V=AINPVM* YDVUB2								00659
IF(INPVM .NE.0) DIUB2V=AINPVM* XDVUB2								00660
C								IRBS5010 00661
GO TO 350								IRBS5020 00662
C								IRBS5030 00663
C UPPER BODY AS A RIGID SYSTEM								IRBS5040 00664
C								IRBS5050 00665
340 CONTINUE								IRBS5060 00666
VIUBS = WUB*ZDSUB								IRBS5070 00667
VIUBV = WUB*ZDVUB								IRBS5080 00668
VIUBP = ZDPUB + 180.								IRBS5090 00669
SIUBS = WUB*YDSUB								IRBS5100 00670
SIUBV = WUB*YDVUB								IRBS5110 00671
SIUBP = YDPUB + 180.								IRBS5120 00672
DIUBS = WUB*XDSUB								IRBS5130 00673
DIUBV = WUB*XDVUB								IRBS5140 00674
DIUBP = XDPUB + 180.								IRBS5150 00675
RIUBS = IOXUB*RAUBS								IRBS5160 00676
RIUBV = IOXUB*RAUBV								IRBS5170 00677
RIUBP = RAUBP + 180.								IRBS5180 00678
PIUBS = IOYUB*PAUBS								IRBS5190 00679
PIUBV = IOYUB*PAUBV								IRBS5200 00680
PIUBP = PAUBP + 180.								IRBS5210 00681
C ALLOW VIBRATORY MASS TO BE USED								00682
IF(IVERVM.NE.0) VIUBV =VERVM * ZDVUB								00683
IF(INPVM .NE.0) SIUBV =AINPVM* YDVUB								00684
IF(INPVM .NE.0) DIUBV =AINPVM* XDVUB								00685
C								IRBS5220 00686
350 CONTINUE								IRBS5230 00687
C								IRBS5240 00688
C SET UP VECTORS FROM RECORDED DATA								IRBS5250 00689
C								IRBS5260 00690
PHV(1) = P1HV								IRBS5270 00691
PHV(2) = P2HV								IRBS5280 00692
PHV(3) = P3HV								IRBS5290 00693
PLV(1) = P1LV								IRBS5300 00694
PLV(2) = P2LV								IRBS5310 00695
PLV(3) = P3LV								IRBS5320 00696
PHS(1) = P1HS								IRBS5330 00697
PHS(2) = P2HS								IRBS5340 00698
PHS(3) = P3HS								IRBS5350 00699
PLS(1) = P1LS								IRBS5360 00700

1	2	3	4	5	6	7	8	
.....0.....0.....0.....0.....0.....0.....0.....0.....0								U1005 CD COUNT
	PLS(2) = P2LS							IRBS5370 00701
	PLS(3) = P3LS							IRBS5380 00702
	PHP(1) = P1HP							IRBS5390 00703
	PHP(2) = P2HP							IRBS5400 00704
	PHP(3) = P3HP							IRBS5410 00705
	PLP(1) = P1LP							IRBS5420 00706
	PLP(2) = P2LP							IRBS5430 00707
	PLP(3) = P3LP							IRBS5440 00708
C	CALCULATIONS PER ISOLATOR							IRBS5450 00709
C	DO 380 I=1,3							IRBS5460 00710
	FHV(I) = AH * PHV(I)							IRBS5470 00711
	FLV(I) = AL * PLV(I)							IRBS5480 00712
	IF (IDEBUG.EQ.0) GO TO 370							IRBS5490 00713
	WRITE (6,360)AH,AL,PHV(I),PLV(I),FHV(I),FLV(I),PHP(I),PLP(I)							IRBS5500 00714
360	FORMAT(' ***',8F12.4)							IRBS5510 00715
370	CONTINUE							IRBS5520 00716
	XX(I) = FHV(I)*SIN(PHP(I)) - FLV(I)*SIN(PLP(I))							IRBS5530 00717
	YY(I) = FHV(I)*COS(PHP(I)) - FLV(I)*COS(PLP(I))							IRBS5540 00718
	FIV(I) = SQRT(XX(I)**2 + YY(I)**2)							IRBS5550 00719
	FIVP(I) = ATAN2(XX(I),YY(I))							IRBS5560 00720
	FIVPX(I) = FIVP(I) * 57.29578							IRBS5570 00721
C								IRBS5580 00722
C	CORRECTED STEADY LOADS PER ISOLATOR							IRBS5590 00723
C	FIM(I) = PHS(I)*AH - PLS(I)*AL							IRBS5600 00724
	CALL CIP(IERR(I),NC,LOC,X,Y,SELLIN,FIM(I),ERROR(I),DM1,DM2,DM3,2)							IRBS5610 00725
	FITS(I) = FIM(I) + ERROR(I)							IRBS5620 00726
C								IRBS5630 00727
C	MEASURING SYSTEM LOADS							IRBS5640 00728
C								IRBS5650 00729
	FIT(I) = FITS(I) + FIV(I)*COS(FIVP(I))							IRBS5660 00730
C								IRBS5670 00731
380	CONTINUE							IRBS5680 00732
	IF (IDEBUG.EQ.0) GO TO 400							IRBS5690 00733
	WRITE (6,390)(XX(I),YY(I),FIV(I),FIVPX(I),I=1,3)							IRBS5700 00734
390	FORMAT(2X,4F12.3)							IRBS5710 00735
400	CONTINUE							IRBS5720 00736
C								IRBS5730 00737
	FLCT(1) = FLC1TS + FLC1TV * COS(FLC1TP)							IRBS5740 00738
	FLCT(2) = FLC2TS + FLC2TV * COS(FLC2TP)							IRBS5750 00739
	FLCT(3) = FLC3TS + FLC3TV * COS(FLC3TP)							IRBS5760 00740
	FLCT(4) = FLC4TS + FLC4TV * COS(FLC4TP)							IRBS5770 00741
C								IRBS5780 00742
C	CALCULATION OF SHAKER FORCE (MEASURED)							IRBS5790 00743
C								IRBS5800 00744
	VIUBP = VIUBP / 57.29578							IRBS5810 00745
	DIUBP = DIUBP / 57.29578							IRBS5820 00746
	SIUBP = SIUBP / 57.29578							IRBS5830 00747
								IRBS5840 00748
								IRBS5850 00749
								IRBS5860 00750



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APPENDIX III-C

Sample Test Cases

ACTIVE ISOLATOR FULL SCALE SHAKE TEST

PREPARED BY J.DEFELICE

CHECKED BY

DATE JAN.3,1973

VIBRATORY EXCITATION--LONGITUDINAL 500 LB FORCE 30HZ (RUN 6C)

MASS DATA

	UPPER PART OF LOWER BODY	LOWER PART OF LOWER BODY	TOTAL UPPER BODY	LOWER BODY	TOTAL AIRCRAFT
W	4808.000	3004.700	7812.700	26187.300	34000.000
X	337.160	335.650	336.579	336.350	336.403
Y	.000	.000	.000	.000	.000
Z	257.120	199.980	235.144	129.820	154.022
IX	2060.000	1522.000	1922.750	113336.000	305494.310
IY	2372.000	1980.000	20003.673	1720534.000	1913473.800

GEOMETRY

	NO.	X	Y	Z	ANGLE
ISOLATORS:					
	1	306.600	.000		
	2	356.100	25.200		
	3	356.100	-25.200		
LOAD CELLS (MEASURING) :					
	1	324.000	-20.000	197.125	
	2	322.000	18.300	197.125	
	3	363.600	20.000	197.125	
	4	363.600	-19.000	197.125	
LOAD CELLS (TORQUE APP):					
	1	335.700	26.200	197.125	65.000
	2	348.300	-26.200	197.125	65.000

# ACCELEROMETERS:

	X	Y	Z	ANGLE
1	337.000	.000	270.000	
2	324.000	.000	253.900	
3	349.000	.000	256.100	
4	337.000	-14.000	255.000	
5	337.000	14.000	255.000	
6	337.000	14.000	255.000	
7	337.000	14.000	255.000	
8	326.000	.000	202.000	
9	359.000	.000	200.000	
10	336.000	-18.000	200.000	
11	336.000	21.000	200.000	
12	336.000	21.000	200.000	
13	336.000	21.000	200.000	

## PROPULSIVE FORCE:

370.000 .000 259.700 -1.000

## SHAKER FORCE:

335.900 .000 261.900

## CORELLATION POINT

336.400 .000 258.000

## ISOLATOR CHARACTERISTICS

PRECHARGE , HIGH SIDE =	300.000	AIR VOLUME REACTION SURFACE , HIGH SIDE =	12.393
PRECHARGE , LOW SIDE =	1100.000	AIR VOLUME REACTION SURFACE , LOW SIDE =	14.193
GAIN =	100.000	DAMPING =	.000
SHAKER FORCE =	500.000	LONGITUDINAL ANGLE =	90.000
FREQUENCY =	30.000	LATERAL ANGLE =	.000

\*\*\* D \*\*\*

25.200	25.200	60.875	.000	29.600
19.700	.179	20.000	14.400	20.000
27.200	.700	26.200	11.900	26.200
33.600	1.700			

VIBRATORY LOAD LONGITUDINAL EXCITATION 500 LB FORCE 30HZ (RUN 6C)

RECORDED DATA

ITEM	CHANNEL	STEADY	VIBRATORY	PHASE
DR	1	.000	.000	360.000
TAP1	2	.000	.000	360.000
TAP2	3	.000	.000	360.000
FLC1T	4	1.000	30.000	360.000
FLC2T	5	78.000	32.000	360.000
FLC3T	6	-37.000	17.000	360.000
FLC4T	7	42.000	31.000	360.000
PIH	8	1348.000	6.000	360.000
P1L	9	474.000	2.000	180.000
P2H	10	1184.000	7.000	180.000
P2L	11	462.000	2.000	360.000
P3H	12	971.000	6.000	180.000
P3L	13	253.000	2.000	360.000
DEL1	14	-.012	.016	359.820
DEL2	15	.000	.002	359.820
DEL3	16	-.001	.019	359.820
ACC1	17	-.030	.020	360.000
ACC2	18	.000	.020	360.000
ACC3	19	.000	.030	180.000
ACC4	20	.000	.010	360.000
ACC5	21	.000	.000	360.000
ACC6	22	-.010	.090	360.000
ACC7	23	.070	.000	360.000
ACC8	24	-.010	.050	360.000
ACC9	25	.010	.060	180.000
ACC10	26	.000	.020	360.000
ACC11	27	.000	.000	360.000
ACC12	28	.000	.020	360.000
ACC13	29	.000	.000	360.000

VIBRATORY LOAD LONGITUDINAL EXCITATION 500 LB FORCE 30HZ (R/JN 6C)-----

-----  
 PROCESSED DATA  
 -----

EXTERNAL APPLIED LOADS AT A

V	34000.000	S	.000	D	.000	MX	.000	MY	.000	MZ	.000
---	-----------	---	------	---	------	----	------	----	------	----	------

CORRECTED ISOLATOR LOADS

FI1	9946.896	FI2	8073.122	FI3	8401.791
-----	----------	-----	----------	-----	----------

PROCESSED DATA  
-----

	X	Y	Z	ROLL	PITCH
<b>ACCELERATIONS:</b>					
UPPER PART OF UPPER BODY					
STEADY	-.010	.070	-.030	.000	-.000
VIBRATORY	.094	.001	.020	.138	.772
PHASE	-.000	-.000	360.000	180.000	-.000
LOWER PART OF UPPER BODY					
STEADY	.000	.000	-.030	.000	-.234
VIBRATORY	.020	.000	.020	.198	1.287
PHASE	-.000	180.000	360.000	180.000	-.000
<b>INERTIA FORCES:</b>					
UPPER PART OF UPPER BODY					
STEADY	-48.080	336.560	-144.240	.000	.000
VIBRATORY	453.106	3.640	96.160	283.986	1831.184
PHASE	180.000	180.000	540.000	360.000	180.000
LOWER PART OF UPPER BODY					
STEADY	.036	.000	-90.141	.000	-463.200
VIBRATORY	59.894	.031	60.094	301.278	2547.600
PHASE	180.000	360.000	540.000	360.000	180.000
<b>ISOLATOR LOADS:</b>					
STEADY	9946.896				
VIBRATORY	102.744				
PHASE	-.000				
	NO.1		NO.2		NO.3
	8073.122		8401.791		8401.791
	115.137		102.744		102.744
	180.000		180.000		180.000



OUTPUT  
-----

STEADY LOADS AT POINT A:			
	X	Y	Z
MEASURED	38.000	-36.000	26421.809
APPLIED	-0.000	-0.000	26187.300
ROTOR-MEASURED	38.000	-36.000	34234.509
ROTOR-APPLIED			

VIBRATORY ROTOR HEAD FORCES					
	VERTICAL(Z)		LATERAL(Y)		LONGITUDINAL(X)
	AMPLITUDE	PHASE	AMPLITUDE	PHASE	AMPLITUDE
APPLIED	0.000	0.000	0.000	0.000	0.000
MEASURED	41.117	-0.000	2.610	-0.000	500.000
					526.000
					-0.000

\*\*\*\*\*

LOADER CARD INPUT

COLUMN	2	4	6	19	31	43	55	67
*	*	*	*	*	*	*	*	*
2	1	1	0	*	*	*	*	*
2	109	1	750.0	0	10.0			
1	114		90.0					
-1	115		0.0					

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APPENDIX IV

Analysis of Contributions to Error in

Measuring Vibratory Loads

The following is an analysis of three vibratory load conditions which substantiate the major contribution of acceleration measurements to the determination of vibratory loads.

1. Vertical Vibratory Load - 500 lbs. @ 30 Hz.

Isolator Vibratory Loads

#1 104.5 lbs.

#2 90.5 lbs.

#3 64.0 lbs.

Total 259.0 lbs.

Inertia Response of Transmission/Rotor Head

	Acceleration - g's	Force
Rotor Head	.06	288 lbs.
Transmission	.06	180 lbs.
Total		468 lbs.

Total measured load = 727 lbs.

Accelerometer contribution to Total measured load = 468 lbs.

2. Vertical Vibratory Load - 495 lbs. @ 30 Hz.

Isolator Vibratory Loads

#1 -102 lbs.

#2 +102 lbs.

#3 -116 lbs.

Total -116 lbs.

Inertia Response of Transmission/Rotor Head

	Acceleration - g's	Force
Rotor Head	.06	288 lbs.
Transmission	.06	180 lbs.
Total		468 lbs.

Total measured load = 352 lbs.

Accelerometer contribution to Total measured load = 468 lbs.

3. Longitudinal Vibratory Load - 1000 lbs. @ 18.5 Hz.

Loads present in Longitudinal Drag Links

Forward +15 lbs.

Aft	-65 lbs.
Total	-50 lbs.

Inertia Response of Transmission/Rotor Head

	Acceleration - g's	Force
Rotor Head	.23	1105 lbs.
Transmission	.07	210 lbs.
	Total	1315 lbs.

Total measured load = 1270 lbs.

Accelerometer contribution to Total measured load = 1315 lbs.

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2. Von Hardenberg, P.W., and Saltanis, P.B., "Ground Test Evaluation of the Sikorsky Active Transmission Isolation System", USAAMRDL Report 71-38, Sikorsky Aircraft Division, United Aircraft Corporation; prepared under Contract DAAJ02-69-C-0101, September 1971.